

**ANALYTICAL AND EXPERIMENTAL DETERMINATION
OF LOCALIZED STRUCTURE TO BE USED IN
LABORATORY VIBRATION TESTING OF SHELL
STRUCTURE-MOUNTED COMPONENTS, SATURN V**

FINAL REPORT

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ABSTRACT

The final report covers the work performed for the contract entitled, "Analytical and Experimental Determination of Localized Structure to be Used in Laboratory Vibration Testing of Shell Structure-Mounted Components, SATURN V." The contract was executed during the period of April 1965 to July 1967 inclusive. Earlier work performed in the contract was reported in the yearly progress report dated May 1966 (two volumes), and a progress report dated January 1967. Except for an overall work summary and some revised data, the final report describes essentially the additional work items performed since January 1967. Major subjects in the final report include the revised finite difference program for localized shell vibration, shell transient and acoustic responses considering mass attachments, shock and blast overpressure. Also described are the theory and procedure in designing shell scale models.

FOREWORD

The final report was prepared by Northrop Corporation, Norair Division, Hawthorne, California, under contract no. NAS8-20025, "Analytical and Experimental Determination of Localized Structure to be Used in Laboratory Vibration Testing of Shell Structure-Mounted Components, SATURN V."

The subject contract is administered under the direction of the Vibration and Acoustics Branch, Structures Division, Propulsion and Vehicle Engineering Laboratory, George C. Marshall Space Flight Center of the National Aeronautics and Space Administration. Mr. J.H. Farrow and Mr. R.E. Jewell are the principal and alternate technical representatives. Mr. C.E. Lifer served as the alternate technical representative in the early phase of the contract. Mr. L.D. Saint is the program monitor. The program manager at Northrop Norair is Dr. Chintsun Hwang, Member of Technical Management, Structures and Dynamics Research Branch. Dr. W.S. Pi, Dr. N.M. Bhatia and Mr. J.R. Yamane participated in the project. Mr. P.E. Finwall is responsible for the experimental tasks of the program.

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SECTION I
INTRODUCTION

In laboratory vibration testing of shell structure-mounted components, an important requirement is to simulate the localized dynamic characteristics of the segment and the mounted components. The objective of the present contract is to develop design and testing techniques for segmented shell segments from typical SATURN V structures. In the process, to achieve proper design of the elastic supports, a preliminary study was executed on various types of spring supports attached to a rectangular plate. Simultaneously, the overall dynamic responses of the structure were investigated. Based on the overall response data, a part of the shell structure was defined as the shell segment. Proper constraints and boundary conditions were specified to achieve the similarity in dynamic responses as compared to the complete shell structure. In the second year of the program period, additional work items were included concerning the transient and acoustic responses of shell structures as well as shock load and blast overpressure.

During the contract period, a yearly progress report dated May 1966 and an intermediate report dated January 1967 were issued covering the work accomplished up to the reporting time (References 1, 2). The final report covers essentially the work performed during the balance of the contract period. For the sake of completeness, this report also presents a general review of all the work items carried out in the contract.

In analyzing the vibration behavior of a flexibly-supported rectangular plate, a finite difference method was used. The method transformed the fourth order partial differential equation of a plate in rectangular coordinates into a finite difference equation in terms of the normal displacement at selected grid points. Additional grid points were used beyond the plate boundary. Proper boundary conditions were used representing either a free edge or an edge with local flexible supports. Considering the dynamic inertia effect of the plate mass, the problem was reduced to an eigenvalue matrix formulation. The numerical solution of the final matrix equation yielded the modal and frequency data of the rectangular plate. As compared to the

experimental data, the solution by the finite difference method was found to be very reliable. Using this approach, the dynamic effects of various types of flexible supports were evaluated.

To investigate the overall shell dynamic behavior prior to segmentation, four scale models were fabricated based on various parts of SATURN V structures. Duplicate models were made for segmentation purposes. The scale models which were manufactured and tested are listed below and shown in Figures 1-4. The detail technique in scale model design is described in Section V of this final report.

- S-4B Instrument unit, 1:6.67 scale (Figure 1)
- S-2 Thrust cone including simulated rocket engines,
1:10 scale (Figure 2)
- S-2 Forward skirt including Lox tank upper bulkhead,
1:10 scale (Figure 3)
- S-1C Lox tank upper bulkhead including partial cylindrical
shell structure, 1:10 scale. (Figure 4)

Analytical and experimental programs were conducted to determine the vibration and dynamic response behavior of the shell structures. In the analytical phase, partial differential equations were established along the shell meridian. The dependent variables include three displacement components, the angle of rotation and the four shell internal stress components in the same directions as the four displacement variables. The stress variables are the transverse shear Q , the membrane stresses N_{ϕ} , N_r and the meridian bending moment M_{ϕ} . For each circumferential harmonic number, the equations were solved numerically to yield proper modal and frequency data. The dynamic effects of the stringers and the ring stiffeners were handled differently. For the stringers which were located along the shell meridians, their stiffness was averaged and merged with the shell to form a mean stiffness. For the ring stiffeners, the dynamic impedances were formulated individually. The impedances were represented in terms of the increments of the shell internal stresses as functions of the local displacements. These increments were introduced into the differential equations at the ring stiffener locations during numerical integration. The computer program to execute the integration and typical modal and frequency data for shell structure models were presented in Reference 1.

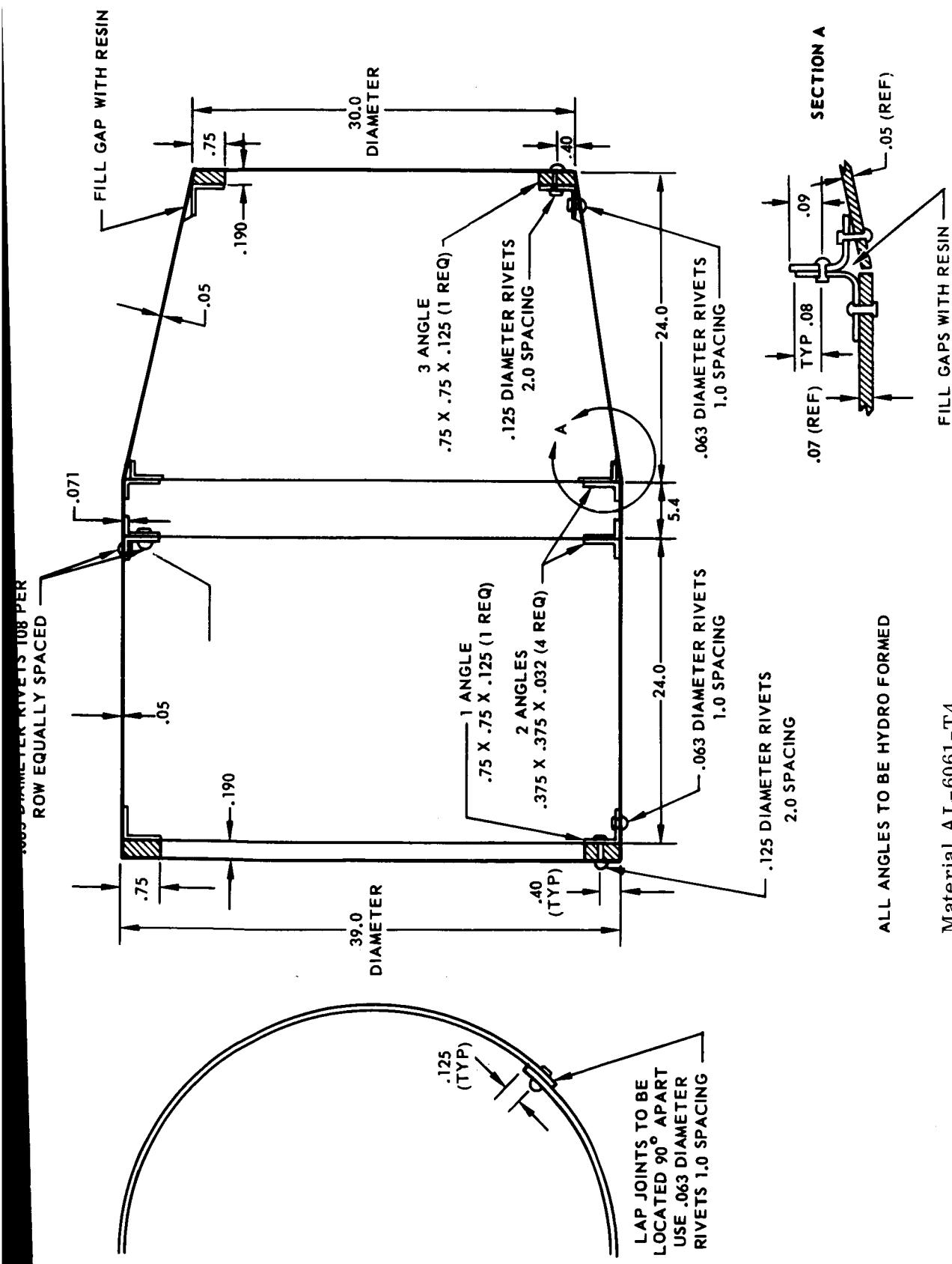
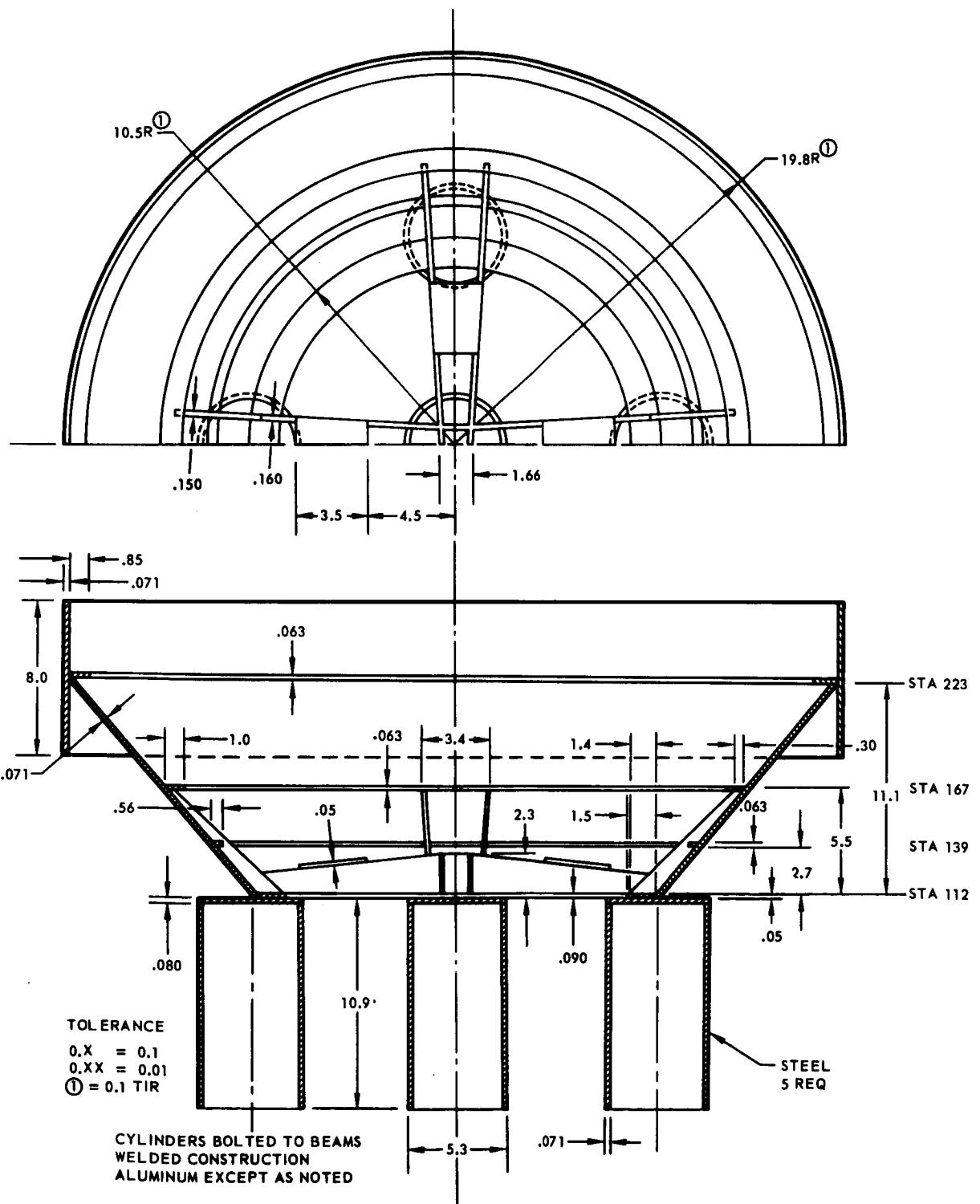


FIGURE 1. INSTRUMENT UNIT SCALE MODEL DESIGN DRAWING

Material AL-6061-T4



**FIGURE 2. S-2 THRUST CONE SCALE MODEL DESIGN
DRAWING**

- NOTE:**
- (1) MATERIAL AL-6061-T4 OR T6
 - (2) BULKHEAD TO BE FORMED BY SPINNING
 - (3) BUTT WELDS TO BE SANDDED SMOOTH

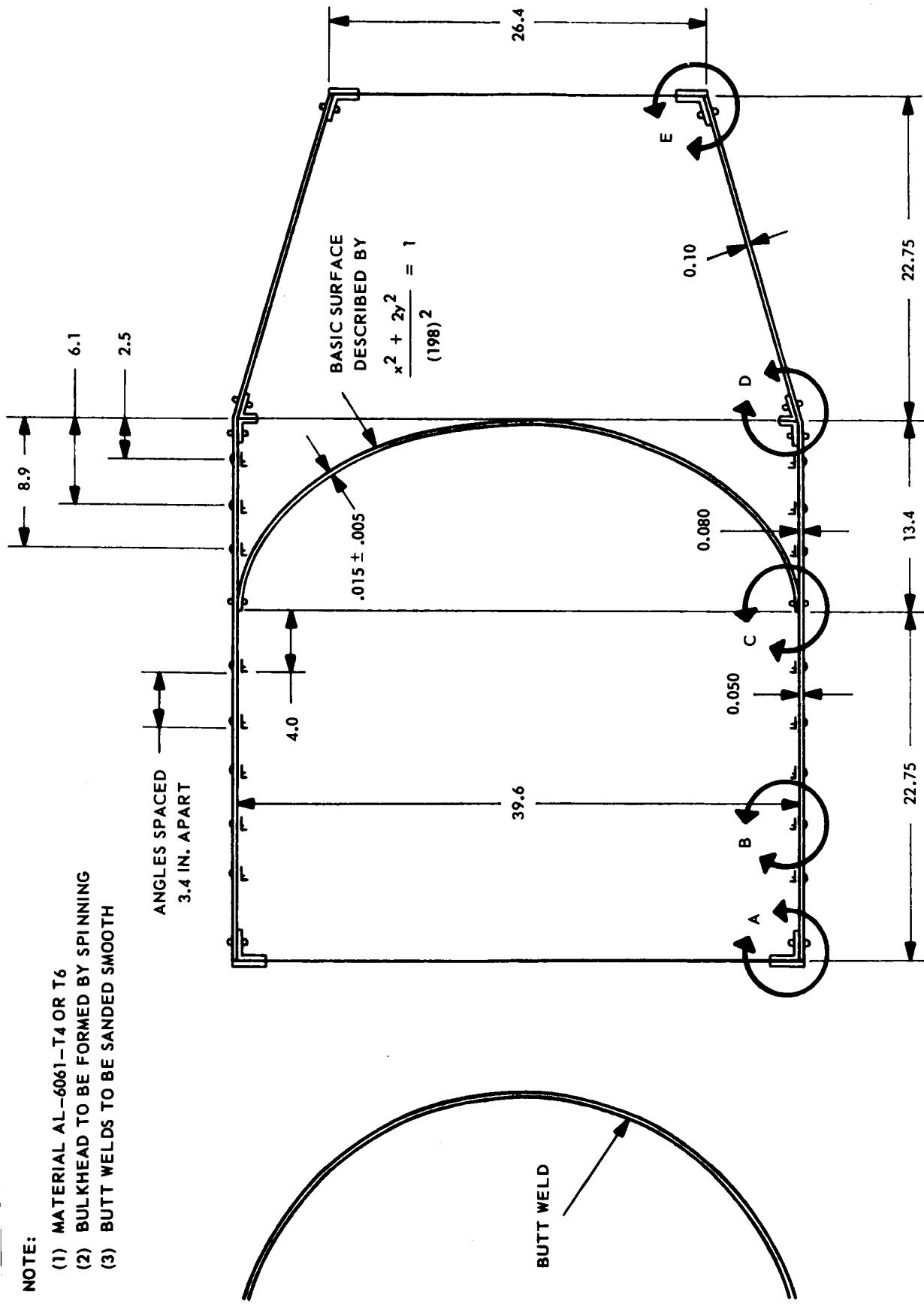
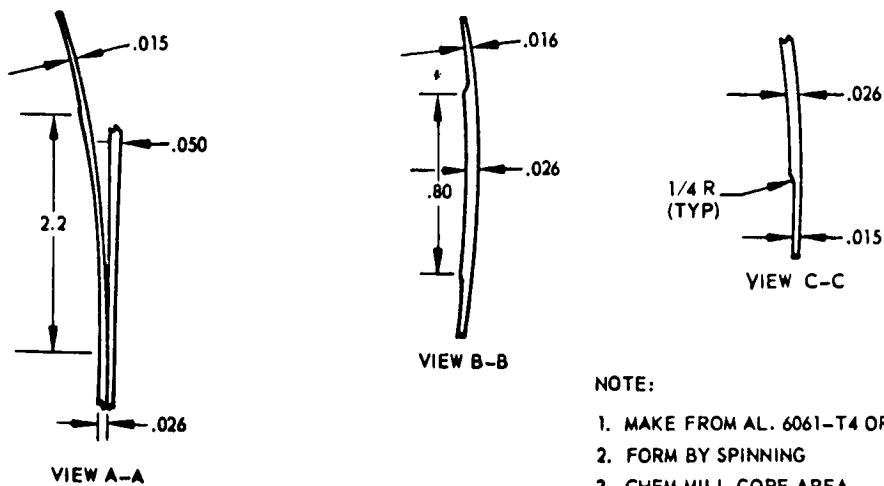
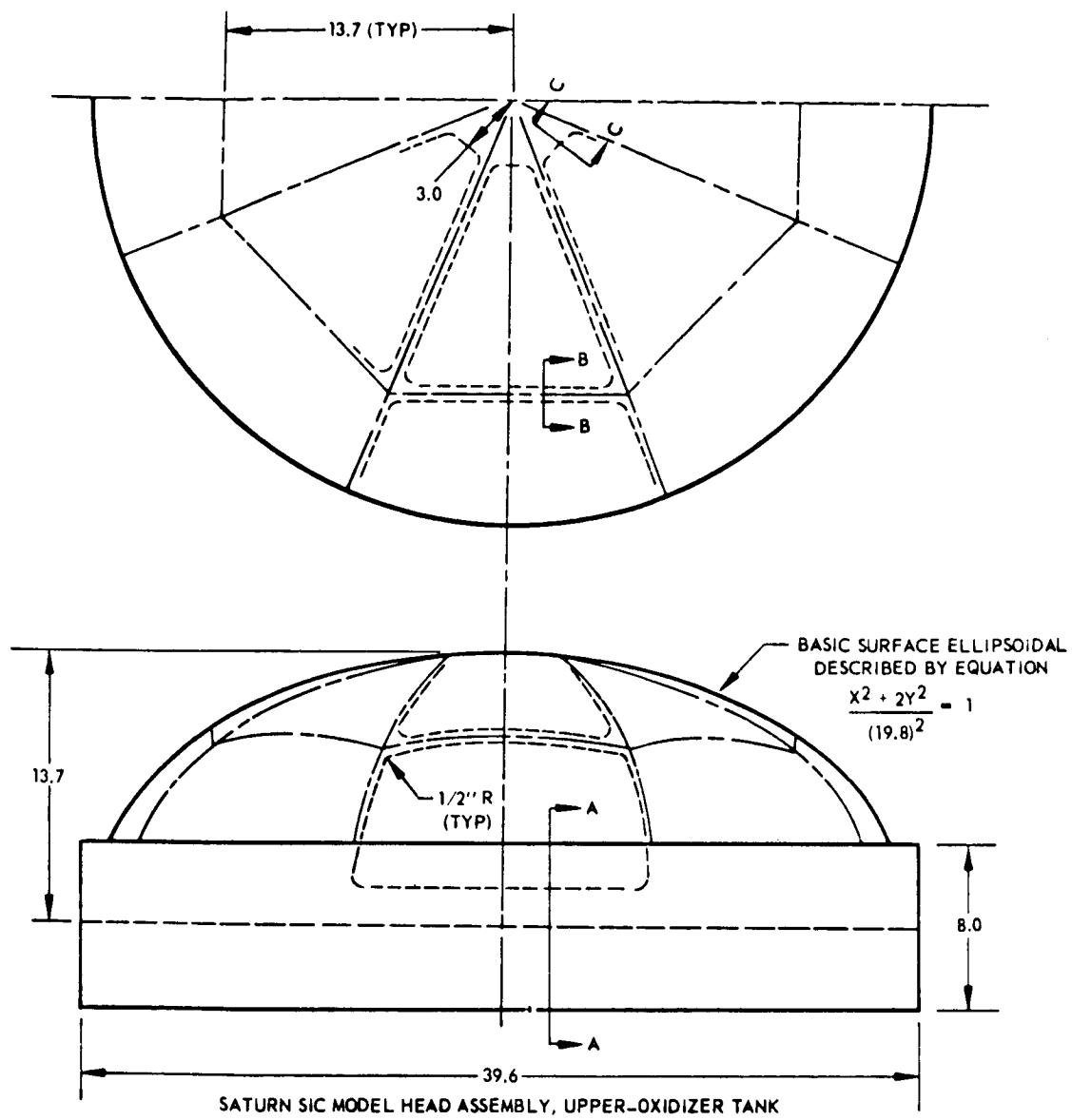


FIGURE 3. S-2 FORWARD SKIRT SCALE MODEL DESIGN DRAWING



GURE 4. S-1C OXIDIZER TANK UPPER BULKHEAD SCALE MODEL DESIGN DRAWING

In the parallel experimental program, the shell models were excited sinusoidally. For overall impedance investigation, a frequency sweep was performed to record the impedance at specific locations as a function of the frequency. In the modal survey corresponding to an observed natural frequency, proximity probes were used to record the shell responses at various locations. A block diagram showing the test instrumentation is given in Figure 5. Test data obtained in the first year performance period may be found in Reference 1.

Tests were also performed on a segmented instrument unit shell structure model with and without simulated component attachments. The response data of the segmented shell were evaluated against the corresponding data of the complete shell structure. During the design and test process, a procedure was developed which may serve as a guide to testing engineers. After the segmentation technique was proven on the scale model shell structure, the full scale SATURN V instrument unit was segmented and tested. The impedance data for the scale model and full scale segmented shell structures were compared using proper scaling relations. The design procedure, as well as the analytical and experimental data of segmented shells were presented in Reference 2.

The analytical prediction of the vibration of a flexibly supported shell segment was based on a finite difference technique. The technique was a generalization of the method applied to the rectangular plate which was described previously in the section. It involved the formulation of the equilibrium and compatibility conditions of a cross-stiffened shell element. Attached masses and the corresponding mass moment of inertia effects were included in the equilibrium equations. Grid points were assumed which covered the shell segment and the neighboring areas. The relating matrix technique and the numerical data were illustrated in Reference 2. Since the completion of the intermediate report, certain modifications and improvements were made on the finite difference computer program which is presented in Section II of the final report.

Additional work items performed in the extended period of the contract included the dynamic and acoustic response investigation of the shell scale models which are described in the final report. Specifically, Section III deals with the transient and impulsive responses of the stiffened shells. Section IV describes the analytical and experimental investigation on the acoustic loading and blast overpressure on a shell structure model. Section V concludes the report with the comprehensive shell scale model design procedure which may be used for future design purposes. The input formats and the detail listings of all computer programs are given in the appendices.

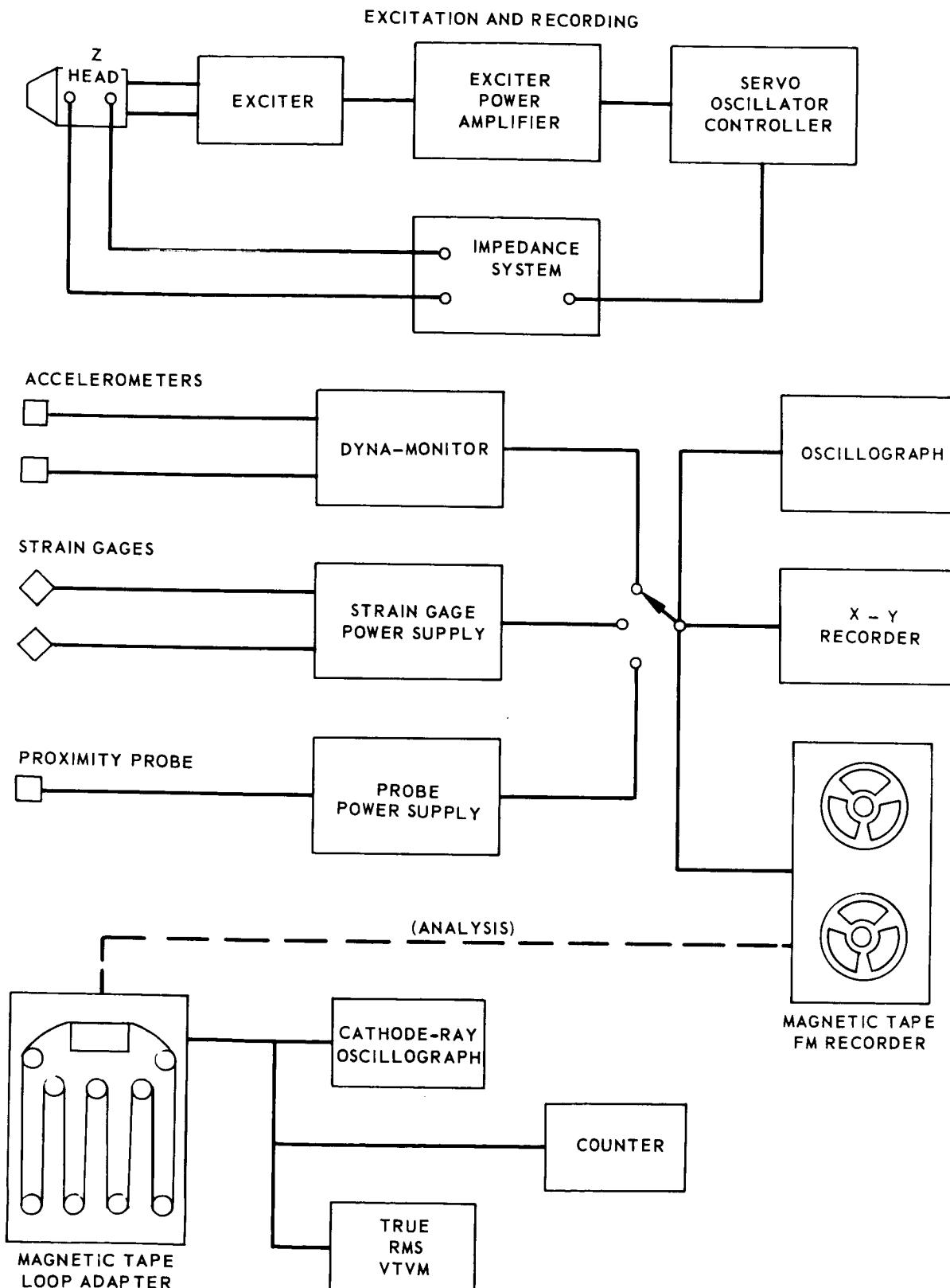


FIGURE 5. INSTRUMENTATION BLOCK DIAGRAM

SECTION II

THE REVISED FINITE DIFFERENCE COMPUTER PROGRAM

GENERAL

The program gives the analytical predictions of natural vibration modes and frequencies of a solid or sandwich curved panel with arbitrary boundary and supporting conditions. It is written in FORTRAN IV language, for the use of IBM 7090 or 7094 data processing systems. The user is required to prepare a complete coefficient matrix, derived from the finite difference expressions of equilibrium, compatibility, and boundary equations. For the detail analysis, the reader is referred to Reference 2. In the program, proper arrangement of the matrix rows is required as shown in the input data format. The program is set up for (172 x 172) coefficient matrix, although its function is mechanized to accommodate any matrix of reasonable size. In the latter case, due consideration should be given to the capacity of the in-core storage space. Furthermore, DIMENSION and EQUIVALENCE statements should be made compatible to the size of the matrix chosen.

The program adapts an overlay structure. The contents in each link are as follows:

a. MAIN

It "CALLS" all subsequent links.

b. \$ ORIGIN BETA

It contains two subroutines - matrix multiplication, and matrix inversion, which are linked directly to the main program.

c. \$ ORIGIN ALPHA - CHN 1 (CHN 1 indicates the first chain.)

It evaluates all elements needed for the coefficient matrix, and puts them into a table form called "E Table". Mass matrices are also formed in this link. This subroutine is linked to ORIGIN BETA.

d. \$ORIGIN ALPHA - CHN 2

Formation of AA3 matrix (compatibility equations) is carried out by transferring the proper elements in the E table to the proper locations in the AA3 matrix. Some matrix algebra are also performed here. This subroutine is linked to ORIGIN BETA, and overlaid on CHN 1.

e. \$ORIGIN ALPHA - CHN 3

Formation of AA1 matrix (equilibrium equations) is carried out in this chain. Some matrix algebra are performed. The subroutine is linked to ORIGIN BETA, and overlaid on CHN 2.

f. \$ORIGIN ALPHA - CHN 4

Formation of AA2 matrix (boundary conditions) is accomplished. With further matrix manipulation, reduction of the coefficient matrix results in an eigenmatrix. This subroutine is linked directly to ORIGIN BETA, and overlaid on CHN 3.

g. \$ORIGIN BETA - CHN 5

Subroutine "MITER", which computes the eigenvalues and eigenvectors, is included in this link. Calling "MITER" is the sole purpose of this link. It is linked directly to the main program, and overlaid on all the aforementioned links except the main program and the COMMON statements.

Users should note that there are three working peripheral tapes specifically assigned to the program in addition to the variable tape numbers connected with the "MITER" subroutine.

INPUT DATA

Figure 6 shows the full scale instrument unit test panel set-up. Prior to the test, a (1:6.67) scale model of the panel was tested. The finite difference computer program was used to obtain the modal data of the scale model. Corresponding to a quarter of the shell panel, the program is set up for the grid pattern shown in Figure 7.

Following is a list of the coding symbols used in the program:

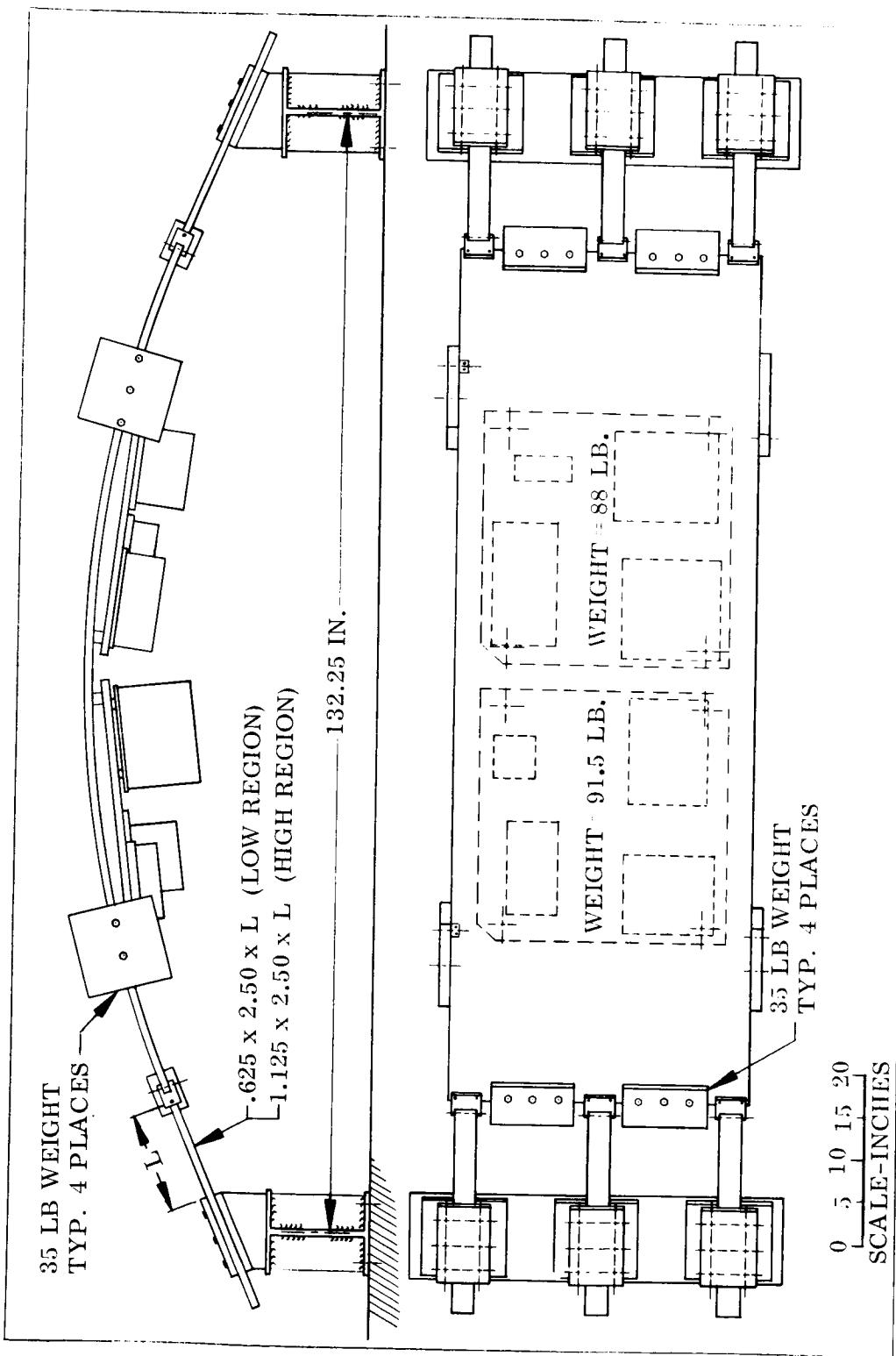


FIGURE 6. TYPICAL STIFFENED SHELL STRUCTURE AND THE SUPPORTING FIXTURE

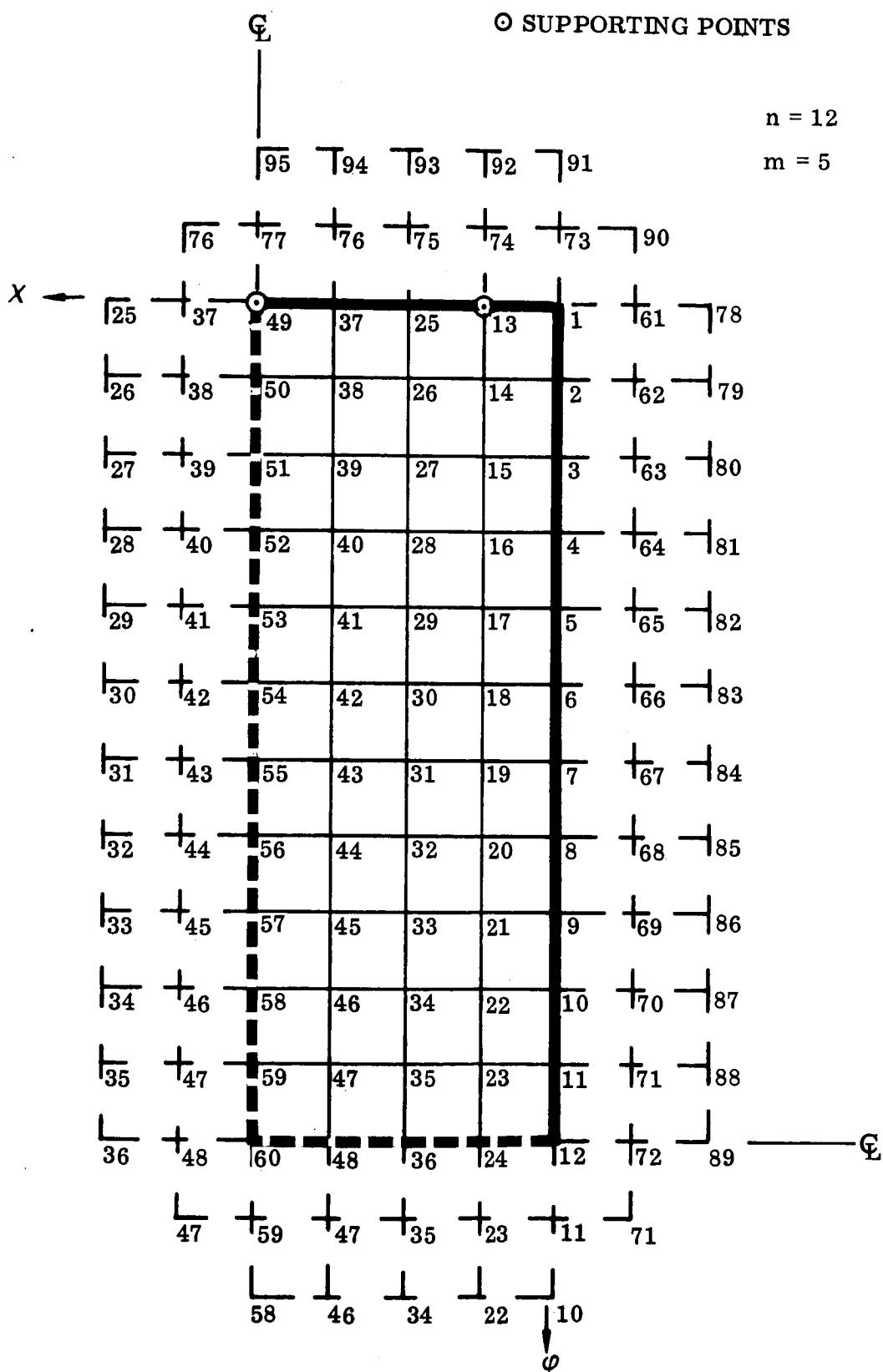


FIGURE 7. GRID PATTERN OVER QUARTER INSTRUMENT UNIT PANEL
USED IN THE FINITE DIFFERENCE PROGRAM

1. Solid Shell

<u>Symbols used in Analysis</u>	<u>Fortran Coding</u>	<u>Definitions</u>
a	A	Radius of curvature (in)
b	B	Length of a panel in x-direction (in)
φ_0	PHIO	Circumferential angle in φ -direction (rad)
ν	PNU	Poisson's ratio
E	E	Young's modulus of elasticity (lb/in ²)
ρ	RHO	Mass density per unit volume (lb-sec ² /in ⁴)
h	H	Shell thickness (in)
n	N	Number of grid points in φ -direction, (=12)
m	M	Number of grid points in x-direction, (=5)
—	NMODE	Number of modes sought
—	IOPT	= 1 for solid shell
$z_{sx}, z_{s\varphi}$	SZX, SZP	Distances from c.g. of the stiffeners to the middle surface of the panel in x-direction and φ -direction, (in.) numbers of values to be read in are controlled by an index, NEXT; their locations in the array by J1, J2 and K2.
$A_{sx}, A_{s\varphi}$	SAX, SAP	Cross sectional area of stiffener * in x-direction and φ -direction (in ²). Controlling indices are J1, J2, K2 and NEXT.
$I_{sx}, I_{s\varphi}$	SIX, SIP	Moment of inertial of stiffener, * in x-direction and φ -direction respectively, about its own centroidal axis (in ⁴). Controlling indices are J1, J2, K2 and NEXT.
K_x	BARKX	Spring constant of a point-support along the axis $x = 0$, (lbs/in) Controlling indices are I1, I2, and NXT1.
K_φ	BARKP	Same as above along the axis $\varphi = 0$, (lbs/in) Controlling indices are I3, I4, and NXT2.
W_x	WX	Weights along the boundary $x = 0$, (lbs) Controlling indices are I5, I6, and NXT3.

<u>Symbols used in Analysis</u>	<u>Fortran Coding</u>	<u>Definitions</u>
$W\varphi$	WP	Weights along the boundary $\varphi = 0$, (lbs) Controlling indices are I5, I6, and NXT4.
W	WT	Weights at interior points of the panel (lbs) Controlling index is JWT
i, j	IM1, IM2	Row and column number, respectively, of off-diagonal term of mass (internal) matrix.
—	M3	Number of off-diagonal mass term
(M_1) i, j, $i \neq j$	AM3	Actual element that corresponds to IM1 and IM2 in the (internal) mass matrix. Number of this off-diagonal terms is limited to six.
—	IETBL, NEC	Row number and column number, respec- tively of the matrix element (E) table. The E table is generated according to the definition of Table 2, Reference 2, for all grid points. A particular element of the table is to be transferred to a particular location in the coefficient matrix A.
—	NXRO	Number of non-zero elements in a row of coefficient matrix.
—	NAC	Column number of the non-zero element in the coefficient matrix.
—	NR, NC, NC1 NC2, NC3	Defined in the diagram shown in INPUT FORMAT. For any other size of coefficient matrix, they are computed as follow

*NOTE: For a stiffener which is placed along one boundary of the panel, double the values of the corresponding cross-sectional area and the moment of inertia as input.

Referring to the diagram shown in the INPUT FORMAT,

For submatrix AA1,

$$\begin{aligned} NR &= MN \\ NC &= 2MN + 3M + 3N + 1 \\ NC1 &= MN \\ NC2 &= 2M + 2N + 1 \\ NC3 &= MN + M + N \end{aligned}$$

For submatrix AA2,

$$\begin{aligned} NR &= 2M + 2N + 1 \\ NC &= MN + 2M + 2N + 1 \\ NC1 &= MN \\ NC2 &= 2M + 2N + 1 \end{aligned}$$

For submatrix AA3,

$$\begin{aligned} NR &= MN + M + N \\ NC &= 2MN + 3M + 3N + 1 \\ NC1 &= MN \\ NC2 &= 2M + 2N + 1 \\ NC3 &= MN + M + N \end{aligned}$$

2. Sandwiched Shell

The definitions made for the solid shell apply to the sandwiched shell with the following exceptions and additional definitions.

<u>Symbols used in Analysis</u>	<u>Fortran Coding</u>	<u>Definition</u>
h	H	Total thickness of outer and inner facings.
c	CORE	Thickness of core.
<u>m</u>	BARM	Mass per unit area of the panel.
<u>—</u>	IOPT	= 2. For sandwiched panel.

The key punch format for the input data and the program listings are attached to the report as APPENDIX A. Typical modal data obtained through the finite difference computes program for the segmented instrument unit panel is shown in Figure 8.

3rd MODE

$f = 518 \text{ CPS}$

$K = 2430 \text{ lb./in.}$

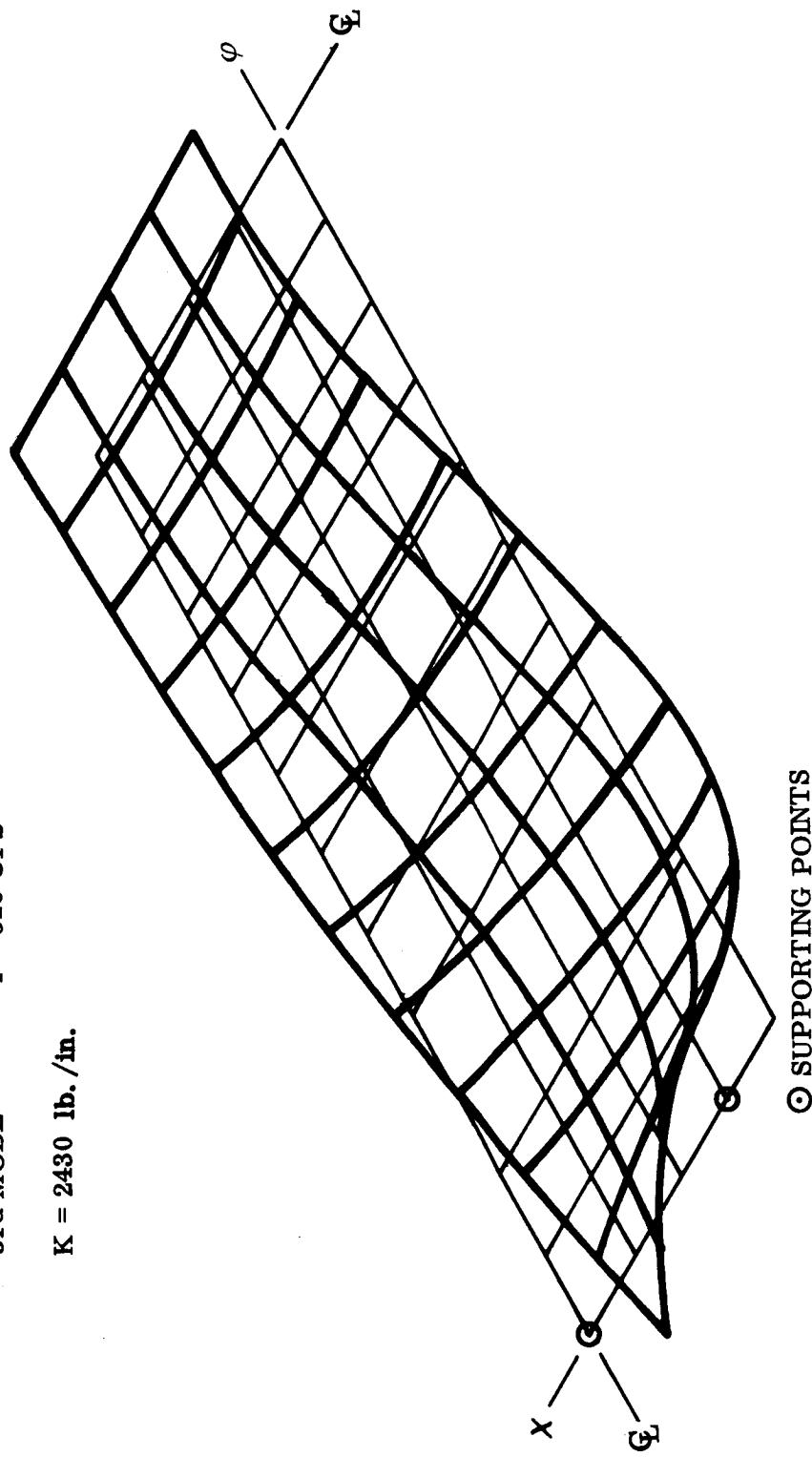


FIGURE 8. TYPICAL ANALYTICAL MODAL PATTERN REPRESENTING
A QUARTER OF THE INSTRUMENT UNIT PANEL

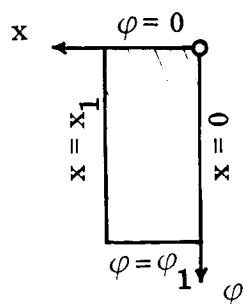
FORMULATION OF THE FINITE DIFFERENCE COMPUTER PROGRAM

In order that the user may adapt the finite difference computer program to his specific need, the present subsection describes the detail formulation and organization of the program. With this information, it is expected that the user will be able to organize a coefficient matrix for a shell panel structure with arbitrary boundary and support conditions. Applying the coefficient matrix as input data, the finite difference computer program may be used to generate the desired modal information.

Basic Grid Pattern Setup

The following procedures may be used to generate the basic grid pattern for a cross-stiffened cylindrical panel with arbitrary dimensions and boundary conditions.

1. Develop the curved panel into a plane surface, draw the boundary lines and mark down the location of the edge point supports, concentrated weight attachments, and center line of the stiffeners.
2. Mark all axes of symmetry, if any.
3. Set the coordinate axes which coincide with two boundary lines as shown in the following figure.



4. Cover the panel proper with two sets of equidistant grid lines $x = \text{constant}$ and $\phi = \text{constant}$. The grid pitch is Δx in the x direction and $a\Delta\phi$ in the ϕ direction, where a is the radius of the cylindrical panel.

Use the following guide lines:

- a. Each boundary line, axis of symmetry or the center line of a stiffener coincides with one of the grid lines.
- b. Each point support or concentrated weight attachment coincides with a grid point (the intersection of two grid lines) if possible. Otherwise locate the said point close to a grid point.

- c. Make the grid pitches Δx and $a\Delta\varphi$ equal or close to being equal.
 - d. For a panel with one axis of symmetry, only half of the panel need be considered. For a panel with two axes of symmetry only one quarter of the panel need be considered in order to obtain symmetrical mode shapes.
5. Assign numbers to each grid point on the part of the panel including points on the boundary lines and the axes of symmetry (see Figure 7). The station numbers start from 1 to (nxm) , where n is the total number of grid lines in φ -direction while m is the total number of grid lines in x direction. In Figure 7, it may be seen that $n = 12$ and $m = 5$ for the grid pattern under consideration.
6. Set up the necessary exterior stations according to the following rules:
- a. Extend two grid pitches from the points on the panel edges.
 - b. Extend one grid pitch in each direction from the corner points of the part of the panel concerned.
 - c. Extend two grid pitches from the points on the axes of symmetry.
7. Assign numbers to exterior stations starting from $(nxm) + 1$ according to the following order.
- a. Points one pitch away from the edges in the x -direction (station 61 - 72 in Figure 7).
 - b. Points one pitch away from the edges in the φ -direction station 73 - 77 in Figure 7).
 - c. Points two pitches away from the edges in the x -direction (station 78 - 89 in Figure 7).
 - d. Points one pitch in each direction away from the corner points which are the intersections of two edges (station 90 in Figure 7).
 - e. Points two pitches away from the edges in the φ -direction (station 91 - 95 in Figure 7).
8. Identify the station numbers of the points beyond the axes of symmetry to be the same as those of their image points with respect to the axes of symmetry. (station 10 - 11, 22 - 23, 25 - 48, 58 - 59, 71, 76 in Figure 7.)

9. Write down proper boundary conditions at each point on the edges. Detailed description of the boundary conditions is shown on Page 25 through Page 26 of Reference 2.

Formulation of the Coefficient Matrix

After the grid pattern has been completed, the procedures shown on Page 39 through Page 40 of Reference 2 may be used to generate the coefficient matrix [A]. The simplified finite difference operators are given below. The definitions of the index numbers are given in the Index Table on Page 41, Reference 2.

1. Equilibrium Equation Operator. Apply to all stations on the part of the panel including the boundary points (sta. 1 - 60 in Figure 7).

0	0	E6	0	0	(A1)			
0	E5	E4	E5	0				
E3	E2	E8 (w_i)	E2	E3				
0	E5	E4	E5	0				
0	0	E6	0	0				
<table border="1" style="width: 100%; border-collapse: collapse;"> <tr> <td style="padding: 5px;">E9</td> <td style="padding: 5px;">E7 (ϕ_i)</td> <td style="padding: 5px;">E9</td> </tr> </table>						E9	E7 (ϕ_i)	E9
E9	E7 (ϕ_i)	E9						

In the above table, the symbols E2 through E9 indicate the coefficients of the finite difference equation. They are identified on P. 31, 32, Reference 2. The symbols used in the above and following tables may also be identified with the corresponding items in Table 2, Reference 2.

2. Boundary Condition Operator for V_X

The following operator is applied to points along the edge $X = 0$. (Sta. 1 - 12 in Figure 7).

0	E67	0	E11	0	(A2)
E66	E36	E10 (w_i)	E12	E1	
0	E67	0	E11	0	

This operator may be applied to the points along the edge $X = X_1$. In the latter case, a negative spring constant is to be entered as input for each spring support.

3. Boundary Condition Operator for $M_{X\varphi}$

The condition is applicable to the corner points which are the intersections of two edges. Enter positive spring constants for the corners $X = 0, \varphi = 0$ and $X = X_1, \varphi = \varphi_1$. Enter negative spring constants for the corners $X = 0, \varphi = \varphi_1$ and $X = X_1, \varphi = 0$. (Sta. 1 in Figure 7).

E66	0	E1	(A3)
0	E68 (w_i)	0	
E1	0	E66	

4. Boundary Condition Operator for $M_{X\varphi}$

The condition is applicable to points along the boundary lines $X = 0$ and $X = X_1$. (Sta. 1 through 12 in Figure 7).

0	E14	0	(A4)
E1	E15 (w_i)	E1	
0	E14	0	

5. Boundary Condition Operator for V_φ .

The condition is applicable to points along the edges $\varphi = 0$, (positive spring constant input) and $\varphi = \varphi_1$ (negative spring constant input). (Sta. 1, 13, 25, 37, 49 in Figure 7).

0	E1	0	
E17	E18	E17	
0	E16 (w_i)	0	(A5)
E37	E69	E37	
0	E66	0	

6. Boundary Condition Operator for M_φ .

The condition is applicable to points along the edges $\varphi = 0$ and $\varphi = \varphi_1$. (Sta. 1, 13, 25, 37, 49 in Figure 7).

0	E1	0	
E19	E20 (w_i)	E19	(A6)
0	E1	0	

7. Compatibility Condition Operator.

Apply only to the interior points. (Sta. 14-24, 26-36, 38-48, 50-60 in Figure 7).

E22	E21 (w_i)	E22		
0	0	E28	0	0
0	E27	E26	E27	0
E25	E24	E23 (Φ_i)	E24	E25
0	E27	E26	E27	0
0	0	E28	0	0

(A7)

8. Boundary Condition Operator for S_X .

Apply to all points along the edges $X = 0$ and $X = X_1$ except the corner points. The following tables are used for points not on the axis of symmetry. (Sta. 2 through 11 in Figure 7).

E64	0	E30		
0	0	0		
(w_i)				
E30	0	E64		

E66	0	E1		
0	0	0		
(Φ_i)				
E1	0	E66		

(A8a)

Apply the following to points on the axis of symmetry (Sta. 12 in Figure 7).

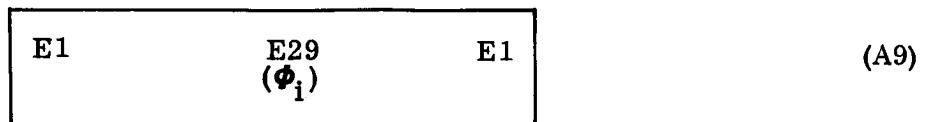
E64	0	E30		
E30	0	(w_i)	E64	

E66	0	E1		
E1	0	(Φ_i)	E66	

(A8b)

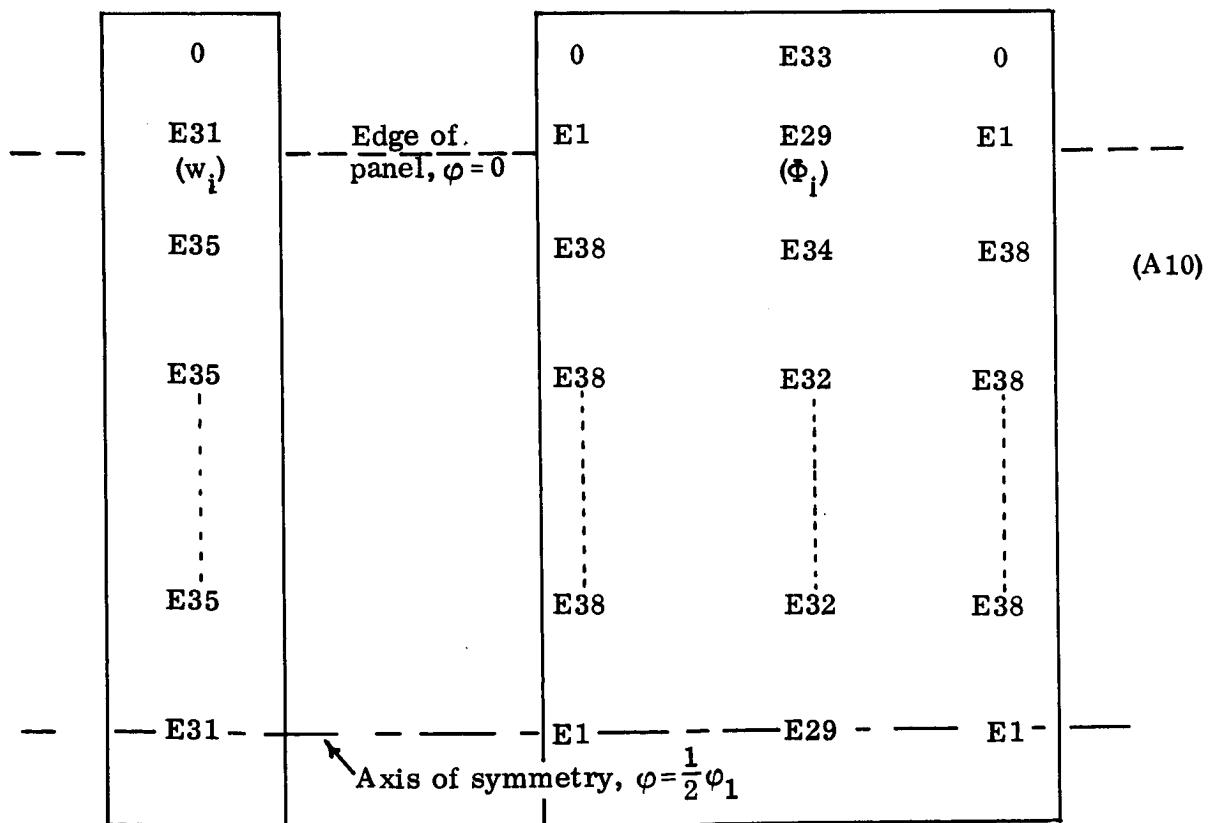
9. Boundary Condition Operator for N_φ .

Apply to points along the edges $\varphi = 0$ and $\varphi = \varphi_1$ where there is no spring support. (Sta. 1, 25, 37 in Figure 7).



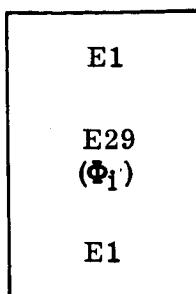
10. Boundary Condition Operator for Displacement v.

The condition specifies that no tangential displacement is admissible at the spring supports. Apply to points along the edges $\varphi = 0$ with spring supports. (Sta. 13 and 49 in. Figure 7). This condition can be used only if an axis of symmetry exists as shown below:



11. Boundary Condition Operator for N_x .

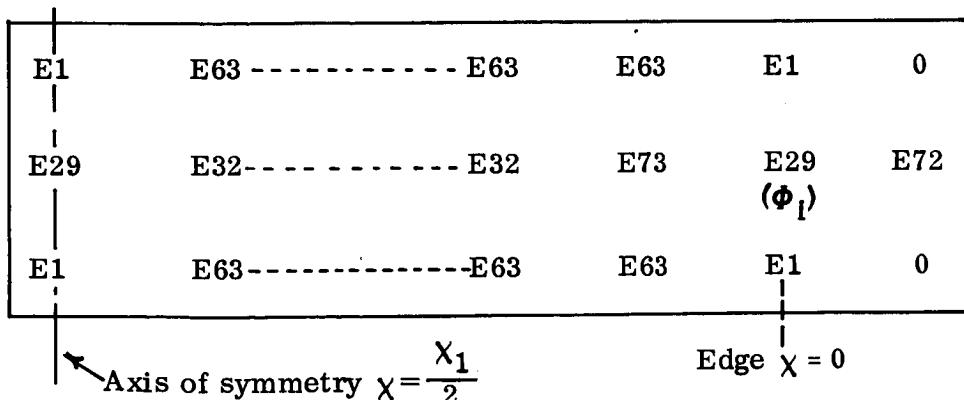
Apply to points along the edges $x = 0$ and $x = x_1$ without spring support.
(Sta. 1 through 12 in Figure 7).



(A11)

12. Boundary Conditions Operator for u .

Apply to points along the edge $x = 0$ with spring supports. This condition can be used only if there is an axis of symmetry in the φ direction.



(A12)

where $E72$ and $E73$ are two new index numbers which are not given in Table 2,
Reference 2. They are defined as:

$$E72 = -\nu \lambda^2 \frac{K}{K_\varphi}$$

$$E73 = -4 + \nu \lambda^2 \frac{K}{K_\varphi}$$

This condition is not used in the computer program shown in Appendix A since no support exists at $x = 0$.

13. Boundary Condition Operator for $N_{\varphi X}$.

Apply to points along the edges $\varphi = 0$ and $\varphi = \varphi_1$ except the corner points. The following table is used for point not on an axis of symmetry. (Sta. 13, 25, 37 in Figure 7).

E66	0	E1	
0	0 (Φ_i)	0	(A 13a)
E1		E66	

The following table is used for point on the axis of symmetry. (Sta. 49 in Figure 7.)

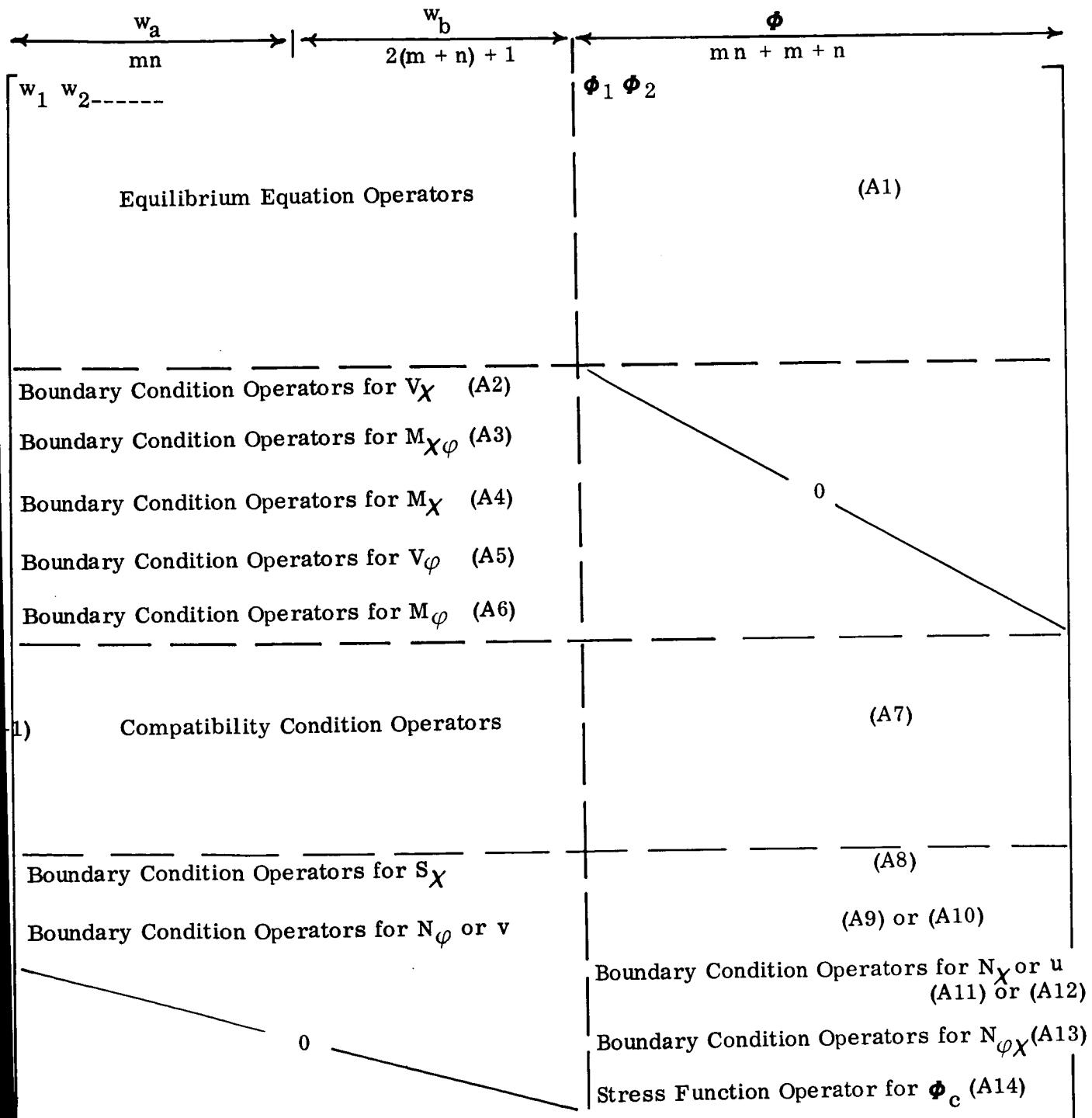
E66	E1	
0 (Φ_i)	0	(A 13b)
E1	E66	

14. Stress Function Operator for Φ_c .

Apply to one selected point on the panel. (Sta. 60 in Figure 7.)

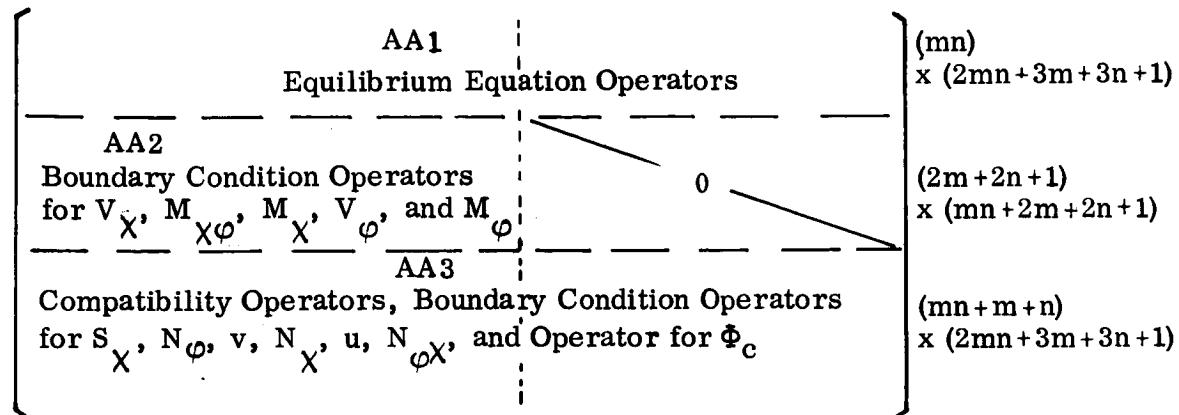
1 (Φ_i)	
-------------------	--

The following diagram shows the details of organizing the coefficient matrix. The diagram may be compared with a similar diagram shown on page 40, Reference 2.



Preparation of Coefficient Matrix Input Data

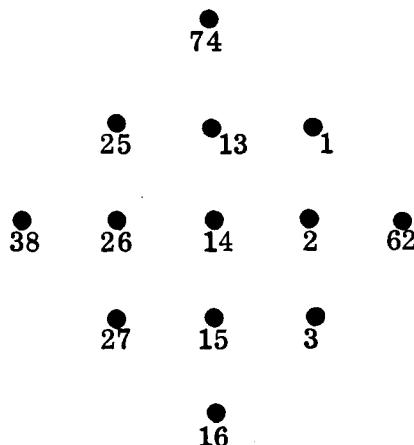
The coefficient matrix is read in as three submatrices, named "AA1", "AA2", and "AA3", which are shown below. In the computer program, the submatrices are generated in the order: AA3, AA1, AA2, to facilitate data reduction:



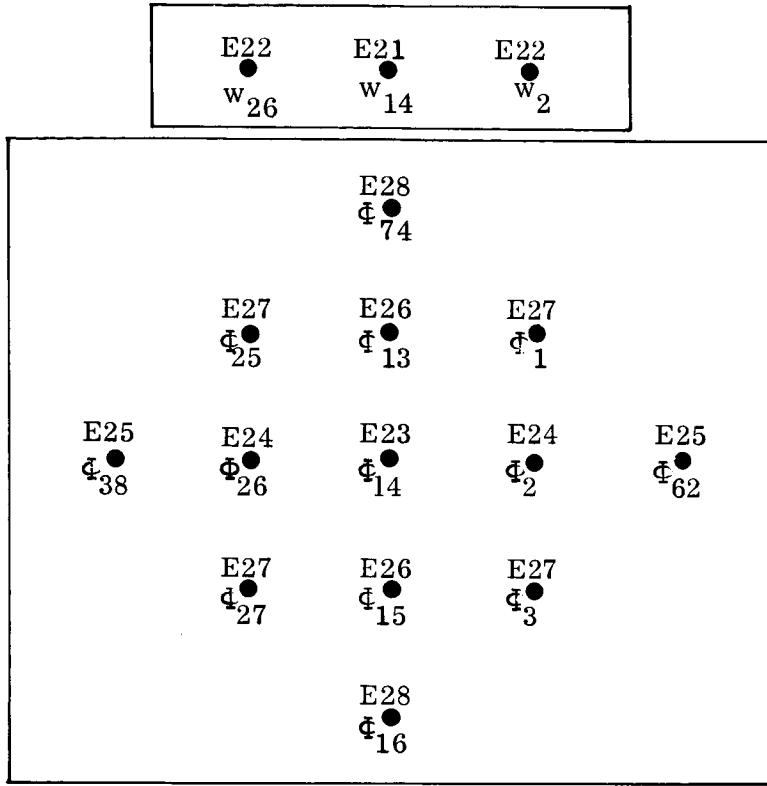
Corresponding to the grid pattern shown in Figure 7, where $m=5$, $n=12$, the sizes of the three submatrices are (60×172) , (35×95) , and (77×172) , respectively. In order to explain the coefficient matrix input in detail, two examples showing the generation of a row of the submatrix are illustrated below.

Example 1. The first row of AA3 matrix represents the compatibility condition (A7) at point 14. The finite difference operators and the related grid point stations are shown below:

Grid Pattern in the neighborhood of point (14):



The corresponding coefficient of w_i and Φ_i are:



From the above diagrams, the 96th row of the coefficient matrix, i.e., the first row of AA3 matrix, may be generated in the following manner. There is a total of 16 non-zero elements in this row.

Column No.	2 14 26 96 97 98 108 109 110 111 120 121 122 133 157 169
Index No.	E22 E21 E22 E27 E24 E27 E26 E23 E26 E28 E27 E24 E27 E25 E25 E28

Note that Column Number of $w_r = r$, i.e., the grid number is the matrix column number.

Column Number of $\phi_r = r + (mn + 2m + 2n + 1) = r + 95$, i.e., the grid number plus 95 is the matrix column number.

The input data card which is identified as (AA30001A) in Appendix A is read as:

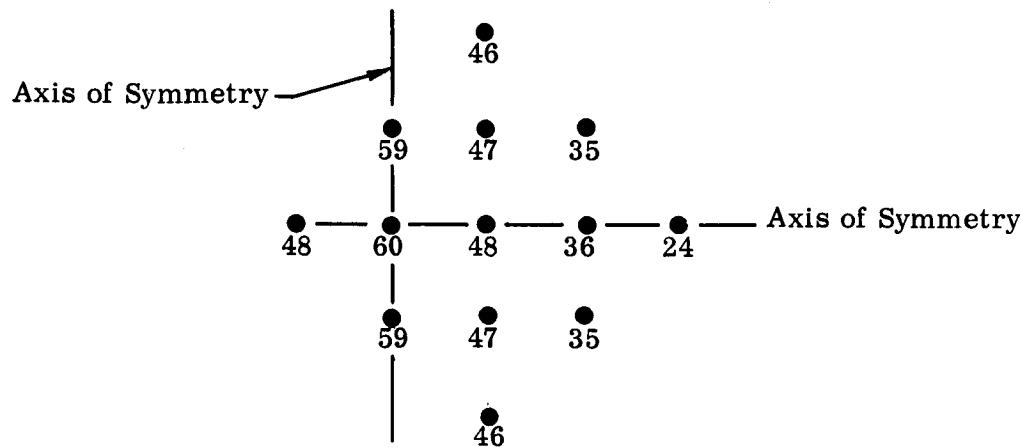
016 (002022) (014021) (026022) (096027) (097024) (098027) (108026) (109023) ...

where the parentheses are added for convenience in reading. In the card, the first number (016) indicates the total number of non-zero elements appearing in this row. The second number (002) is the column number of the first non-zero element, while the third number (022) is the corresponding index number. The fourth and fifth numbers are the column number and index number of the second non-zero element of the row,

and so on. The station number (14) of the center point to which the finite difference operator is applied is entered as the first number of the input data card AA300000, which reflects the row number of the matrix element on the (E) table generated.

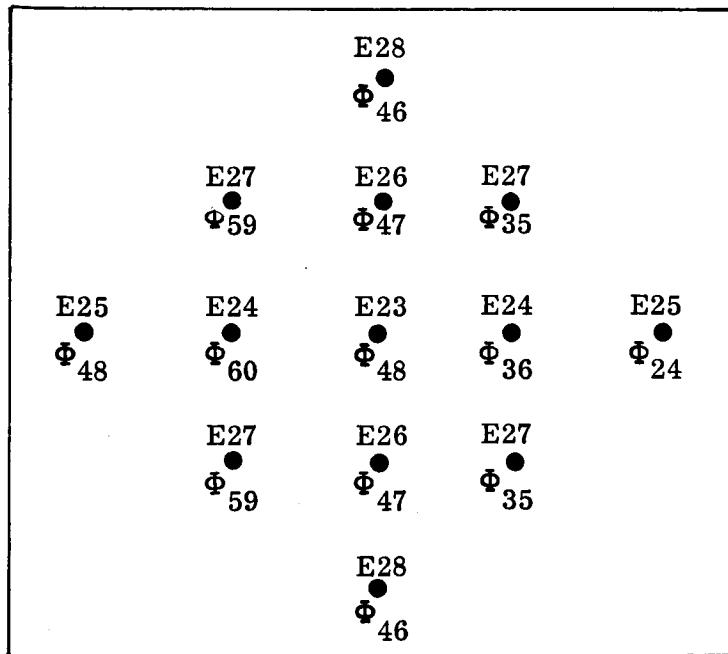
Example 2. When the finite difference operator is applied to a point one or two pitches away from the axis of symmetry, some new index numbers will be assigned. Consider the compatibility condition (A7) applied to grid point 48 as an example:

Grid Pattern in the neighborhood of point (48):



The corresponding coefficients for w_i and ϕ_i are

E22	E21	E22
w_{60}	w_{48}	w_{36}



Corresponding to the above operators, the 33rd row of the AA3 matrix is:

Col. No.	36	48	60	119	130	131	141	142	143	154	155
Index No.	E22	E21	E22	E25	(E27+E27)	E24	(E28+E28)	(E26+E26	(E23+E 25)	(E27+E27)	E24

The total number of non-zero elements in this row is 11. Referring to Table 2, Reference 2, the following symbols are defined:

$$E61 = 2 \times E27$$

$$E62 = 2 \times E28$$

$$E60 = 2 \times E26$$

$$E52 = E23+E25$$

The corresponding input for the 33rd row of AA3 matrix card (AA30033) is

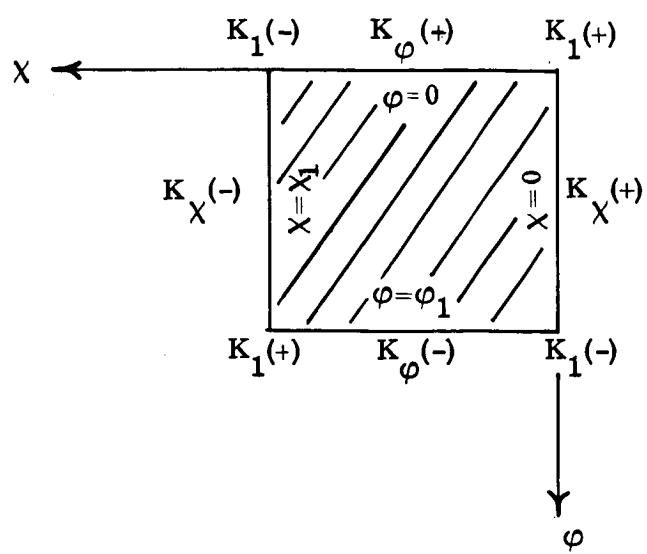
011 (036022)(048021)(060022)(119025)(130061)(131024)(141062)(142060)(143052)(154061)
(155024).

The 33rd number in input data cards AA30000 is 48. The latter number identifies the proper location of the (E) table from which the computed elements are selected.

Additional Information

The computer program described in Appendix A deals with a curved panel with double axes of symmetry supported at discrete points along the edges. The corresponding grid pattern is fixed as shown in Figure 7. The following suggestions are listed in order to modify the program for application to other structures:

1. Change dimension and equivalence statements to fit the requirements.
Evidently, all dimensions dependent on the grid sizes m, n , are to be modified.
2. For a panel with either one or none axis of symmetry, modify the statements used to compute the grid pitches Δx and $a\Delta\varphi$.
3. The boundary conditions $u = 0$ or $v = 0$ can be applied only if there is an axis of symmetry in φ or x direction, respectively. If the condition $u = 0$ is used, two new index numbers, E72 and E73, should be defined in the program.
4. The sign of the spring constant input of a point support is dependent on the specific edge where the support is located. The sign may be determined using the following diagram.



SECTION III

THE RESPONSE OF STIFFENED SHELL STRUCTURES TO TRANSIENT AND IMPULSIVE LOADINGS

The objective of this analysis is to develop a technique for handling stiffened shell structure response problems involving transient and impulsive loads. For this purpose, the normal-mode method of analysis is developed. The response solutions for the displacements and stresses are expressed in terms of a series of normal modes of free vibration for the built-up shell structure. With the use of energy expressions and Lagrange's equations, a set of uncoupled equations for the generalized coordinates is obtained. The solutions to these equations are in terms of time and space integrals.

A method is also developed to determine the normal modes and natural frequencies for a mass attached structure using natural frequency and modal data of the same structure without mass attachments. Transient response solution is obtained for the complete structure with mass attachments. The advantage of this procedure is that an uncoupled set of dynamic equations is obtained for the solution of the transient response problem involving the mass attached structure.

Detail programming information related to this section is presented in Appendix B of this report.

TRANSIENT RESPONSE SOLUTION

In this subsection, analytical solution for the transient response of built-up shell structures is developed. Details of the computer program are described. The computer program has been checked-out considering the response of the instrument unit shell model. In order to determine the convergence properties of this method, a test case of simply supported cylindrical shell is also illustrated.

A. Theory

The Lagrange's equations in generalized coordinates, for a system with external and/or non-conservative forces, may be written in the following form:

$$\frac{d}{dt} \left(\frac{\partial T}{\partial \dot{q}_i} \right) - \frac{\partial (T - V)}{\partial q_i} = Q_i, \quad i=1, 2, \dots, n \quad (1)$$

where

- T = Total kinetic energy of the built-up structure
- V = Total potential energy of the structure
- $q_i(t)$ = i^{th} generalized coordinate for the i^{th} normal mode
- Q_i = i^{th} generalized component of the external and non-conservative forces

For the present problem, the non-conservative force is assumed to be a viscous damping force. Therefore,

$$Q_i = \iint_s [\vec{P} - (\lambda \rho h) \dot{\vec{U}}] \cdot \vec{U}_i ds_1 ds_2 \quad (2)$$

where

- P = External pressure acting on the system
- λ = Viscous damping coefficient/unit mass
- ρ = Material density
- h = Thickness
- s_1, s_2 = Orthogonal coordinates covering the reference surface of the structure

The displacement vector $\vec{U}(s_1, s_2, t)$ is expressed in terms of the normal modes as

$$\vec{U}(s_1, s_2, t) = \sum_{i=1}^{\infty} q_i(t) \vec{U}_i(s_1, s_2) \quad (3)$$

Substituting the expression for \vec{U} from Equation (3) into Equation (2) yields

$$Q_i = \iint_s [\vec{P} - (\lambda \rho h) \sum_{j=1}^{\infty} q_j \vec{U}_j] \cdot \vec{U}_i ds_1 ds_2$$

In view of the orthogonality of normal mode displacements,

$$\begin{aligned} \iint_s \rho h (\vec{U}_j \cdot \vec{U}_i) ds_1 ds_2 &= 0, \quad \text{if } i \neq j \\ \iint_s \rho h (\vec{U}_i \cdot \vec{U}_i) ds_1 ds_2 &= N_i \neq 0, \quad \text{if } i=j \end{aligned} \quad (4)$$

the generalized force may be expressed in the following manner:

$$Q_i = \iint_S (\vec{P} \cdot \vec{U}_i) ds_1 ds_2 - \lambda N_i \dot{q}_i \quad (5)$$

Equation (1) is rewritten with the viscous damping term as follows:

$$\frac{d}{dt} \left(\frac{\partial T}{\partial \dot{q}_i} \right) - \frac{\partial (T - V)}{\partial q_i} + \lambda N_i \dot{q}_i = P_i \quad (6a)$$

where

$$P_i = \iint_S (\vec{P} \cdot \vec{U}_i) ds_1 ds_2 \quad (6b)$$

In general, since T and V are functions of \vec{U} and \vec{U} , the kinetic and potential energy terms are quadratic functions of the generalized coordinates \dot{q}_i and q_i , respectively. Detail development of these terms for a shell of revolution with ring stiffeners is given later in the section. At the present, only the general forms of the expressions are considered. Terms of Equation (6a) are expressed by:

$$\frac{d}{dt} \frac{\partial T}{\partial \dot{q}_i} = \sum_{j=1}^{\infty} m_{ij} \ddot{q}_j \quad (7a)$$

and

$$\left(\frac{\partial V}{\partial q_i} \right) = \sum_{j=1}^{\infty} k_{ij} \dot{q}_j \quad (7b)$$

In general, the generalized mass and spring terms are of the following form:

$$m_{ij} = \iint_S |_{ij} ds_1 ds_2 \quad (8a)$$

$$k_{ij} = \iint_S |_{ij} ds_1 ds_2 \quad (8b)$$

The m_{ij} terms are derived in Subsection 1.B for a specific case. It is observed that, since m_{ij} and k_{ij} are derived from the free vibration modes of a built-up structure, they satisfy the following conditions:

$$m_{ij} = k_{ij} = 0, \text{ for } i \neq j$$

and

$$m_{ii} \neq 0, k_{ii} \neq 0$$

With the use of Equations (7a, b), the Lagrange Equation (6a) for the case of undamped free vibrations becomes:

$$m_{ii} \ddot{q}_i + k_{ii} q_i = 0$$

The solution to the above homogeneous equation is $q_i = e^{i\omega_i t}$. Hence,

$$k_{ii} = \omega_i^2 m_{ii} \quad (9)$$

For the forced vibration case considering damping, Equation (6a) becomes:

$$m_{ii} \ddot{q}_i + \omega_i^2 m_{ii} q_i + \lambda N_i \dot{q}_i = P_i(t), \quad i=1, 2, \dots, \infty \quad (10a)$$

This uncoupled set of equations may be written as

$$\ddot{q}_i + \lambda \frac{N_i}{m_{ii}} q_i + \omega_i^2 q_i = \frac{1}{m_{ii}} P_i \quad (10b)$$

The solution to Equation (10b) is given by

$$\begin{aligned} q_i(t) &= \exp \left[\frac{\lambda t}{2} \frac{N_i}{m_{ii}} \right] (A_i \cos \gamma_i t + B_i \sin \gamma_i t) \\ &+ \frac{1}{m_{ii} \gamma_i} \int_0^t P_i(\tau) \exp \left[-\frac{\lambda N_i}{2m_{ii}} (t-\tau) \right] \sin \gamma_i (t-\tau) d\tau \end{aligned} \quad (11a)$$

where

$$\gamma_i = \left[\omega_i^2 - \left(\frac{\lambda N_i}{2m_{ii}} \right)^2 \right]^{1/2} \quad (11b)$$

and A_i and B_i are determined from initial displacement and velocity of the structure.

The displacement solution to the transient response problem is obtained by summation of the series in Equation (3) with the generalized coordinate expressions provided by Equation (11). The procedure for computing the generalized coordinate expressions $q_i(t)$ is as follows: For a given external pressure \bar{P} acting on the system, the generalized pressure terms $P_i(t)$ are computed using Equation (6b). The values of N_i and m_{ii} are obtained for a particular problem using Equations (4) and (8a), respectively. The expressions for A_i and B_i are obtained from the initial displacement and velocity of the structures. For the case when initial displacement and velocity of the structure are zero, $A_i = 0$, $B_i = 0$. The expression for $q_i(t)$ is then obtained by integrating either analytically or numerically, the second term on the right-hand side of Equation (11a).

B. Energy Expressions

In this subsection, kinetic energy expressions are given for a shell of revolution, as well as for ring stiffeners and mass attachments. These expressions are used to derive the generalized mass expression m_{ij} for the complete built-up structure. The kinetic energy expression for a shell of revolution is:

$$T_s = \frac{1}{2} \int_0^{\ell} \int_0^{2\pi} \rho h (\dot{\vec{U}} \cdot \dot{\vec{U}}) r d\theta ds \quad (12)$$

where $\dot{\vec{U}}$ is the velocity vector at any point of the shell. Substituting from Equation (3) the normal mode expansion terms into Equation (12) yields:

$$\begin{aligned} T_s &= \frac{1}{2} \int_0^{\ell} \int_0^{2\pi} \rho h \left(\sum_{i=1}^{\infty} \dot{q}_i \vec{U}_i \right) \cdot \left(\sum_{j=1}^{\infty} \dot{q}_j \vec{U}_j \right) r d\theta ds \\ \frac{\partial T_s}{\partial \dot{q}_i} &= \int_0^{\ell} \int_0^{2\pi} \rho h (\vec{U}_i \cdot \left(\sum_{j=1}^{\infty} \dot{q}_j \vec{U}_j \right)) r d\theta ds \\ &= \sum_{j=1}^{\infty} \dot{q}_j \iint \rho h (\vec{U}_i \cdot \vec{U}_j) r d\theta ds \\ \frac{d}{dt} \left(\frac{\partial T_s}{\partial \dot{q}_i} \right) &= \sum_{j=1}^{\infty} \ddot{q}_j \iint \rho h (u_i u_j + v_i v_j + w_i w_j) r d\theta ds \end{aligned} \quad (13)$$

where u , v , and w are the displacement components for the displacement vector \vec{U} .

The kinetic energy expression for all the ring stiffeners is:

$$T_R = \sum_R \int_0^{2\pi} \left\{ \frac{(\rho A)_R a_R}{2} \left[(\dot{u} + \beta e_n)^2 + (\dot{w} - \beta e_\theta)^2 + \dot{v}^2 \right] + \frac{\rho R a_R}{2} (I_\theta + I_n) \dot{\beta}^2 \right\}_{s=s_R} d\theta \quad (14)$$

where

- R = Subscript identifying rings
- A = Ring cross-sectional area
- a_R = Radius of ring Number R
- β = Rotation of shell at point of attachment of ring
- e_n = Eccentricity of ring centroid along the normal to the shell surface
- e_θ = Meridional distance from ring centroid to shell point of attachment
- I_θ = Area moment of inertial of ring cross-section about the axis parallel to meridian tangent of the shell
- I_n = Area moment of inertia of ring cross-section about the axis normal to surface of the shell
- s = Meridian location of ring stiffener

Substituting the normal mode expansion terms into Equation (14), there is obtained:

$$T_R = \sum_R \int_0^{2\pi} \left\{ \frac{\rho R A_R a_R}{2} \left(\left[\sum_{j=1}^{\infty} (u_j + \beta_j e_n) \dot{q}_j \right]^2 + \left[\sum_{j=1}^{\infty} (w_j - \beta_j e_\theta) \dot{q}_j \right]^2 + \left[\sum_{j=1}^{\infty} v_j \dot{q}_j \right]^2 \right) + \frac{\rho R a_R}{2} (I_\theta + I_n) \left(\sum_{j=1}^{\infty} \beta_j \dot{q}_j \right)^2 \right\}_{s=s_R} d\theta \quad (15)$$

which further yields:

$$\frac{d}{dt} \left(\frac{\partial T_R}{\partial \dot{q}_i} \right) = \sum_R \left[\sum_{j=1}^{\infty} \ddot{q}_j \int_0^{2\pi} \left\{ \rho R A_R a_R [(u_i + \beta_i e_n)(u_j + \beta_j e_n) + (w_i - \beta_i e_\theta)(w_j - \beta_j e_\theta) + (v_i v_j)] + \rho R a_R (I_\theta + I_n) \beta_i \beta_j \right\}_{s=s_R} d\theta \right] \quad (15)$$

The kinetic energy expression for all the mass attachments identified by subscript L is:

$$T_M = \sum_L \frac{1}{2} M_L (\dot{\vec{U}} \cdot \dot{\vec{U}}) \Bigg|_{\begin{array}{l} \theta = \theta_L \\ s = s_L \end{array}} \quad (16)$$

Therefore,

$$\frac{d}{dt} \left(\frac{\partial T_M}{\partial \dot{q}_i} \right) = \sum_L M_L \sum_{j=1}^{\infty} \ddot{q}_j (\vec{U}_i \cdot \vec{U}_j) \Bigg|_{\begin{array}{l} \theta = \theta_L \\ s = s_L \end{array}} \quad (17)$$

The individual terms from Equations (13), (15) and (17) are added together to obtain:

$$\frac{d}{dt} \left[\frac{\partial (T_S + T_R + T_M)}{\partial \dot{q}_i} \right] = \sum_{j=1}^{\infty} m_{ij} \ddot{q}_j \quad (18a)$$

where

$$\begin{aligned} m_{ij} &= \int_0^L \int_0^{2\pi} \rho h (u_i u_j + v_i v_j + w_i w_j) r d\theta ds \\ &+ \sum_R \int_0^{2\pi} \left\{ \rho A_R a_R \left[(u_i + \beta_j e_n) (u_j + \beta_j e_n) \right. \right. \\ &\quad \left. \left. + (w_i - \beta_i e_\theta) (w_j - \beta_j e_\theta) + (v_i v_j) \right] \right. \\ &\quad \left. + \rho R a_R (I_\theta + I_n) \beta_i \beta_j \right\}_{s=s_R} d\theta \end{aligned}$$

$$+ \sum_L M_L (u_i u_j + v_i v_j + w_i w_j) \Bigg|_{\begin{array}{l} \theta = \theta_L \\ s = s_L \end{array}} \quad (18b)$$

If the normal modes are considered for a shell with ring stiffeners and mass attachments, these normal mode terms will yield the condition that

$$m_{ij} = 0 \text{ for } i \neq j$$

$$m_{ii} \neq 0$$

And Equations (18a, b) become:

$$\frac{d}{dt} \left(\frac{\partial T}{\partial \dot{q}_i} \right) = m_{ii} \ddot{q}_i \quad (19a)$$

and

$$\begin{aligned} m_{ii} &= \int_0^R \int_0^{2\pi} \rho h (u_i^2 + v_i^2 + w_i^2) r d\theta ds \\ &+ \sum_R \int_0^{2\pi} [\rho R A_R a_R \left[(u_i + \beta_i e_n)^2 + (w_i - \beta_i e_\phi)^2 + v_i^2 \right] \\ &+ \rho R a_R (I_\phi + I_n) \beta_i^2] \Big|_{\substack{s=s_R \\ s=s_L}} d\theta \\ &+ \sum_L M_L (u_i^2 + v_i^2 + w_i^2) \Big|_{\substack{\theta=\theta_L \\ s=s_L}} \end{aligned} \quad (19b)$$

For the case of rotationally symmetric built-up shell structures, displacement vector $\vec{U}(s_1, s_2, t)$ of Equation (3) above can be expressed in terms of the three displacement components u, v, w and Equation (3) written as:

$$\vec{U}(s, \theta, t) \equiv \begin{Bmatrix} u \\ v \\ w \end{Bmatrix} = \sum_{i=1}^n q_i(t) \begin{Bmatrix} u_i(s) \cos n_i \theta \\ v_i(s) \sin n_i \theta \\ w_i(s) \cos n_i \theta \end{Bmatrix} \quad (20)$$

where n_i = circumferential wave number for the i^{th} mode. In Equation (20), summation of N normal modes is considered, $i=1, 2, \dots, N$ identify the natural frequencies of the structure in an ascending order. Substituting the i^{th} terms of u, v, w from Equation (20) into Equation (19b) and integrating with respect to θ the following expression is reached:

$$m_{ii} = \pi \int_0^R r \rho h (u_i^2 + v_i^2 + w_i^2) ds$$

$$\begin{aligned}
& + \frac{\Sigma}{R} \pi \left\{ \rho_R A_R a_R \left[(u_i + \beta_i e_n)^2 + (w_i - \beta_i e_\theta)^2 + v_i^2 \right] \right. \\
& \left. + \rho_R a_R (I_\theta + I_n) \beta_i^2 \right\}_{s=s_R} + \sum_L M_L \left[(u_i^2 + w_i^2) \cos^2 n_i \theta_L \right. \\
& \left. + v_i^2 \sin^2 n_i \theta_L \right]_{s=s_L} \quad (21)
\end{aligned}$$

In Equation (21), the values of $u_i(s)$, $v_i(s)$, $w_i(s)$ must be known as a functional relation or in the form of numerical values. Correspondingly, the first term of Equation (21) may be integrated analytically or numerically. The computer program which performs the numerical integration will be described later in the section.

C. Transient Point Force Acting on Stiffened Shell Structure

The theory described in subsection (A) is applied to the transient response of a built-up shell structure acted upon by an arbitrary transient point force. The force is applied radially at location (s_o, θ_o) on the stiffened shell structure. To evaluate the generalized force P_i from Equation (6b), the radial component P_r of the load \vec{P} is taken as nonzero:

$$\begin{aligned}
P_i &= \int_0^\ell \int_0^{2\pi} P_r w_i(s, \theta) r d\theta ds \\
&= F_r(t) w_i(s_o, \theta_o) \quad (22)
\end{aligned}$$

where F_r is the total applied force given by Equation (22).

The expression for the generalized coordinate $q_i(t)$ is obtained by substituting P_i from Equation (22) into Equation (11a).

$$\begin{aligned}
q_i(t) &= \exp \left(-\frac{\lambda t}{2} \frac{N_i}{m_{ii}} \right) (A_i \cos \gamma_i t + B_i \sin \gamma_i t) \\
&+ \frac{w_i(s_o) \cos(n_i \theta_o)}{m_{ii} \gamma_i} \int_0^t F_r(\tau) \exp \left[-\frac{\lambda N_i}{2m_{ii}} (t - \tau) \right] \sin \gamma_i (t - \tau) d\tau \quad (23)
\end{aligned}$$

The integration of the second term of Equation (23) is performed numerically. The numerical integration subroutine will be described in Appendix B.

If the initial displacement and velocity of the shell structure are assumed to be zero, then in Equation (22),

$$A_i = 0, \quad B_i = 0$$

In order to illustrate the method, consider a damped sinusoidal point force acting on the stiffened shell structure. The force may be represented as:

$$F_r(t) = F_0 e^{-\alpha t} \cos \omega t \quad (24)$$

where

ω = forcing function frequency (rad/sec)

α = decay factor for the forcing function.

Substituting the right hand side of Equation (24) into Equation (23) yields:

$$q_i(t) = \frac{w_i(s_o) \cos(n_i \theta_o) F_o}{m_{ii} \gamma_i} \int_0^t \cos \omega \tau e^{-\alpha \tau} e^{-\frac{\lambda(t-\tau)}{2}} \sin \gamma_i(t-\tau) d\tau$$

The integral of this expression is given by

$$\begin{aligned} q_i(t) &= \frac{w_i(s_o) \cos(n_i \theta_o) F_o}{m_{ii} \gamma_i} e^{-\frac{\lambda t}{2}} \left\{ \frac{\sin \gamma_i t}{2} \left[e^{-\beta t} \left\{ \frac{-\beta \cos(\omega + \gamma_i)t + (\omega + \gamma_i) \sin(\omega + \gamma_i)t}{\beta^2 + (\omega + \gamma_i)^2} \right. \right. \right. \\ &\quad \left. \left. \left. + \frac{-\beta \cos(\omega - \gamma_i)t + (\omega - \gamma_i) \sin(\omega - \gamma_i)t}{\beta^2 + (\omega - \gamma_i)^2} \right\} + \beta \left\{ \frac{1}{\beta^2 + (\omega + \gamma_i)^2} + \frac{1}{\beta^2 + (\omega - \gamma_i)^2} \right\} \right] \right. \\ &\quad \left. + \frac{\cos \gamma_i t}{2} \left[e^{-\beta t} \left\{ \frac{\beta \sin(\omega + \gamma_i)t + (\omega + \gamma_i) \cos(\omega + \gamma_i)t}{\beta^2 + (\omega + \gamma_i)^2} \right. \right. \right. \\ &\quad \left. \left. \left. - \frac{\beta \sin(\omega - \gamma_i)t + (\omega - \gamma_i) \cos(\omega - \gamma_i)t}{\beta^2 + (\omega - \gamma_i)^2} \right\} - \left\{ \frac{(\omega + \gamma_i)}{\beta^2 + (\omega + \gamma_i)^2} - \frac{(\omega - \gamma_i)}{\beta^2 + (\omega - \gamma_i)^2} \right\} \right] \right\} \end{aligned} \quad (25)$$

where

$$\beta \equiv \alpha - \frac{\lambda}{2} \left(\frac{N_i}{m_{ii}} \right) \quad (26)$$

In case the damping is caused by the structural elements including the shell, rings and mass attachments, the following condition may be used:

$$N_i = m_{ii}$$

So that, γ_i and β in Equations (11b) and (26) are written as:

$$\gamma_i = \left[\omega_i^2 - \left(\frac{\lambda}{2} \right)^2 \right]^{1/2},$$
$$\beta = \left(\alpha - \frac{\lambda}{2} \right) \quad (27a, b)$$

D. Computer Program for the Transient Response of the Stiffened Shell Structure

A computer program is developed to determine the transient response of a stiffened shell structure. The program is designed for rotationally symmetric built-up shell structures. However, it may be adapted to handle other types of structures, if the normal mode and frequency data are available. The initial displacement and velocity of the structure are assumed to be zero.

The program is designed to handle any arbitrary time dependent point force acting on the shell structure. For this purpose, Equation (23), which involves numerical integration, is employed. The numerical values of the forcing function $F_r(t)$ must be given at a sufficient number of time intervals in order that the numerical integration subroutine may yield accurate response data. The $F_r(t)$ data is part of the input to the computer program. In the sample run described in subsection (E), a damped sinusoidal forcing function, Equation (24), is used. The program generated response data, $q_i(t)$, are compared with the values computed with the use of Equation (25), which was obtained through analytical integration.

The analytical expressions used in the computer program are given by Equations (20), (21), (23), (24), and (27a,b). The keypunch input data format and program listing are given in Appendix B. The input data parameters and the notations used in the program consist of the following:

TIN	= initial time, (sec.)
DELT	= increment of time, (sec.)
TEND	= end of the time, (sec.)
LAMDA (λ)	= damping coefficient
FFCT (F_r)	= forcing function at $t = 0$, (lbs.)
ALFA (α)	= decay coefficient
THO	= location of loading (circumferentially)
OMFC (ω)	= forcing function frequency (rad/sec)
NX, NY	= axial half wave number and circumferential wave number
U, V, W, BETA	= axial, circumferential, lateral, and rotatory displacement, respectively
ITAO	= number of incremental time to describe the history of forcing function
MT	= number of incremental time to describe the history of vibro-response + 1
TETA	= circumferential location where response is desired, (rad.)
NM	= number of modes considered
NS	= number of sections, a minus sign is attached if a ring stiffener exists at the rightmost end of the structure
NRESP	= number of points responses are sought
NSEG	= number of segments in the section
NSTAT	= total number of segments in the entire structure + 1
LOCFC	= segment at which the force is applied

$(LRESP(I), I=1, NRESP)$	= segments at which responses are sought
$\phi M(\omega_i)$	= modal frequency, (rad./sec.)
$NCH(n_i)$	= circumferential wave number
$NDEGR$	= number of degrees of the polynomial describing the meridian configuration
$AC(I), I=1, 7$	= coefficients of the polynomial
XA	= beginning point (left to right) of the section (in.)
XB	= ending point of the section, (in.)
$HA(h)$	= thickness of shell at the particular section (in.)
$RHOS(\rho)$	= material mass density for the section, ($lb\cdot sec^2/in^4$)
$KT2$	= 1, if ring exists only at the first segment of the section = 2, if ring exists at every segment in the section = 0, no rings in the section
$RHQ(\rho_R)$	= mass density of ring material ($lb\cdot sec^2/in^4$)
$EN(e_n)$	= distance, in the direction perpendicular to the shell surface, from shell point of attachment to the centroid of the ring cross section, (in.)
$EPHI(e_\phi)$	= meridional distance from shell point of attachment to the centroid of the ring cross section, (in.)
$IX(I_\phi)$	= area moment of inertia about the axis parallel to meridian tangent of the shell surface at the point of attachment, (in^4)
$IY(I_n)$	= area moment of inertia about the axis perpendicular to the surface of the shell, (in^4)
$RRAD(a_R)$	= principal radius of curvature of the ring, (in.)
$AREA(A_R)$	= cross sectional area, (in^2)
NR	= 0, cylindrical shell case - no stiffener = 1, general stiffened shell structure

Z_1 , Z_2 , Z_3 , Z_4 and FQ

These are the data punched out from General Stiffened Shell program. The only data used in the present program are Z_1 and FQ .

Z_1 = circumferential wave number

FQ = natural frequency (cps)

The response data generated by the computer program is plotted in Figure 9.

E. Transient Response Program Applied to Simply Supported Cylindrical Shell

The transient response of a simply supported cylindrical shell is considered in this subsection. Equations to compute the natural frequencies and normal modes for the cylindrical shell are given. The method to generate the input data for the transient response program is described. The particular set of geometric parameters for the cylindrical shell is selected to have comparable overall dimensions and identical material properties as the instrument unit scale model described previously.

The natural frequencies of a simply supported shell are given by the equation:

$$\omega_{nk}^2 = \frac{1}{12\left(\frac{a}{h}\right)^4\left(\frac{h}{\nu}\right)^2} \left\{ \left[n^2 + \left(k\pi \frac{a}{l} \right)^2 - 1 \right]^2 + \frac{12(1 - \nu^2)\left(\frac{a}{h}\right)^2 \left(k\pi \frac{a}{l} \right)^4}{\left[n^2 + \left(k\pi \frac{a}{l} \right)^2 \right]^2} \right\} \quad (28)$$

where

n = circumferential wave number

k = axial half wave number

The lowest n frequencies determined by Equation (28) are arranged in ascending order and numbered from $i = 1, 2, \dots, n$, such that

$$\omega_1 < \omega_2 < \dots < \omega_n \quad (29)$$

where corresponding to the i^{th} frequency ω_i ,

$$n = n_i, \quad k = k_i$$

Material: Aluminum
Transient Force Frequency = 87.2 cps
9 modes solution

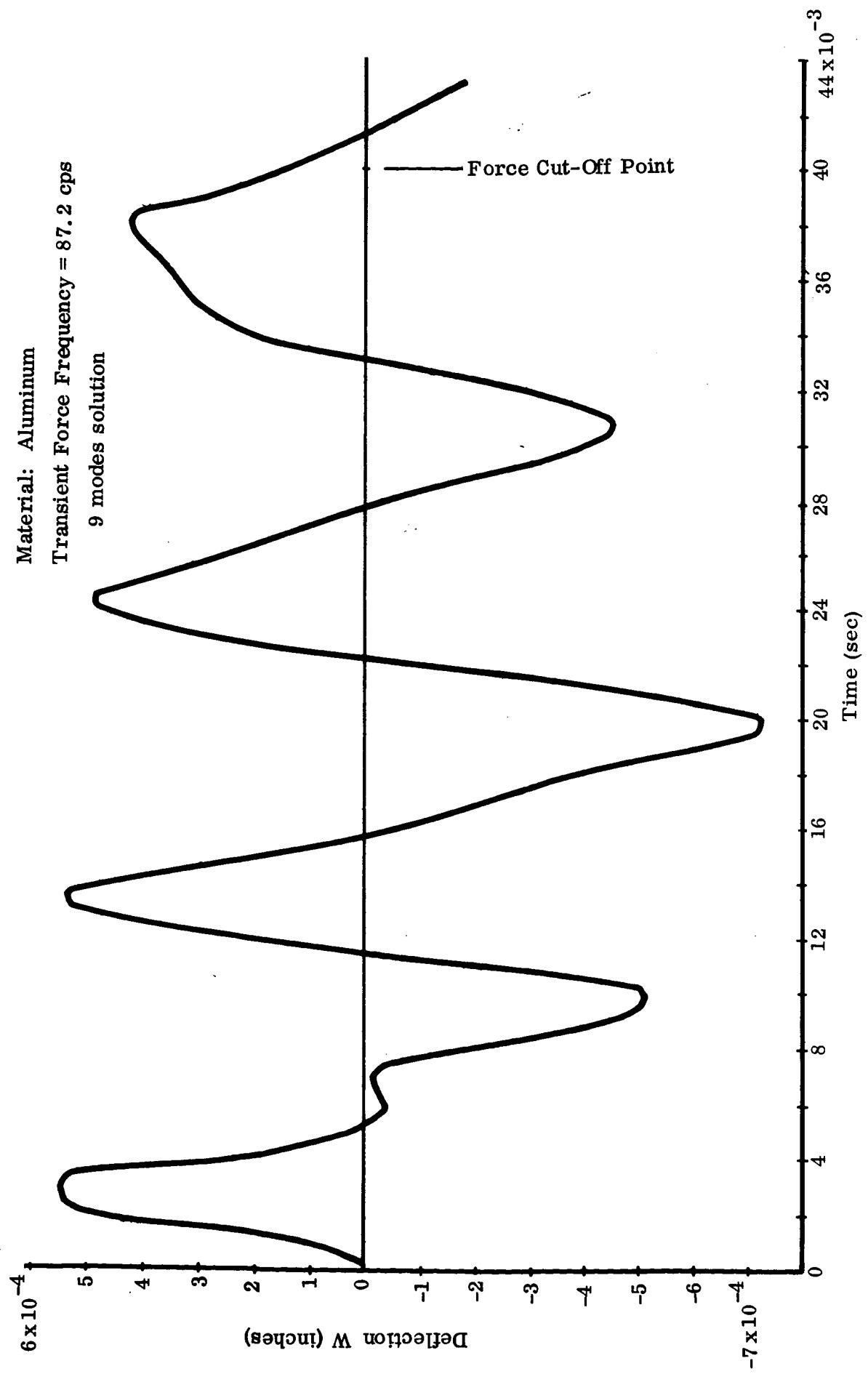


FIGURE 9. TRANSIENT RESPONSE OF INSTRUMENT UNIT STRUCTURE

The normal modes of a simply supported cylindrical shell are given by the expressions:

$$\left. \begin{array}{l} u = A \cos n\theta \cos \left(k_i \pi \frac{a}{l} x \right) \\ v = B \sin n\theta \sin \left(k_i \pi \frac{a}{l} x \right) \\ w = C \cos n\theta \sin \left(k_i \pi \frac{a}{l} x \right) \end{array} \right\} \quad \begin{array}{l} \text{for } 0 \leq x \leq \left(\frac{l}{a} \right) \\ 0 \leq \theta \leq 2\pi \end{array} \quad (30)$$

where, $x = \frac{s}{a}$, the normalized axial coordinate

Comparing the above expressions to normal mode terms in Equation (20), the mode displacements are obtained as shown below:

$$\left. \begin{array}{l} u_i = A_i \cos \left(k_i \pi \frac{a}{l} x \right) \\ v_i = B_i \sin \left(k_i \pi \frac{a}{l} x \right) \\ w_i = C_i \sin \left(k_i \pi \frac{a}{l} x \right) \end{array} \right\} \quad (31)$$

In Equations (31), the constants A_i and B_i are expressed in terms of C_i by the following relations:

$$A_i = \left\{ \frac{\left[-\nu \left(k_i \pi \frac{a}{l} \right)^2 + n_i^2 \right] \left(k_i \pi \frac{a}{l} \right)}{\left[\left(k_i \pi \frac{a}{l} \right)^2 + n_i^2 \right]^2} \right\} C_i \quad (32a)$$

$$B_i = \left\{ \frac{\left[(2 + \nu) \left(k_i \pi \frac{a}{l} \right)^2 + n_i^2 \right] n_i}{\left[\left(k_i \pi \frac{a}{l} \right)^2 + n_i^2 \right]^2} \right\} C_i \quad (32b)$$

These equations are obtained by substituting u , v , and w from Equation (30) into the differential equations for a cylindrical shell. Equations (28), (31) and (32a,b) are used to generate data for the frequencies and modal displacements for a given set of geometric parameters described below. These data are then used as input to the transient response program where the response can be computed at any point on the shell structure. In the sample run, the response was computed at the point where load was applied.

Geometric Parameters for Test Case

The simply supported cylindrical shell considered for the test run has the same overall dimensions and material properties as the instrument unit scale mode. These are:

$$\text{diameter } d = 39.0 \text{ inch } (a = 19.5 \text{ inch})$$

$$\text{thickness } h = 0.05 \text{ inch}$$

$$\text{length } l = 53.4 \text{ inch}$$

The material properties for aluminum are

$$\text{Young's modulus } E = 10.3 \times 10^6 \text{ psi}$$

$$\text{Poisson's ratio } \nu = 0.3$$

$$\text{Material density } \rho = 0.2591 \times 10^{-3} \text{ lb. sec.}^2/\text{in.}^4$$

In Figure 10 frequency spectra for $k = 1, 3, 5, 7, 9$, and 11 are plotted for the particular test case with geometric parameters described below. The lowest hundred frequencies are numbered as shown. The fundamental frequency corresponding to $k_1 = 1$ and $n_1 = 6$ is $\omega_1 = 461.6 \text{ rad. /sec.}, (73.6 \text{ cycles/sec.})$.

In Figure 11, the graph of radial displacement response as a function of time is shown. This response is for the case where a sinusoidal force is applied at a point on the middle span of the cylindrical shell. The forcing function frequency is identical to the fundamental frequency of the structure. The difference between the fundamental mode response and summation of the response for the lowest 18 modes is significant. The peak value of displacement for the 18-mode response is more than twice the peak for fundamental mode response alone. Convergence of the modal response is found to be quite rapid. The solution for the lowest 12 modes differed from the lowest 18 mode solution by only 0.5 percent for the peak value of displacement response.

Computer Program for Test Case

The analytical expressions used in this computer program are given by Equations (28), (31) and (32a,b). The nomenclature for input data are as follows:

A Radius of the cylinder (in)

H Thickness of the cylinder skin (in)

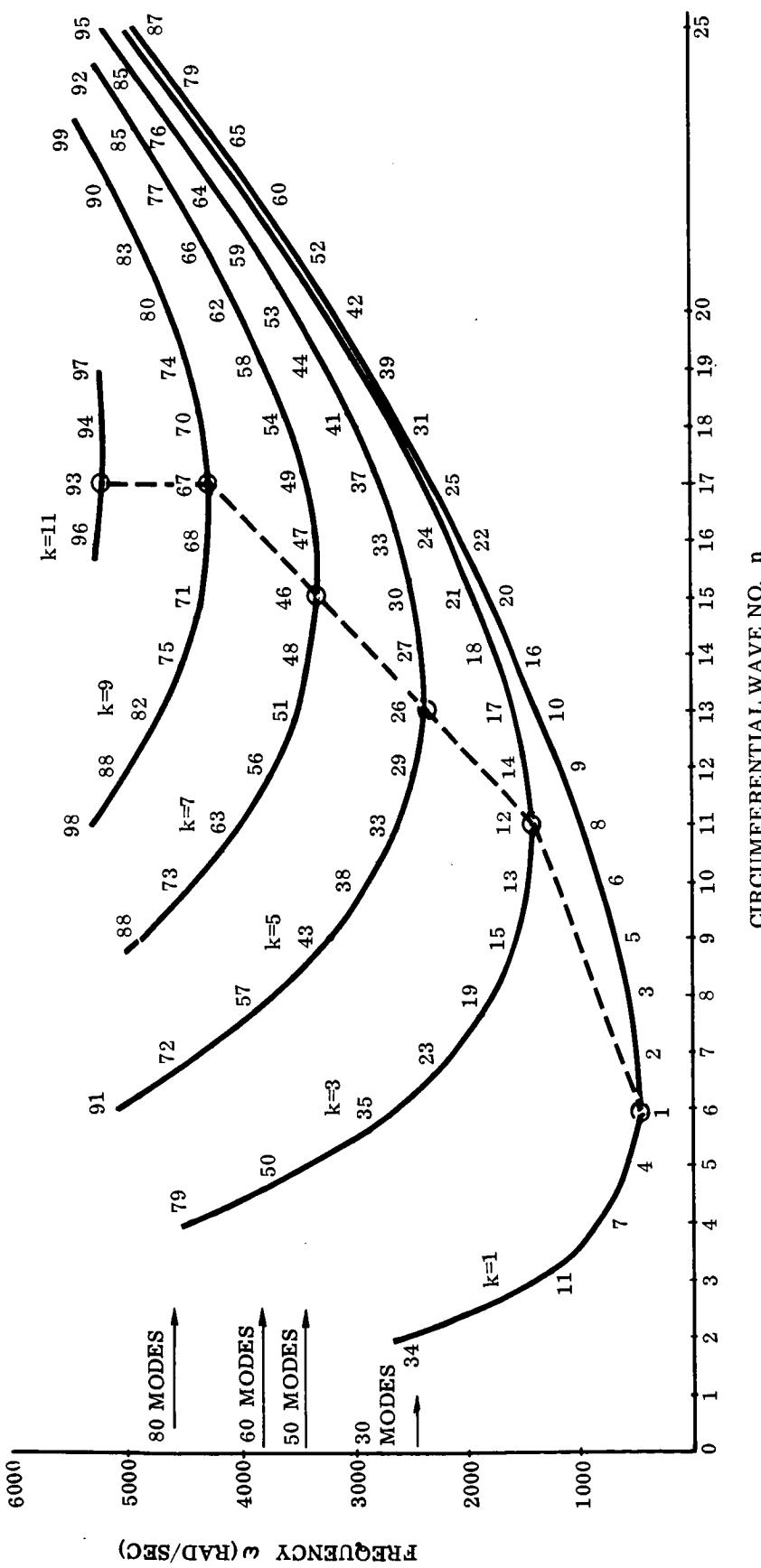


FIGURE 10. NATURAL FREQUENCIES FOR A SIMPLY SUPPORTED CYLINDRICAL SHELL

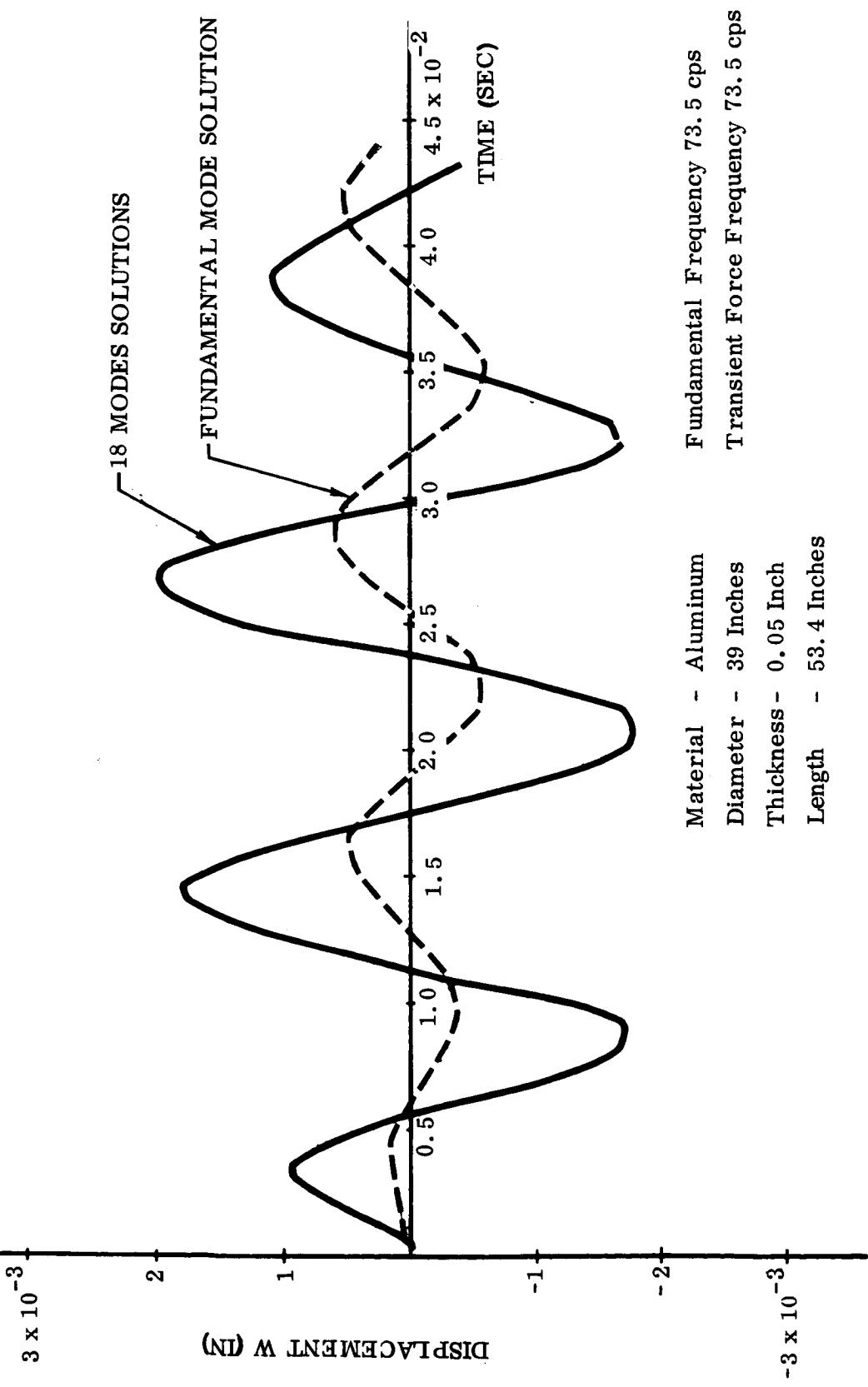


FIGURE 11. RESPONSE OF CYLINDRICAL SHELL STRUCTURE TO TRANSIENT LOAD

E	Young's modulus of elasticity (psi)
NU	Poisson's ratio (0)
RHO	Material density of cylinder skin ($\text{lb-sec}^2/\text{in}^4$)
L	Length of the cylinder (in)
KF	Highest axial half wave number of interest (0)
NF	Highest circumferential wave number of interest (0)
NKF	Highest number of mode of interest (0)
KINC	= 1 covers all axial half wave number of interest = 2 all even or all odd of axial half wave number (if KS = 1, odd; if KS = 2, even)
KS	Lowest axial half wave number of interest (0)
NS	Number of interior stations to determine the mode shape + 1 (0)

The keypunch input format is shown in Appendix B.

NATURAL FREQUENCIES AND NORMAL MODES OF SHELL STRUCTURES WITH MASS ATTACHMENTS

The mass attachments change the natural frequencies and mode shapes of the structural system. Even very small masses, which may change the natural frequencies only slightly, can cause significant changes of modal shapes.

The present method of approach is as follows: The normal modes and natural frequencies for a mass attached structure are obtained with the use of modal and frequency data for a structure without mass attachments. By the use of Lagrange's equations developed in the section, an eigenvalue matrix equation is formulated. The eigenvalues and eigenvectors of this matrix equation are then used to determine the frequencies and mode shapes of the mass attached structure. These data are then used as input to the transient response program. The advantage of this procedure is that an uncoupled set of differential equations is obtained which deals with the transient response of shell structures with attached masses.

A. Theory

The analytical approach to obtain the natural frequencies and mode shapes of a mass attached stiffened shell structure is based on the use of normal mode data for

the unloaded stiffened shell. For this purpose, Equation (10a) developed previously is modified to include the effect of additional mass attachments and the damping and forcing function terms are set equal to zero:

$$m_{ii} \ddot{q}_i + \omega_i^2 m_{ii} q_i + \left(\sum_{j=1}^{\infty} M_{ij} \ddot{q}_j \right) = 0 \quad (33)$$

where m_{ii} is given by Equation (21) and M_{ij} is identical to the last term of Equation (18b):

$$M_{ij} = \sum_L M_L (u_i u_j + v_i v_j + w_i w_j) \quad \left| \begin{array}{l} \theta = \theta_L \\ s = s_L \end{array} \right. \quad (34a)$$

For a rotationally symmetric shell structure, with the use of Equation (20), Equation (34a) is rewritten as:

$$M_{ij} = \sum_L M_L \left[(u_i u_j + w_i w_j) \cos n_i \theta_L \cos n_j \theta_L + v_i v_j \sin n_i \theta_L \sin n_j \theta_L \right]_{s=s_L} \quad (34b)$$

Corresponding to the free harmonic vibration motion of the complete system, the generalized coordinate q_i is given by:

$$q_i^{(r)}(t) = A_i^{(r)} \sin \omega_r t \quad (35)$$

where r represents the r^{th} mode of free vibration of the complete system. Substitution of q_i from Equation (35) into Equation (33) gives:

$$-\omega_r^2 m_{ii} A_i^{(r)} + \omega_i^2 m_{ii} A_i^{(r)} - \omega_r^2 \left(\sum_{j=1}^n M_{ij} A_j^{(r)} \right) = 0, \quad i = 1, 2, 3, \dots, n \quad (36)$$

Both sides of this equation are multiplied by $\left(\frac{\omega_1^2}{\omega_i^2 \omega_r^2 m_{ii}} \right)$ to obtain:

$$\left(\frac{\omega_1^2}{\omega_r^2} \right) A_i^{(r)} = \left(\frac{\omega_1^2}{\omega_i^2} \right) \left[A_i^{(r)} + \frac{1}{m_{ii}} \left(\sum_{j=1}^n M_{ij} A_j^{(r)} \right) \right], \quad i = 1, 2, \dots, n \quad (37)$$

Due to practical limitations in computation, only the first n modes are considered in Equation (37). It is also noted that $\omega_1 \dots \omega_i$ are the known frequencies of the unloaded shell, ω_r is the unknown frequency of the mass attached shell. The equation may be rewritten as:

$$\left(\frac{\omega_1^2}{\omega_r^2} \right) A_i^{(r)} = \left(\frac{\omega_1^2}{\omega_i^2} \right) \left[\sum_{j=1}^n \left(A_j^{(r)} + \frac{M_{ij}}{m_{ii}} A_j^{(r)} \right) \right], \quad i = 1, 2 \dots n \quad (38)$$

The corresponding matrix equation is:

$$\left(\frac{\omega_1^2}{\omega_r^2} \right) \begin{Bmatrix} A_1 \\ A_2 \\ \vdots \\ A_i \\ \vdots \\ A_n \end{Bmatrix}^{(r)} = \begin{bmatrix} \left(1 + \frac{M_{11}}{m_{11}} \right) & \left(\frac{M_{12}}{m_{11}} \right) & \dots & \dots & \dots & \dots \\ \left(\frac{\omega_1}{\omega_2} \right)^2 \left(\frac{M_{21}}{m_{22}} \right) & \left(\frac{\omega_1}{\omega_2} \right)^2 \left(1 + \frac{M_{22}}{m_{22}} \right) & \dots & \dots & \dots & \dots \\ \dots & \dots & \dots & \dots & \dots & \dots \\ \dots & \dots & \dots & \dots & \dots & \dots \\ \dots & \dots & \dots & \dots & \dots & \dots \\ \dots & \dots & \dots & \dots & \dots & \left(\frac{\omega_1}{\omega_n} \right)^2 \left(1 + \frac{M_{nn}}{m_{nn}} \right) \end{bmatrix} \begin{Bmatrix} A_1 \\ A_2 \\ \vdots \\ A_i \\ \vdots \\ A_n \end{Bmatrix}^{(r)} \quad (39)$$

Equation (39) represents an eigenvalue problem with eigenvalues $\frac{1}{\lambda_r} = \left(\frac{\omega_1^2}{\omega_r^2} \right)$ and

eigenvectors $\begin{Bmatrix} A_1 \\ A_2 \\ \vdots \\ A_n \end{Bmatrix}^{(r)}$. The eigenvalues and eigenvectors for the matrix can be

computed with the use of the Miters eigenvalue subroutine supplied in the program package.

B. Simply Supported Cylindrical Shell with Mass Attachment

The theory developed above is used to determine natural frequencies and mode shapes for a simply supported cylindrical shell with mass attachments. The generalized mass expression m_{ii} for the shell is given by the first term of Equation (21):

$$m_{ii} = \pi a^2 \rho h \int_0^{(\ell/a)} (u_i^2 + v_i^2 + w_i^2) dx \quad (40)$$

where $x = \frac{s}{a}$ the normalized axial coordinate. Substitution of the expressions u_i , v_i , w_i from Equations (31) into the above equations gives:

$$m_{ii} = \pi a^2 \rho h \int_0^{(\ell/a)} \left[A_i^2 \cos^2(k_i \pi \frac{a}{\ell} x) + (B_i^2 + C_i^2) \sin^2(k_i \pi \frac{a}{\ell} x) \right] dx$$

therefore,

$$m_{ii} = \pi a^2 \rho h \frac{\ell}{2a} (A_i^2 + B_i^2 + C_i^2) \quad (41)$$

In Equation (41), the constants A_i and B_i are expressed in terms of C_i through Equations (32a, b). Furthermore, it is assumed that $C_i = 1$, so that the maximum w_i displacement becomes unity. The generalized mass expression M_{ij} , given by Equation (34b), becomes:

$$M_{ij} = \sum_L M_L \left\{ \left[A_i A_j \cos(k_i \pi \frac{a}{\ell} X_L) \cos(k_j \pi \frac{a}{\ell} X_L) + C_i C_j \sin(k_i \pi \frac{a}{\ell} X_L) \sin(k_j \pi \frac{a}{\ell} X_L) \right] \cdot \cos n_i \theta_L \cos n_j \theta_L + B_i B_j \sin(k_i \pi \frac{a}{\ell} X_L) \sin(k_j \pi \frac{a}{\ell} X_L) \sin n_i \theta_L \sin n_j \theta_L \right\} \quad (42)$$

Using Equations (41), (42), and the natural frequencies ω_i determined from Equation (28) above, the $(n \times n)$ matrix Equation (39) is solved in terms of the eigenvalues and eigenvectors.

The displacement modes for the shell are then determined by the use of equations:

$$\begin{Bmatrix} u(x, \theta) \\ v(x, \theta) \\ w(x, \theta) \end{Bmatrix} = \sum_{i=1}^n A_i^{(r)} \begin{Bmatrix} A_i \cos(n_i \theta) \cos(k_i \pi \frac{a}{\ell} x) \\ B_i \sin(n_i \theta) \sin(k_i \pi \frac{a}{\ell} x) \\ C_i \cos(n_i \theta) \sin(k_i \pi \frac{a}{\ell} x) \end{Bmatrix} \quad (43)$$

C. Computer Program for Cylindrical Shell with Mass Attachments

A computer program is developed, based on the above theory, to determine the natural frequencies and mode shapes of a simply supported cylindrical shell with mass attachments. This program can be generalized to handle rotationally symmetric built-up shells with attached masses. The analytical expressions used in the program are given by Equations (32a, b), (39), (41), (42) and (43). The Miters subroutine is used to determine the eigenvalues and eigenvectors of the matrix in Equation (39).

The input data parameters for the program are:

A (a)	Radius of cylinder (in)
CL (ℓ)	Length of cylinder (in)
H (h)	Thickness of the skin (in)
Rho (ρ)	Mass density of the skin material ($\text{lb-sec}^2/\text{in}^4$)
XL (X_L)	Meridional distance from the left end to the location where mass is attached (in)
THL (θ)	Cylindrical angle (rad)
THI	Initial angular location where response is sought (rad)
AMASS (M_L)	Attached mass ($\text{lb-sec}^2/\text{in}$)
INC	Number of increment to determine stations in either axial or circumferential direction
NMODE	The least number of modes, with which convergence will be secured
IMODE	Starting number of mode with which behavior of convergence can be observed ($IMOD < NMODE$)
IINC	Number of modes to be added to IMOD in convergence study. The modes are added in successive batches until NMODE is reached.

The output data consists of:

ω_r - r^{th} natural frequency of mass attached shell (rad/sec)

$$\left\{ \begin{array}{l} A_1 \\ A_2 \\ \cdot \\ \cdot \\ A_n \end{array} \right\}^{(r)} - r^{\text{th}} \text{ mode shape vector}$$

$$\begin{Bmatrix} u \\ v \\ w \end{Bmatrix} - \text{displacements at various locations of the shell}$$

The simply supported cylindrical shell used previously in the unloaded configuration is now investigated with additional mass attachment. The mass M_L and location of the mass are:

$$M_L = 0.259 \times 10^{-2} \text{ lb sec}^2/\text{in.}, \text{ which is the mass corresponding to a 1 lb weight.}$$

$$X_L = \left(\frac{l}{2a}\right)$$

$$\theta_L = 0$$

The two lowest natural frequencies and the corresponding mode shapes are computed using 30, 40 and 50 modes of the unloaded shell. The computed results show that the frequencies and mode shapes converge for the 40 mode solution. Furthermore, it was found that for the 10 mode solution, frequencies converge to within 3 percent of the 40 mode solution. However, the mode shapes from the 10 mode solution did not show good agreement with the mode shape from 40 mode solution. This is the usual result of the energy approach, namely the frequencies converge to the correct values faster than the mode shapes as the number of modes used for computation is increased.

Using 40 modes, the computed frequencies and corresponding mode shapes for the lowest two modes of the mass attached shell are shown in Figures 12, 13, and 14. Figure 12 shows the displacement w along the axial coordinate of the shell for modes 1 and 2. Figure 13 and 14 show the displacement w along the circumferential coordinate of the shell. Similar results have been obtained experimentally (Reference 4) for a cylindrical shell with somewhat different geometric parameters. In general, the mode shapes obtained in the analysis are very similar to the above quoted experimental results.

D. Stiffened Shell Structures with Mass Attachments

The computer program described above for the mass-attached cylindrical shells can be readily generalized to determine the natural frequencies and mode shapes of rotationally symmetric shell structures with mass attachments. The basic theory and analysis are identical for both type shells. A brief description of the method for writing this computer program is now given.

The program is designed to compute the eigenvalues and eigenvectors of the $(n \times n)$ matrix in Equation (39). The expressions for m_{ii} and M_{ij} are given by Equations (21) and (34b), respectively. In these equations, the values of $u_i(s)$, $v_i(s)$ and $w_i(s)$ must be known

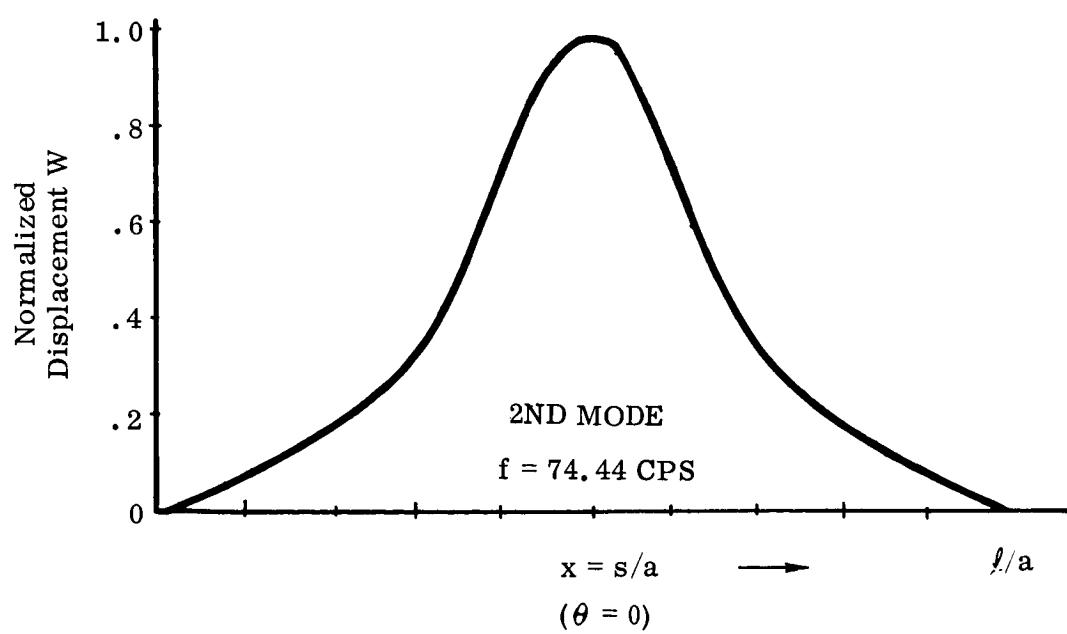
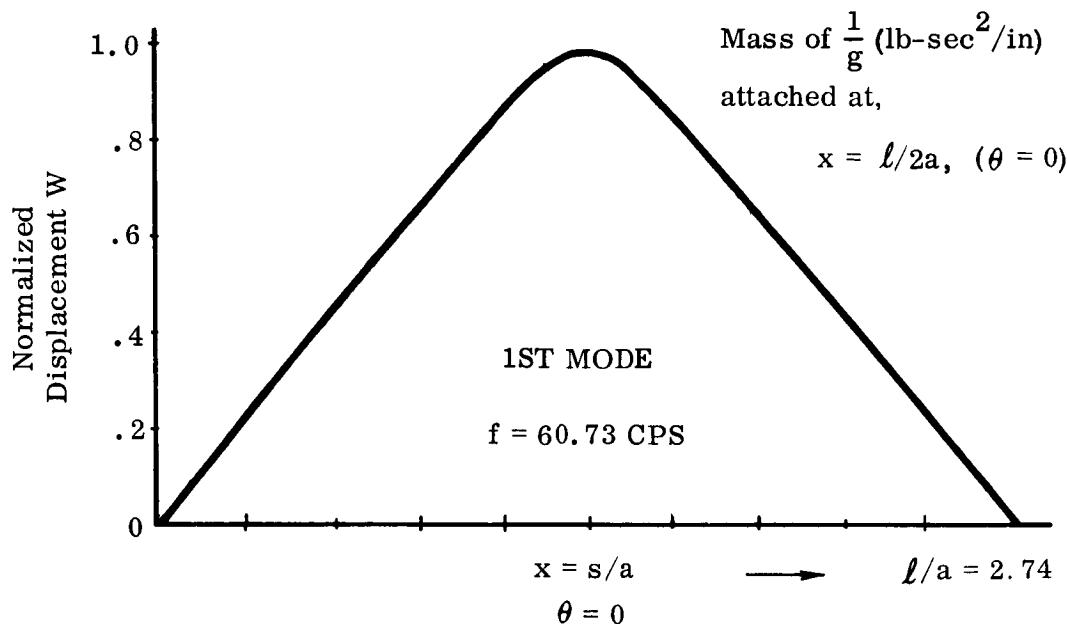


FIGURE 12. AXIAL DEFLECTION

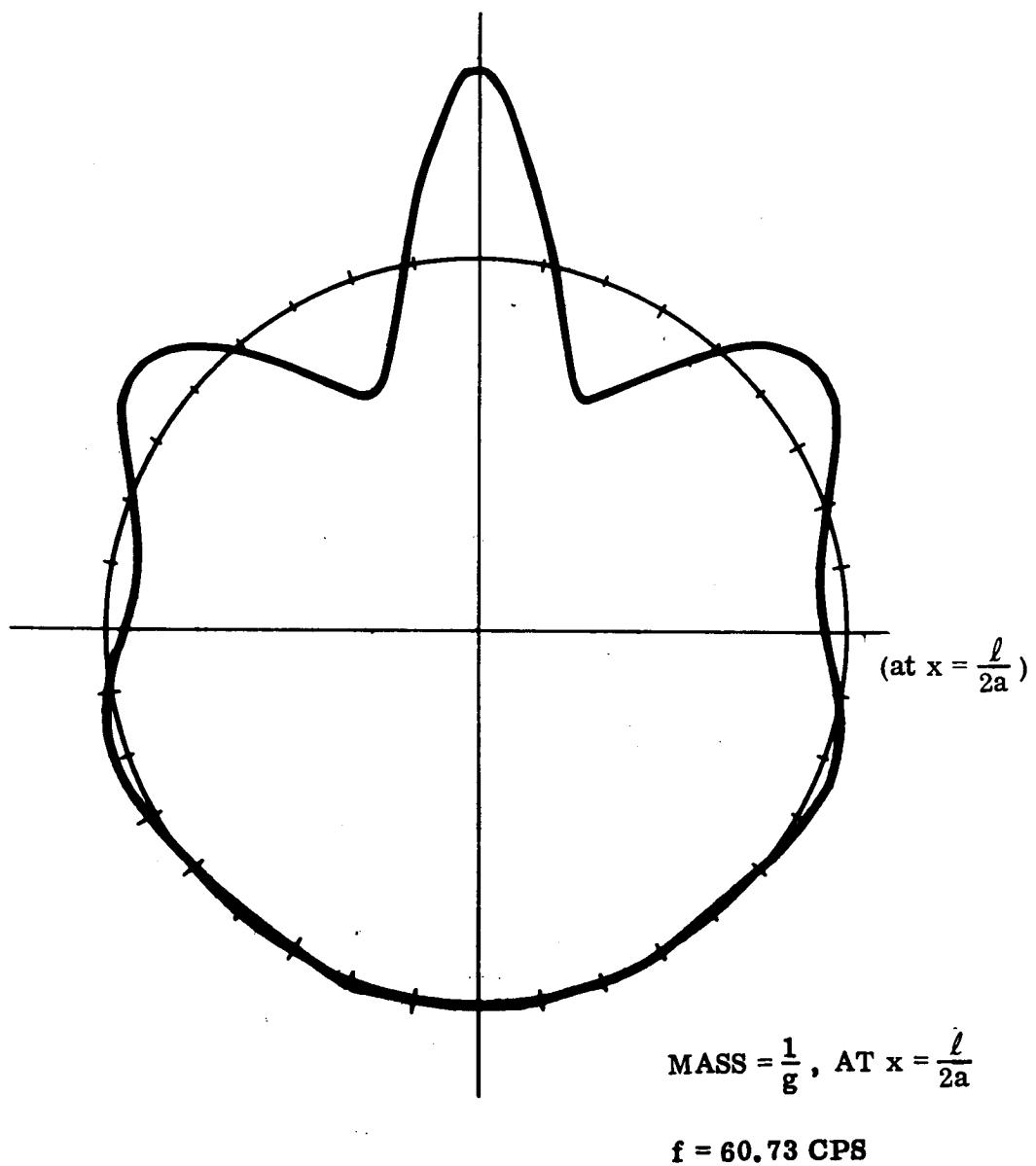


FIGURE 13. CIRCUMFERENTIAL DEFLECTION (THE FIRST MODE)

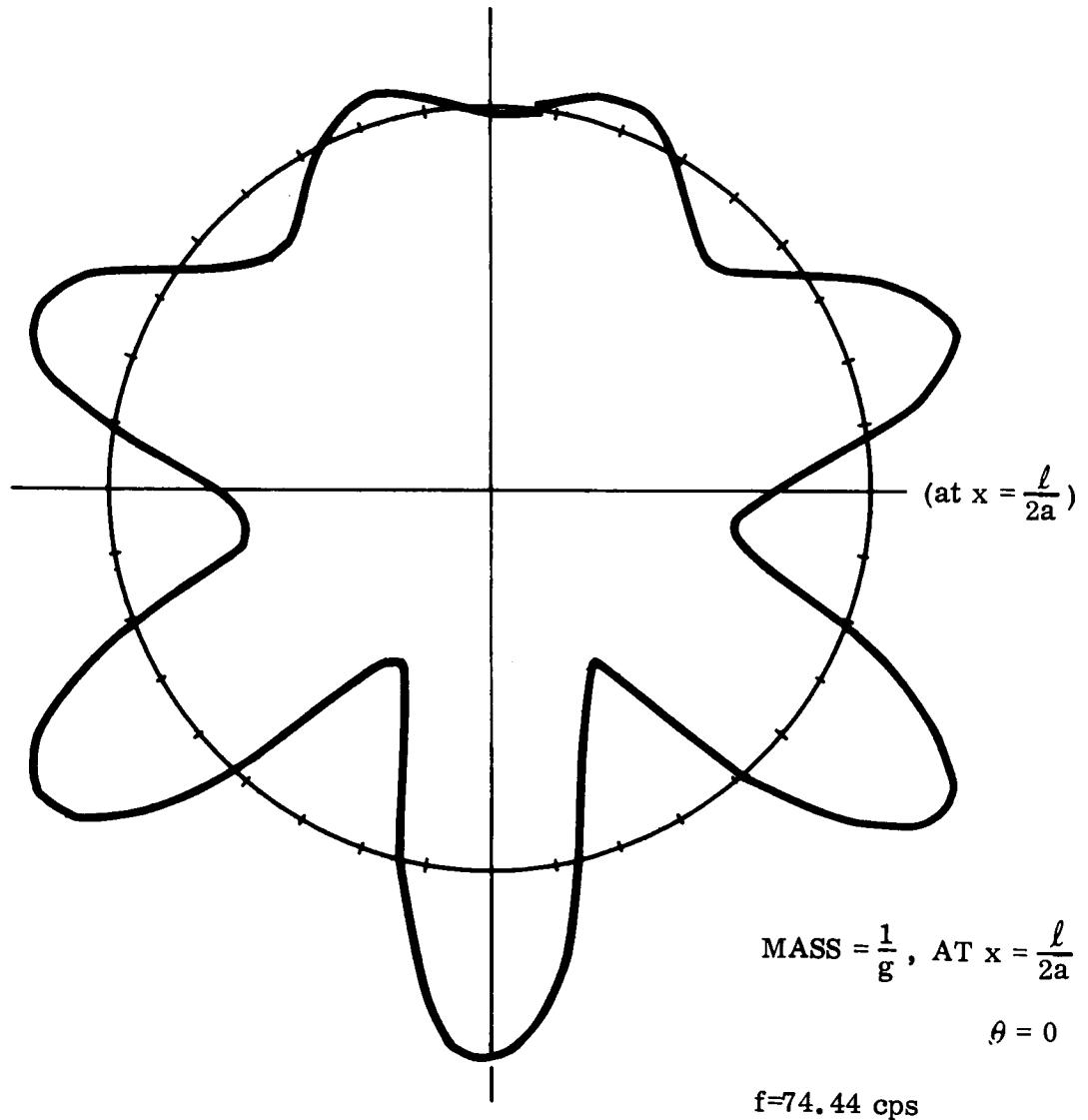


FIGURE 14. CIRCUMFERENTIAL DEFLECTION (THE SECOND MODE)

as a functional relation or in the form of numerical values at sufficient number of locations along the axial coordinate of the built-up shell structure. Hence, the first term of Equation (21) may be integrated either analytical or numerically. The numerical integration subroutine has been used in the transient response computer program described previously. The same subroutine is used in the present program.

The input data to this computer program consist of the geometric parameters for the built-up shell and sufficient number of natural frequencies with corresponding mode shapes to obtain convergent eigenvalues for the $(n \times n)$ matrix. The computed values of the eigenvectors are then used to determine the displacement components of the shell. The displacements are computed by using Equation (20) which is rewritten for the rotationally symmetric shell case as

$$\begin{Bmatrix} u(s, \theta) \\ v(s, \theta) \\ w(s, \theta) \end{Bmatrix} = \sum_{i=1}^n A_i^{(r)} \begin{Bmatrix} u_i(s) \cos n_i \theta \\ v_i(s) \sin n_i \theta \\ w_i(s) \cos n_i \theta \end{Bmatrix}$$

where superscript (r) represents the r^{th} mode of the mass attached shell. The input data parameters and the notations used in the program are similar to those used for the transient response program discussed above.

SECTION IV

ACOUSTIC LOADING AND BLAST OVERPRESSURE

RANDOM ACOUSTIC TESTING

The Saturn V instrument unit scale model was mounted in a 170 cubic-foot reverberant chamber for broad band random acoustic noise tests. The chamber was lined with 8-inch thick open-cell foam material to provide anechoic conditions. The ends of the specimen were capped, with the lower end bolted securely to the chamber floor. As shown in Figure 15, four accelerometers were mounted to the periphery of the instrument unit 90 degrees apart and one each to the upper and lower shell segments. Three microphones were mounted around the specimen for external sound field measurements. One additional microphone was installed inside the specimen at the geometric center.

Tests were conducted at an overall sound pressure level of 145 dB for two configurations of the instrument unit model. In the first test, no masses simulating instrument packages were installed. In the second test, eight weights were installed on the instrument unit section of the model. Each weight was centered within a 15 degree section simulating instrument package installation.

The recording instrumentation consisted of Endevco Dyna-Monitors, B & K and Altec Microphones and Power Supplies, C.E.C. Recording Oscillograph, and a Sanborn 14-Channel Magnetic Tape Recorder. A block diagram of the essential components of instrumentation used for recording and analysis of the acoustic data are shown in Figure 16.

In the random acoustic tests performed, the real time data were recorded on magnetic tapes. The loop data were processed into power spectral data using standard instrumentation. Specifically, autocorrelation function was generated based on the real time data. The Fourier transformation of the autocorrelation function yielded the power spectral data which were plotted by the X-Y recorder. Typical input and response power spectra for the random acoustic excitation are shown subsequently in this section under the heading "Acoustic Response Analysis of Stiffened Shell Models" together with the analytical response spectrum.

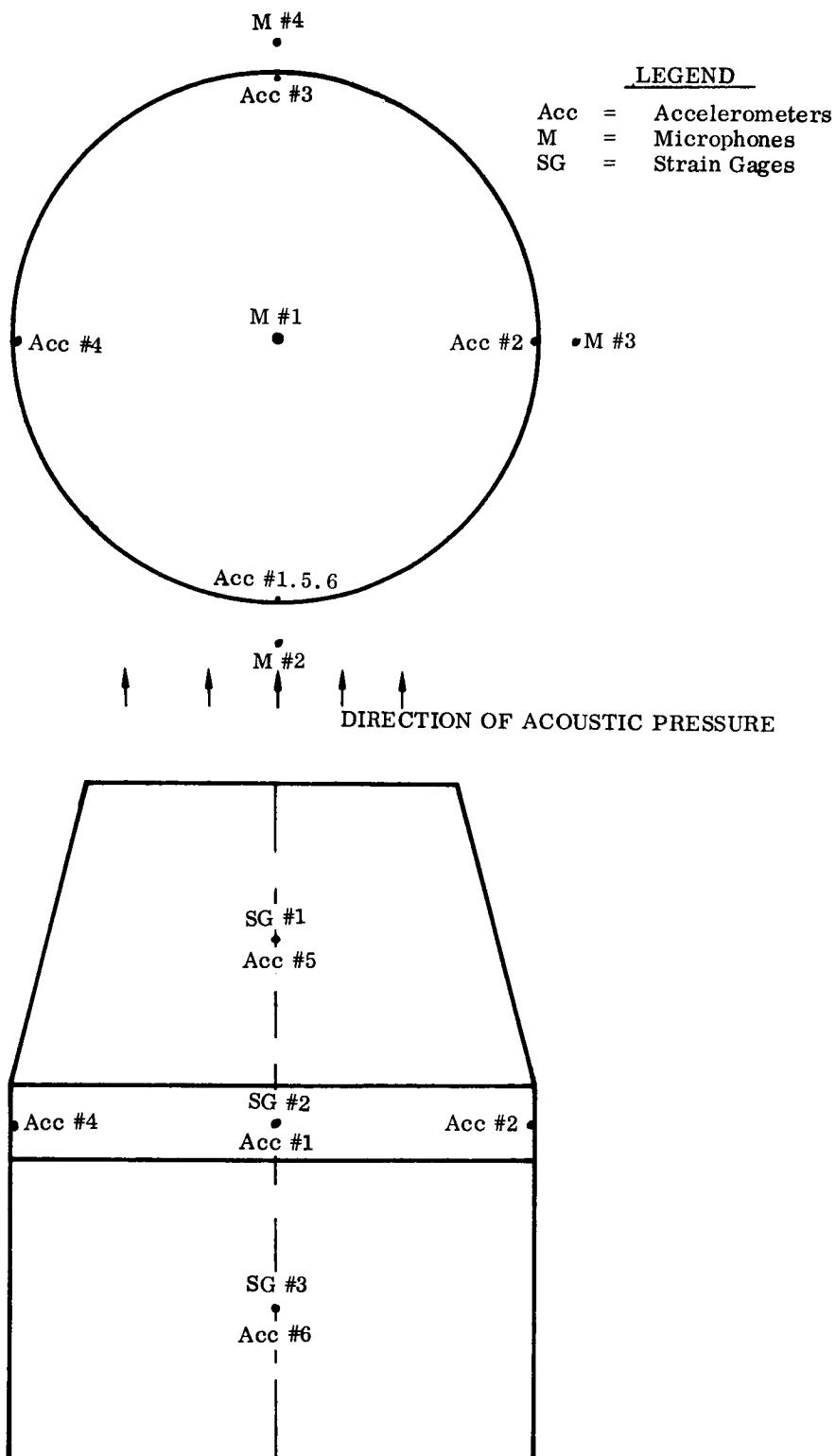


FIGURE 15. ACOUSTIC TEST INSTRUMENTATION ON THE INSTRUMENT UNIT SCALE MODEL

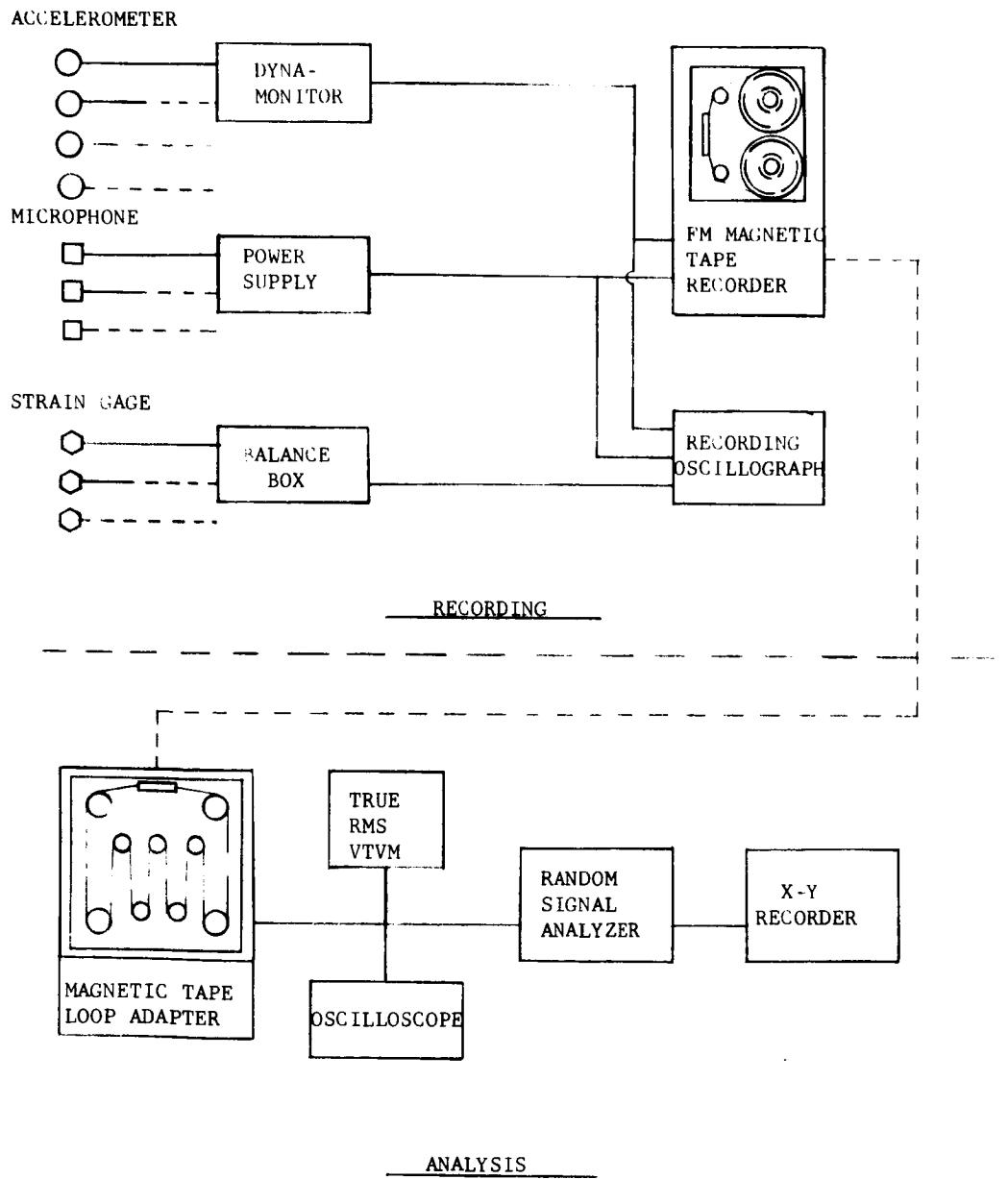


FIGURE 16. BLOCK DIAGRAM, RECORDING AND ANALYSIS EQUIPMENT FOR ACOUSTIC TESTS

BLAST OVERPRESSURE TESTING

The same instrument unit scale model was subjected to a simulated blast overpressure condition in the same configurations as that used for broad band random acoustic testing. The capped ends of the specimen allow atmospheric pressure to exist within the specimen cavity while the external surface is exposed to the over-pressure pulse. The tests were conducted in the reverberant chamber.

The overpressure pulse was obtained by applying a controlled electrical impulse to the Norair MK-V acoustic generator. The generator employs a unique poppet valve design which controls the high pressure air flow at the mouth of the horn. The single electrical impulse produces a high pressure wave in the form of a shock front sawtooth. Peak pressures approaching 0.5 psi were obtained by this method. Six accelerometers and four microphones were monitored and recorded for both specimen configurations tested. The locations of the pickups were the same as that used in the acoustic tests which are shown in Figure 15. In addition to the above mentioned instrumentation, an attempt was made to record the surface bending strain at three points on the specimen surface. It was found during initial tests that the strain levels were too low to be accurately recorded for analysis purposes. Typical real time data are shown in Figures 17 and 18 for the unloaded instrument unit scale model. In Figure 17, the accelerometer data are recorded in six (6) channels. The corresponding strain gage and microphone data are recorded in Figure 18.

ACOUSTIC RESPONSE ANALYSIS OF STIFFENED SHELL MODELS

A (1:6.67) scale model of the instrument unit shell structure was tested in Northrop Norair acoustic chamber. The detail test setup was described in previous subsections. The real time data were recorded on magnetic tape loops and processed into power spectra using standard instrumentation. Typical input and response power spectra generated in this manner are plotted in Figures 19 and 20. An analytical technique described in References 5 and 6 was used to generate the response data using the input spectrum and the previously obtained natural frequency modes of the shell scale model. The data obtained analytically are over-plotted in Figure 20. The following is a description of the analysis.

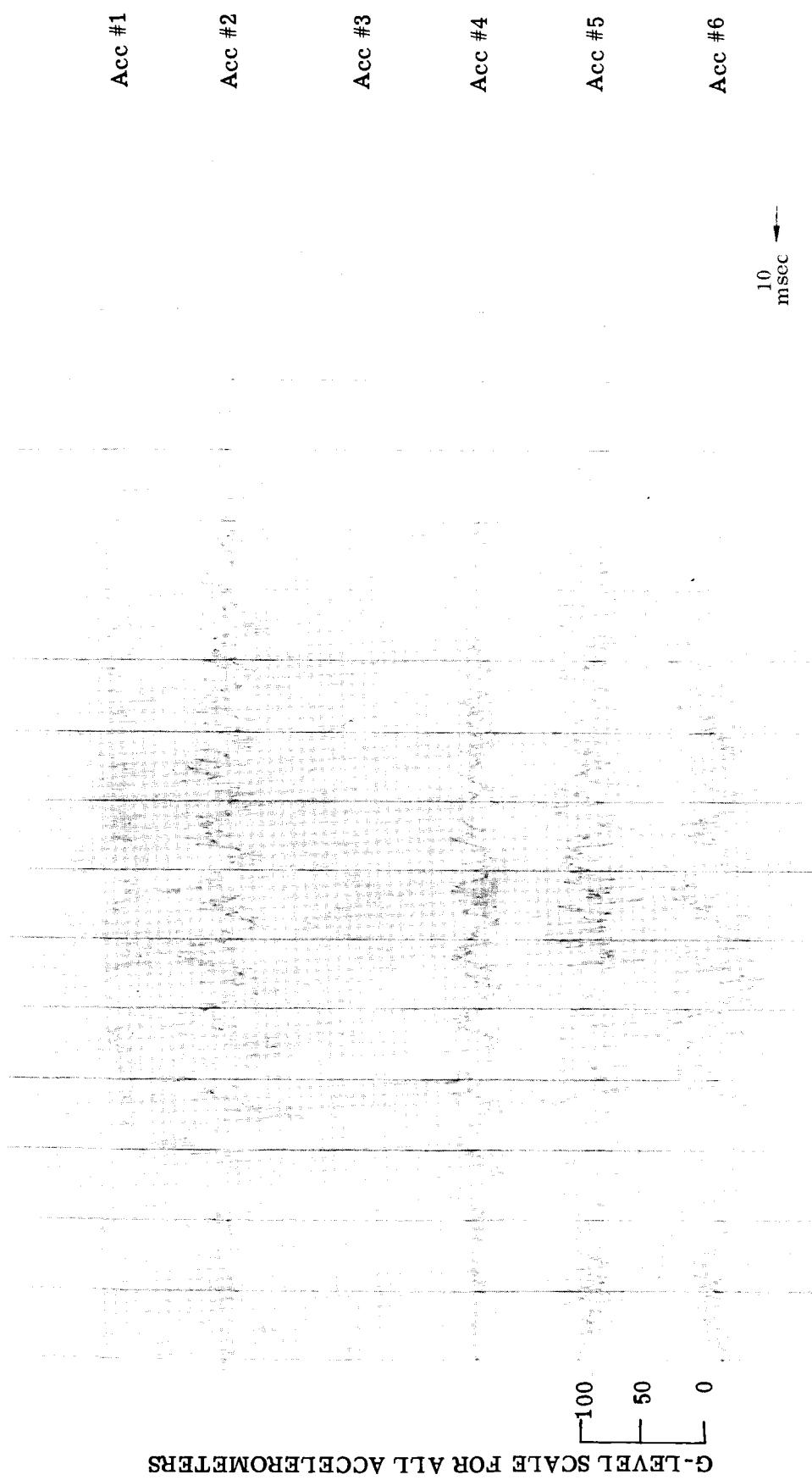


FIGURE 17. ACCELEROMETER DATA OF THE INSTRUMENT UNIT SCALE MODEL

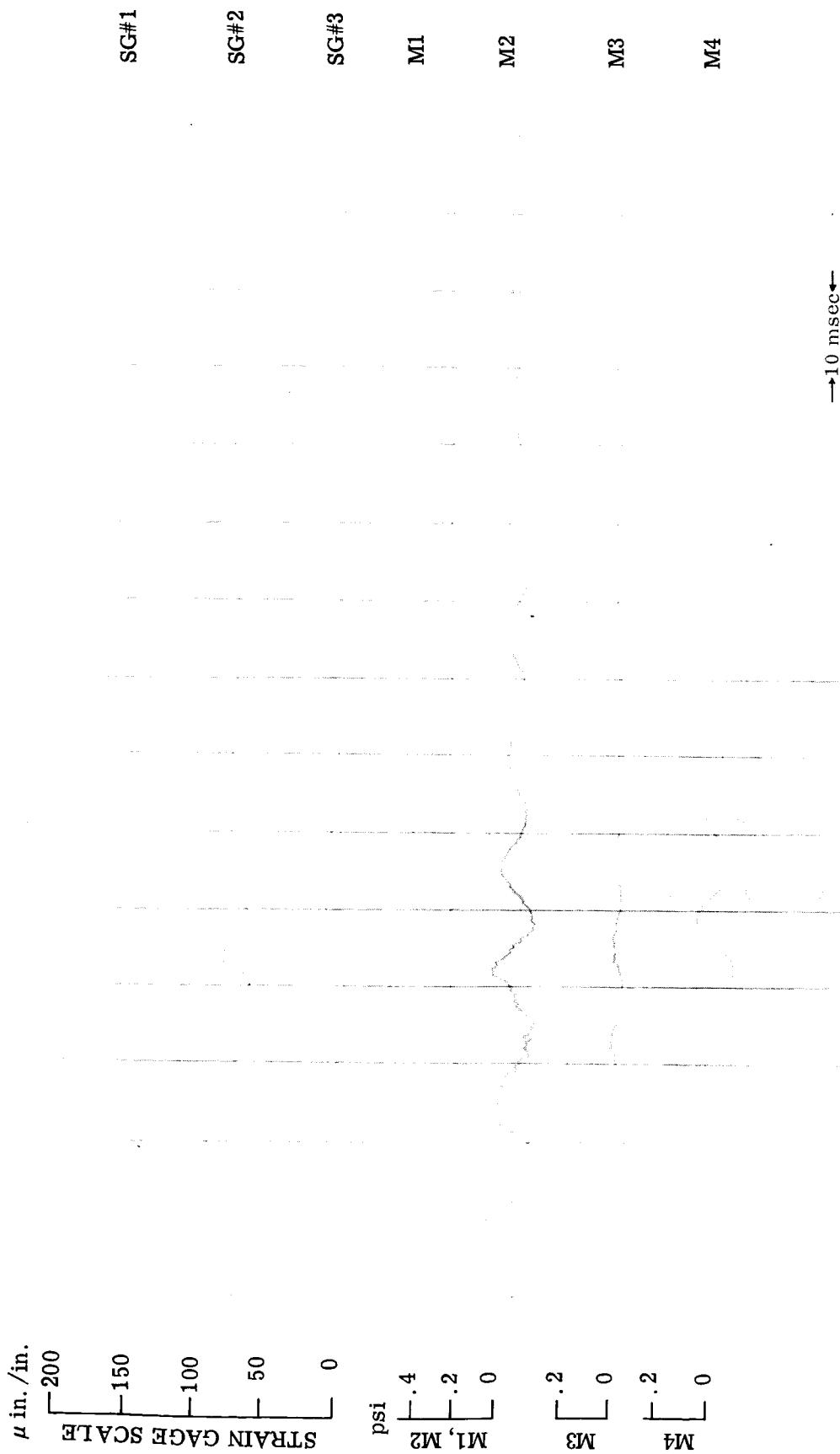


FIGURE 18. STRAIN GAGE AND MICROPHONE DATA OF THE INSTRUMENT UNIT SCALE MODEL

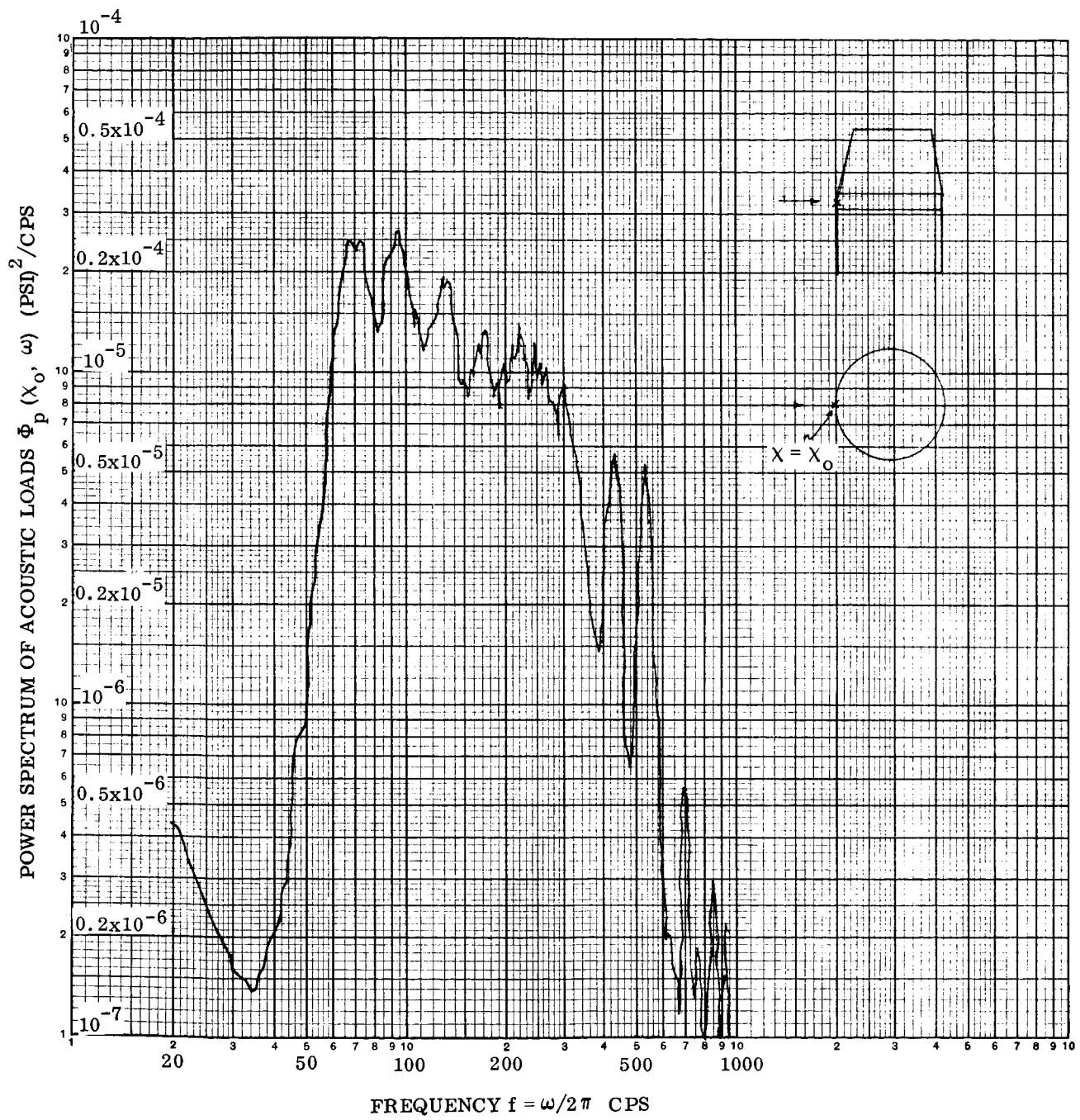


FIGURE 19. TYPICAL INPUT POWER SPECTRUM OF ACOUSTIC LOADS APPLIED TO INSTRUMENT UNIT SCALE MODEL

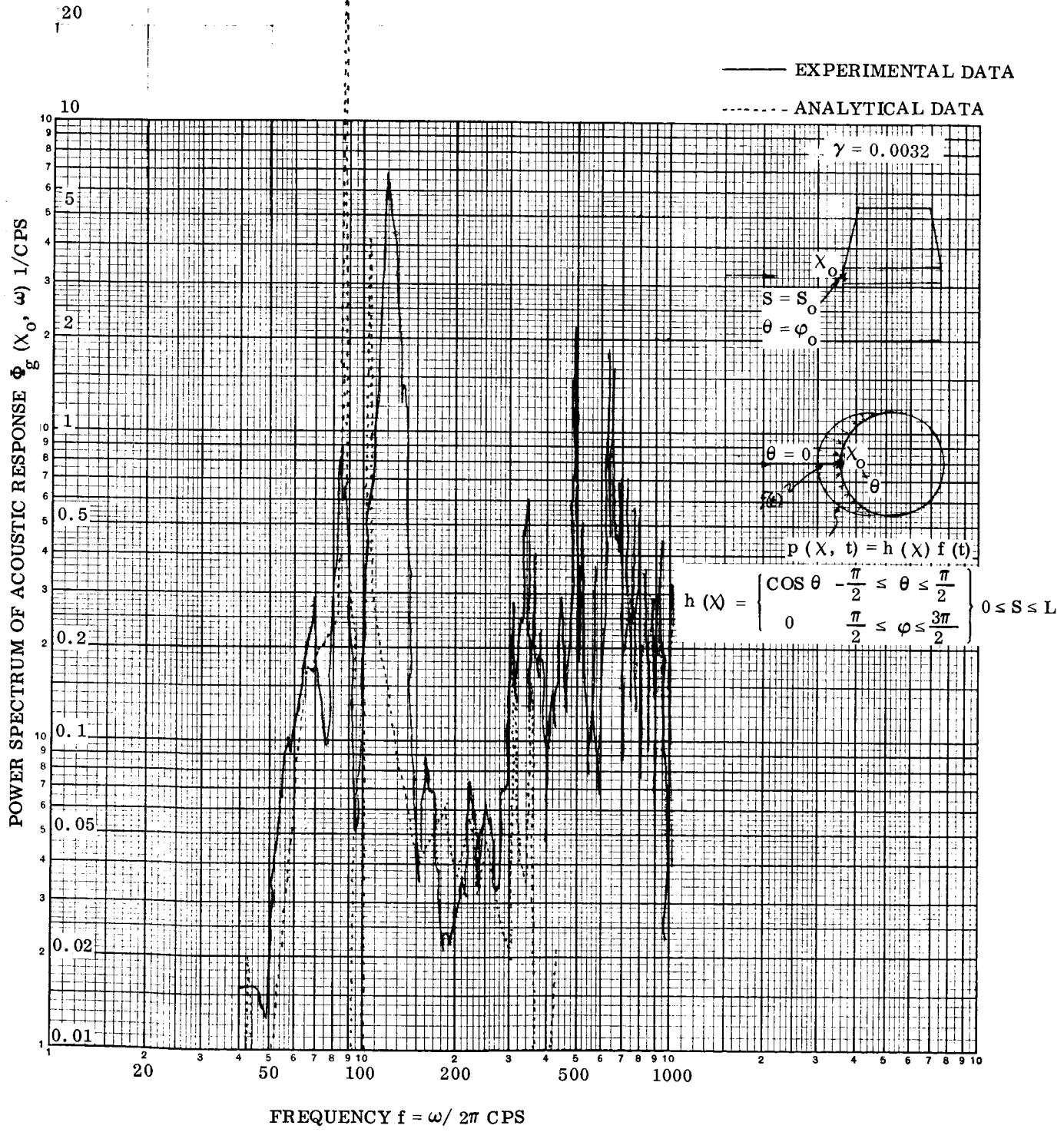


FIGURE 20. TYPICAL ACOUSTIC RESPONSE POWER SPECTRUM OF INSTRUMENT UNIT SCALE MODEL

Let the Fourier transform of the acoustic pressure be expressed as $F_p(x, \omega)$, it may be shown that the expression for the power spectrum of the radial displacement is:

$$\phi_w(x, \omega) = \sum_k \sum_j \frac{w_k(x)}{M_k[(\omega_k^2 - \omega^2) + iC_k \omega]} \frac{w_j(x)}{M_j[(\omega_j^2 - \omega^2) - iC_j \omega]} \times$$

$$\iint_{\bar{A} \bar{A}} \lim_{T \rightarrow \infty} \sqrt{\frac{\pi}{2T^2}} F_p^T(x_1, \omega) \overline{F_p^T(x_2, \omega)} w_k(x_1) w_j(x_2) d\bar{A}_1 d\bar{A}_2 \quad (1)$$

In the above formulation, only the normal pressure is considered. The following conventions are used:

$w_k(x)$ = the k^{th} normal mode of the shell

ω_k = the k^{th} eigen-frequency (rad/sec)

M_k = the generalized mass corresponding to the k^{th} mode

$$= \int_{\bar{A}} m(w_k^2 + u_k^2 + v_k^2) d\bar{A} + \sum_r \rho_r a_r \int_0^{2\pi} \left\{ A \left[(u_k + \beta_k e_n)^2 \right. \right.$$

$$\left. \left. + v_k^2 + (w_k - \beta_k e_\phi)^2 \right] + (I_x + I_y) \beta_k^2 \right\}_r d\theta \quad (2)$$

C_k = the damping coefficient of the k^{th} mode

\bar{A} = the total surface area of the shell

T = time

r = index for rings in stiffened shell

$i = \sqrt{-1}$

m = mass of shell per unit area

w, u, v = displacement components

β = angle of rotation

ρ_r = density of the ring stiffener material

a_r = radius of the ring stiffener

A = cross section of the ring stiffener

I_x, I_y = moment of inertia of the ring stiffener about its principal axes

e_n, e_ϕ = distance between c.g. of the ring to the point of attachment in n and ϕ directions, respectively, (see Reference 1, p. 23)

The power spectral density as obtained from an external pressure pickup is assumed to be typical of the acoustic input spectra, (Figure 19). The following assumptions are employed to simplify the computations.

$$A) \quad p(x, t) = h(x) f(t) \quad (3)$$

i. e.,

$$F_p^T(x, \omega) = h(x) F^T(\omega) \quad (4)$$

where

$$F^T(\omega) = \frac{1}{\sqrt{2\pi}} \int_{-T}^{T} f(t) e^{-i\omega t} dt \quad (5)$$

$$B) \quad C_k = 2\gamma\omega_k \quad (6)$$

where

$$\gamma = \frac{c}{c_{cr}} \approx \frac{\delta}{2\pi}$$

c = damping coefficient

= the magnitude of the viscous resistance at unit velocity

c_{cr} = critical damping coefficient

δ = the logarithmic decrement of the amplitude

Substituting Equations (3) through (6) into (1) and using the definition of the power spectrum, the following is obtained:

$$\Phi_p(x, \omega) = \lim_{T \rightarrow \infty} \sqrt{\frac{\pi}{2T^2}} F_p^T(x, \omega) \overline{F_p^T(x, \omega)} \quad (7)$$

The power spectral density of the shell acoustic response may be obtained as follows:

$$\begin{aligned} \Phi_w(x, \omega) &= \Phi_p(\omega) \sum_k \sum_j \frac{w_k(x)}{M_k [(\omega_k^2 - \omega^2) + 2i\gamma\omega\omega_k]} \cdot \frac{w_j(x)}{M_j [(\omega_j^2 - \omega^2) - 2i\gamma\omega\omega_j]} \times \\ &\quad \int_A w_k(x_1) h(x_1) \int_A w_j(x_2) h(x_2) d\bar{A}_2 d\bar{A}_1 \\ &= \Phi_p(\omega) \sum_k \sum_j \frac{B_k w_k(x)}{M_k [(\omega_k^2 - \omega^2)^2 + 4\gamma^2 \omega^2 \omega_k^2]} \cdot \frac{B_j w_j(x)}{M_j [(\omega_j^2 - \omega^2)^2 + 4\gamma^2 \omega^2 \omega_j^2]} \times \\ &\quad [(\omega_k^2 - \omega^2)(\omega_j^2 - \omega^2) + 4\gamma^2 \omega^2 \omega_j \omega_k] \end{aligned} \quad (8)$$

where

$$\Phi_p(\omega) = \frac{\Phi_p(x_0, \omega)}{h^2(x_0)} \quad (9)$$

$$B_k = \int_A w_k(x) h(x) d\bar{A} \quad (10)$$

In Equation (10), B_k is the generalized force corresponding to mode $w_k(x)$. For the unloaded instrument unit scale model, $w_k(x) = \bar{w}_k(s) \cos n_k \theta$, the acoustic pressure distribution is assumed as the following:

$$h(x) = \begin{cases} \cos \theta, & -\frac{\pi}{2} \leq \theta \leq \frac{\pi}{2} \\ 0, & \frac{\pi}{2} \leq \theta \leq \frac{3\pi}{2} \end{cases} \quad 0 \leq s \leq L \quad (11)$$

so that

$$\Phi_p(\omega) = \Phi_p(x, \omega) \Big|_{\theta=0, s=s_0}$$

$$B_k \begin{cases} = -\frac{2}{n_k^2 - 1} \cos \frac{n_k \pi}{2} \int_0^L R(s) \bar{w}_k(s) ds, & n_k \neq 1 \\ = (\pi/2) \int_0^L R(s) \bar{w}_k(s) ds, & n_k = 1 \end{cases} \quad (12)$$

In Equations (12), $R(s)$ is the geometrical radius of the shell which is a function of the meridian locations. The solid line in Figure 20 shows the typical experimental power spectrum data of the shell response. The ordinate used in the plot is the spectral density of the acceleration level measured in g's. In other words, it represents the spectral density of $(w\omega^2/\bar{g})$ and has a dimension of (cps^{-1}) since the frequency coordinate is in cps. The dotted line indicates the computed value (Φ_g) corresponding to the shell natural frequencies with $\gamma = 0.0032$. The formula used to compute Φ_g is an alternative version of Equation (8) which is explained below:

$$\Phi_g(x, \omega) = \Phi_w(x, \omega) \frac{\omega^4}{\bar{g}^2}$$

$$= \Phi_p(\omega) \sum_k \sum_j \frac{B_k w_k(x)}{M_k \bar{g} \left[\left(\frac{\omega_k}{\omega} \right)^2 - 1 \right]^2 + 4\gamma^2 \left(\frac{\omega_k}{\omega} \right)^2} \frac{B_j w_j(x)}{M_j \bar{g} \left[\left(\frac{\omega_j}{\omega} \right)^2 - 1 \right]^2 + 4\gamma^2 \left(\frac{\omega_j}{\omega} \right)^2} \times$$

$$\left\{ \left[\left(\frac{\omega_k}{\omega} \right)^2 - 1 \right] \left[\left(\frac{\omega_j}{\omega} \right)^2 - 1 \right] + 4\gamma^2 \left(\frac{\omega_k}{\omega} \right) \left(\frac{\omega_j}{\omega} \right) \right\} \quad (13)$$

where subscript g indicates the response in terms of acceleration level and \bar{g} is the gravitational acceleration.

To mechanize the analysis described above, the following step-by-step procedures may be used.

- Run a stiffened shell program or other similar programs to obtain all the natural modes with frequencies lower than the upper limit of the response frequency range. The program generates the natural frequency ω_k and mode shape w_k, u_k, v_k, β_k , etc. for each mode. Figure 21 shows the effect of the shell response at high frequency region which is influenced by a low frequency mode using the single mode approach. The plot stresses the importance of compiling all the natural frequency modes below the cut-off frequency in acoustic response investigation.
- Compute the generalized mass M_k for each mode using Equation (2).
- Choose a typical input power spectrum of acoustic loads $\Phi_p(x_0, \omega)$ at a certain location $x = x_0$.
- Determine the spacewise pressure distribution function $h(x)$.
- Compute the generalized force B_k for each mode using Equation (10).
- Determine a number of frequencies ω on the input spectrum at which the values of $\Phi_p(x_0, \omega)$ are measured or tabulated. The following frequencies are to be included:
 - A frequency corresponding to the peak of the input spectrum. Also two or three additional frequencies in its neighborhood which completely define the peak pattern.

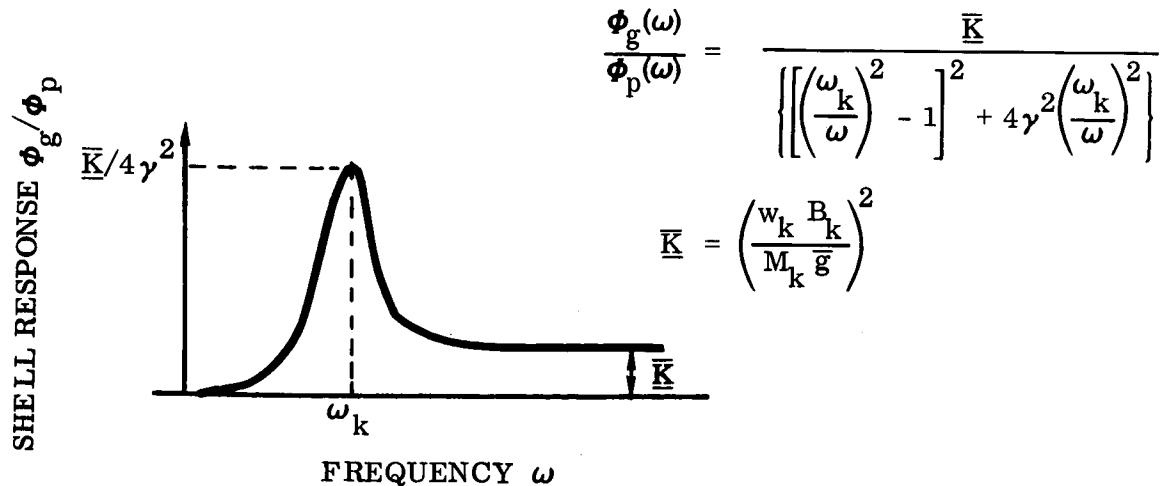


FIGURE 21. TRANSFER FUNCTION CORRESPONDING TO A SINGLE MODE

- b. A shell natural frequency. Also two or three additional frequencies in the neighborhood of the natural frequency.

After the above frequencies have been chosen, additional frequency values are selected so that the entire frequency range is properly covered.

7. Measure $\Phi_p(x_0, \omega)$ at each response frequency ω determined in the previous step.
8. Compute and tabulate $\frac{\Phi_p(x_0, \omega)}{h^2(x_0)} = \phi_p(\omega)$ as a function of ω .
9. Determine the damping coefficient $c = \gamma c_{cr}$ based on experimental data. In case the damping data is not available, the following procedure may be used. The procedure is based on a best fit of the peak values of the analytical and experimental response power spectra at certain mode frequencies. Use the following formula to estimate the peak value of the power spectral density of the shell acoustic response at the natural frequency of the k^{th} mode ($\omega = \omega_k$):

$$\phi_g(x, \omega_k) \approx \phi_p(\omega_k) \left[\frac{B_k w_k(x)}{M_k g} \right]^2 \frac{1}{4\gamma^2}$$

In the above equation, γ is adjusted so that a proper fit may be obtained between the test and analytical data. Several natural frequency modes are used to reach a reasonable mean value of the damping coefficient.

10. Run the acoustic response program to obtain the shell response $\Phi_g(x_1, \omega)$ at a specific location $x = x_1$ with the following data as input:
 - a. The damping constant γ
 - b. The natural frequencies of the modes ω_k , $k = 1, 2 \dots N$
 - c. The generalized forces of the modes B_k , $k = 1, 2 \dots N$
 - d. The generalized masses of the modes M_k , $k = 1, 2 \dots N$
 - e. The response frequencies $\omega(I)$, $I = 1, 2 \dots L$
 - f. The input power spectrum $\Phi_p[\omega(I)]$, $I = 1, 2 \dots L$

g. The amplitudes of the normal mode shapes at $x = x_1$

$$w_k(x_1), k = 1, 2 \dots N$$

where N = the total number of input modes

L = the total number of stations selected in the frequency domain

The complete program and the work instructions to carry out the above analytical procedures are given in Appendix C.

A sample run was performed to predict the acoustic response of the instrument unit scale model in the frequency range between 40 and 350 cps. The spacewise pressure distribution was assumed to be prescribed by Equation (11). Nine natural modes were employed as part of the input. The model data were obtained through the general stiffened shell program. Their validity was confirmed by vibration tests.

Figure 22 shows the driving point impedance plot of the unloaded instrument unit scale model. The computed eigen-frequencies of the input modes used in acoustic analysis are marked in the figure. The low impedance points in between the arrowheads reflect the modes with odd circumferential wave numbers ($n_k = 3, 5, 7 \dots$). It may be seen from Equation (12) that

$$B_k = 0 \text{ for } n_k = 3, 5, 7 \dots$$

Therefore, those modes do not contribute to the response spectrum and are not included in the acoustic response computation. Table 1 shows the detail modal data of the input modes used. The data (w_k) have been normalized and are non-dimensional. It may be seen that $(B_k w_k) / (M_k \bar{g})$ remains to be non-dimensional while \bar{g} has a unit of (inch/second²). The analytical spectrum computed in this manner is plotted as a broken line in Figure 20.

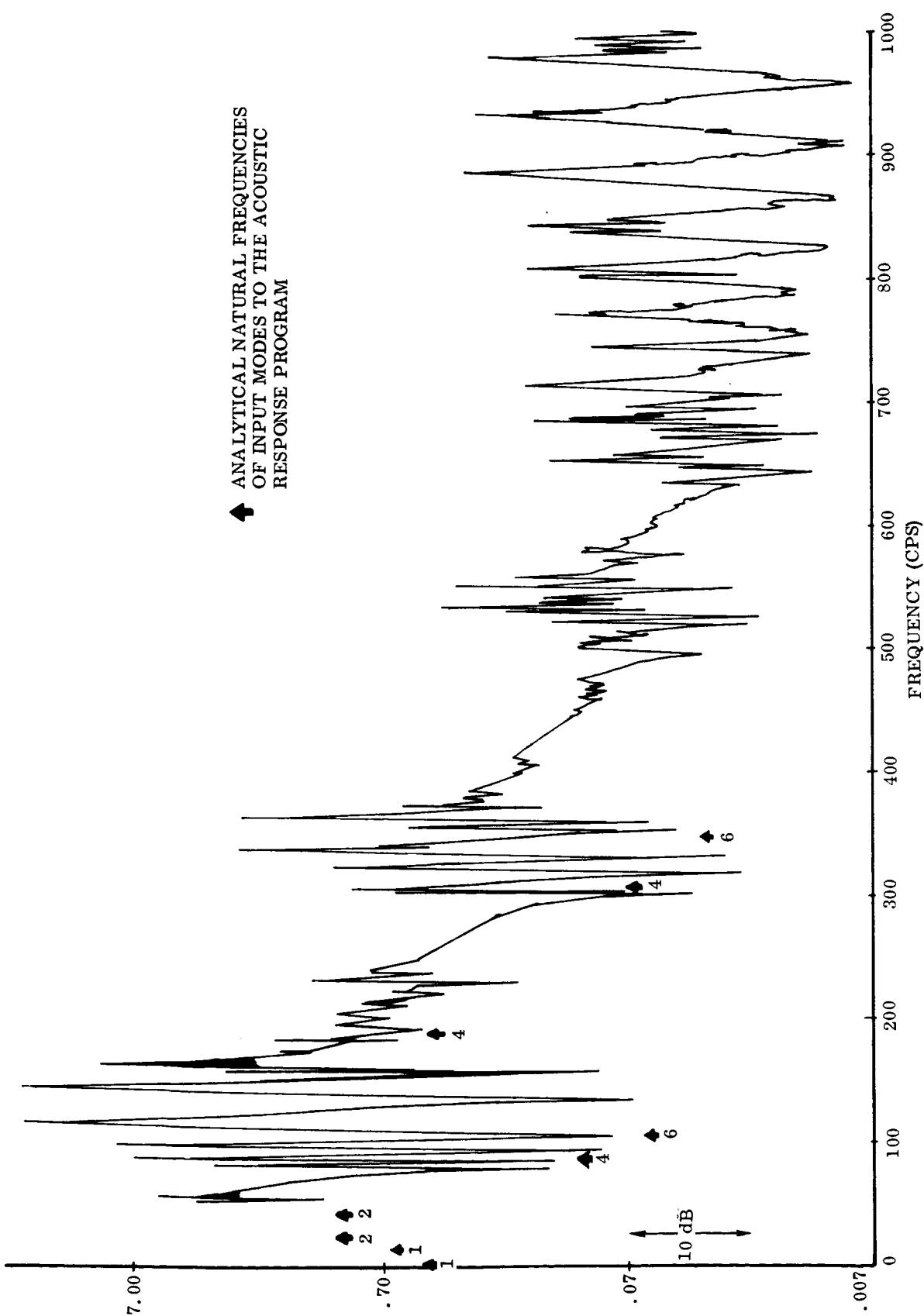


FIGURE 22. DRIVING POINT IMPEDANCE OF UNLOADED INSTRUMENT UNIT SCALE MODEL

TABLE 1. MODAL INPUT DATA USED IN THE INSTRUMENT
UNIT ACOUSTIC RESPONSE PROGRAM

HARMONIC NO.	NATURAL FREQUENCY	AMPLITUDE OF MODAL SHAPE AT $\theta=0$, $S=S_0$	GENERALIZED MASS	INTEGRATION CONSTANT
n_j	$\omega_j / 2\pi$	w_k	M_k (lb. in. sec ²)	B_k (lb. in.)
1	0.11	0.988	0.0908	$926 \times \left(\frac{\pi}{2}\right)$
1	15.0	-0.0235	0.06	$74.5 \times \left(\frac{\pi}{2}\right)$
2	22.8	0.786	0.0377	$732.1 \times \left(\frac{2}{3}\right)$
2	42.6	-0.209	0.0243	$-107.3 \times \left(\frac{2}{3}\right)$
4	87.25	0.99	0.029	$-745 \times \left(\frac{2}{15}\right)$
6	105.0	0.984	0.0263	$628 \times \left(\frac{2}{35}\right)$
4	188.8	0.21	0.01985	$-69.5 \times \left(\frac{2}{15}\right)$
4	306.6	-0.408	0.0192	$-131.7 \times \left(\frac{2}{15}\right)$
6	348.5	-0.753	0.0218	$257 \times \left(\frac{2}{35}\right)$

SECTION V

SHELL SCALE MODEL DESIGN PROCEDURE

To design shell scale models for dynamic investigation, it is important to establish specific similitude relations. The similitude relations are used to define the dimensions, materials and other parameters in model design. They are also used to interpret the model dynamic response data and to predict the corresponding responses in the full scale structure.

During the present investigation, a number of scale models have been designed and tested. In one case, the scale model dynamic response data were compared with the full scale structure data. In general, the procedure has been found satisfactory and is described below.

GENERAL SCALING CONSIDERATIONS

Assume a prototype structure which has a diameter of 260 inches and is made of honeycomb sandwich. The scale model is made of solid aluminum sheet with ring stiffeners. The linear scale ratio is 1:6.67. The basic dimension system of length, force, time is used. In establishing the scaling relations, a prime on a variable is used to indicate the full scale structure (prototype). A non-primed variable is used to indicate the scale model. Corresponding to the basic dimensions (ℓ , F, t), the following nomenclatures are used:

$$\ell' = \lambda \ell$$

$$F' = X F$$

$$t' = \tau t$$

For most other variables, the scaling ratio is represented by σ attached with a subscript indicating the variable under consideration. Thus the scaling factor for the Young's modulus is:

$$\sigma_E = E'/E.$$

To establish proper scaling relations for shell structures, typical shell dynamic equations are established. The procedure used here is similar to the one demonstrated

in the previous reports. Starting with the shell equations originated by Vlasov (Reference 8), the equations in the most general form are:

$$\left\{ \begin{array}{l} \frac{1}{Eh} \Delta \Delta \Phi + \mathcal{D} w = 0 \end{array} \right. \quad (1)$$

$$\left\{ \begin{array}{l} D \Delta \Delta w - \mathcal{D} \Phi = q_n \end{array} \right. \quad (2)$$

In the above equations, h is the shell thickness, D is the shell flexural rigidity, Δ is the Laplace operator in the curvilinear coordinates, \mathcal{D} is another second order differential operator. Φ is an auxiliary stress function whose second derivatives yield the in-plane stresses. Φ has a dimension of (F/l) . w is the normal displacement, q_n is the normal load on the shell. For a cylindrical shell, the cylindrical coordinate system (r, θ, z) is used. The two differential operators may be written as:

$$\Delta = \frac{1}{r^2} \frac{\partial^2}{\partial \theta^2} + \frac{\partial^2}{\partial z^2}$$

$$\mathcal{D} = \frac{1}{r} \left(\frac{1}{r^2} \frac{\partial^2}{\partial \theta^2} + \frac{\partial^2}{\partial z^2} \right)$$

So that Δ has the dimension of $(\frac{1}{l^2})$ and \mathcal{D} has the dimension of $(\frac{1}{l^3})$. The shell flexural rigidity D may be defined below.

For solid shell:

$$D = \frac{Eh^3}{12(1 - \nu^2)} \quad (3)$$

For honeycomb sandwich shell:

$$D = \frac{Eh(h + 2c)^2}{16(1 - \nu^2)} \quad (4)$$

where h is the sandwich skin total thickness and c is the core thickness.

For dynamic case, it is understood that a factor $e^{i\Omega t}$ has been dropped from all terms of (1), (2). The normal load q_n is represented in terms of the inertia forces. Thus, for a solid shell

$$q_n = \rho h \omega^2 w \quad (5)$$

For a honeycomb sandwich shell,

$$q_n = \rho H \omega^2 w \quad (6)$$

where H is the compacted thickness of the shell.

Using the primed variables for the prototype structure and non-primed variables for the scale model, the coupled shell equations (1), (2) for the prototype may be rewritten as follows in terms of the scale model variables:

$$\left\{ \frac{\chi}{\sigma_E \lambda^4} \frac{1}{Eh} \Delta \Delta \Phi + \frac{1}{\lambda^2} \mathcal{D}w = 0 \right. \quad (7)$$

$$\left. \frac{\sigma_D}{\lambda^3} D \Delta \Delta w - \frac{\chi}{\lambda^2} \mathcal{D}\Phi = \sigma_\rho \sigma_Q^2 \sigma_H \lambda \rho h \omega^2 w \right. \quad (8)$$

In order that Equations (7), (8) are equivalent to Equations (1), (2) applied to the scale model, the following scaling relations are required:

$$\chi = \sigma_E \lambda^2 \quad (9)$$

$$\sigma_D = \sigma_E \lambda^3 \quad (10)$$

$$\sigma_Q = \frac{1}{\sqrt{\sigma_H \lambda}} \sqrt{\frac{\sigma_E}{\sigma_\rho}} = \frac{1}{\lambda^2} \sqrt{\frac{\sigma_D}{\sigma_\rho \sigma_H}} \quad (11)$$

In the above derivation, it has been assumed that:

$$\sigma_\Phi = \chi \lambda \quad (12)$$

Since the second derivatives of the auxiliary function Φ yields the membrane stresses, Equation (12) is acceptable as long as the shell thickness is scaled according to (λ). In case this latter condition is not satisfied, then the membrane stress scaling relation is violated. Proper compensation is needed if the membrane stresses are significant. A typical example of discrepancy in membrane stress scaling involves the modeling of a sandwich honeycomb shell using a solid shell. This will be explained later through Equation (10).

Equation (9) defines the scaling relation for the Young's moduli of the prototype and the scale model in terms of the force and length scales. Equation (10) defines the shell flexural rigidity scaling relation which is important in thin shells used in aerospace structures. σ_D is a ratio of the flexural rigidity where proper expression

(3) or (4) is to be used. Equation (11) defines the frequency ratio. The equation is reduced to:

$$\sigma_{\Omega} = \Omega' / \Omega = \frac{1}{\lambda} \sqrt{\frac{\sigma_E}{\sigma_\rho}} \quad (13)$$

for the special case $\sigma_H = \lambda$. For a shell structure where concentrated mass(es) M is attached, the contribution to the inertia term at the right hand side of Equation (72) is of the following type:

$$q_n = \frac{M}{\bar{l}^2} \omega^2 w \quad (14)$$

where \bar{l}^2 is the area of attachment. A scaling relation similar to (11) is then:

$$\sigma_{\Omega} = \sqrt{\lambda} \sqrt{\frac{\sigma_E}{\sigma_M}} = \frac{1}{\lambda} \sqrt{\frac{\sigma_D}{\sigma_M}} \quad (15)$$

which may be rewritten as:

$$\sigma_M = \frac{\sigma_D}{\lambda^2 \sigma_{\Omega}^2} = \lambda^2 \sigma_H \sigma_\rho \quad (16)$$

Equation (16) is essentially a relation of consistent mass scaling for the shell and the attached mass.

Consider a scale model made of the same material as the prototype, $\sigma_E = \sigma_\rho = 1$, the scaling relations are simplified as shown below:

$$\chi = \lambda^2 \quad (17)$$

$$\sigma_D = \lambda^3 \quad (18)$$

$$\sigma_{\Omega} = \frac{1}{\sqrt{\lambda} \sigma_H} \quad (19)$$

$$\sigma_M = \lambda^2 \sigma_H \quad (20)$$

Equation (19) is reduced to $\sigma_{\Omega} = \frac{1}{\lambda}$ if the special condition $\sigma_H = \lambda$ applies. The corresponding scaling relation (20) is then $\sigma_M = \lambda^3$.

Now consider the dynamic impedance of a shell structure, the impedance amplitude may be defined as:

$$|Z(\omega)| = \frac{|F|}{w\omega^2} \quad (21)$$

which has a dimension of mass. The scaling relation for impedance amplitude is then:

$$\sigma_Z = \lambda^2 \sigma_H \sigma_\rho = \frac{\sigma_D}{\lambda^2 \sigma_Q^2} \quad (22)$$

NUMERICAL DATA

A scale model is designed for a Saturn V sandwich honeycomb shell structure. A linear scaling ratio of $\lambda = 6.67$ is used. The scale model uses solid aluminum sheet 6061-T4 with ring stiffeners. The same material is used in the majority portion of the prototype structure. The prototype shell has the following dimensional and material data:

$$R' = 260"$$

$$H' = 0.1125", h' = 0.042"$$

$$c' = 0.90"$$

$$E' = E = 10.3 \times 10^6 \text{ psi}$$

$$\nu' = \nu = 0.30$$

The other dimensional details including stiffeners, attached weights, and supporting conditions are not given here. They may be computed according to the relations established in the section. Applying Equation (4), the flexural rigidity of the prototype shell is:

$$D' = 1.005 \times 10^5 \text{ lb. in.}$$

Assuming a scale model shell thickness of 0.071", the following scaling ratios are obtained:

$$\sigma_H = H'/h = 1.585$$

$$\sigma_D = \frac{1.005 \times 10^5}{338} = 297$$

$$\sigma_Q = \frac{1}{\sqrt{\sigma_H \lambda}} = \frac{1}{\lambda^2} \sqrt{\frac{\sigma_D}{\sigma_\rho \sigma_H}} = \frac{1}{3.25} \quad (23)$$

According to Equation (10), $\sigma_D = \sigma_E \lambda^3 = 296$ which is consistent with the above computed value. Equation (16) is used to compute the mass scaling ratio which yields:

$$\sigma_M = \frac{\sigma_D}{\lambda^2 \sigma_Q^2} = 70.5 \quad (24)$$

which is also equal to σ_Z according to Equation (22). The driving point dynamic impedances for the prototype and the scale model are plotted in Figure 23. In the plot, the ordinate indicates the impedance in terms of $|Z(\omega)| \cdot g$ so that the unit is (lb.) instead of a mass unit. The prototype data are plotted in a solid line. The scale model data which have been modified using Equations (23), (24), are plotted in a broken line.

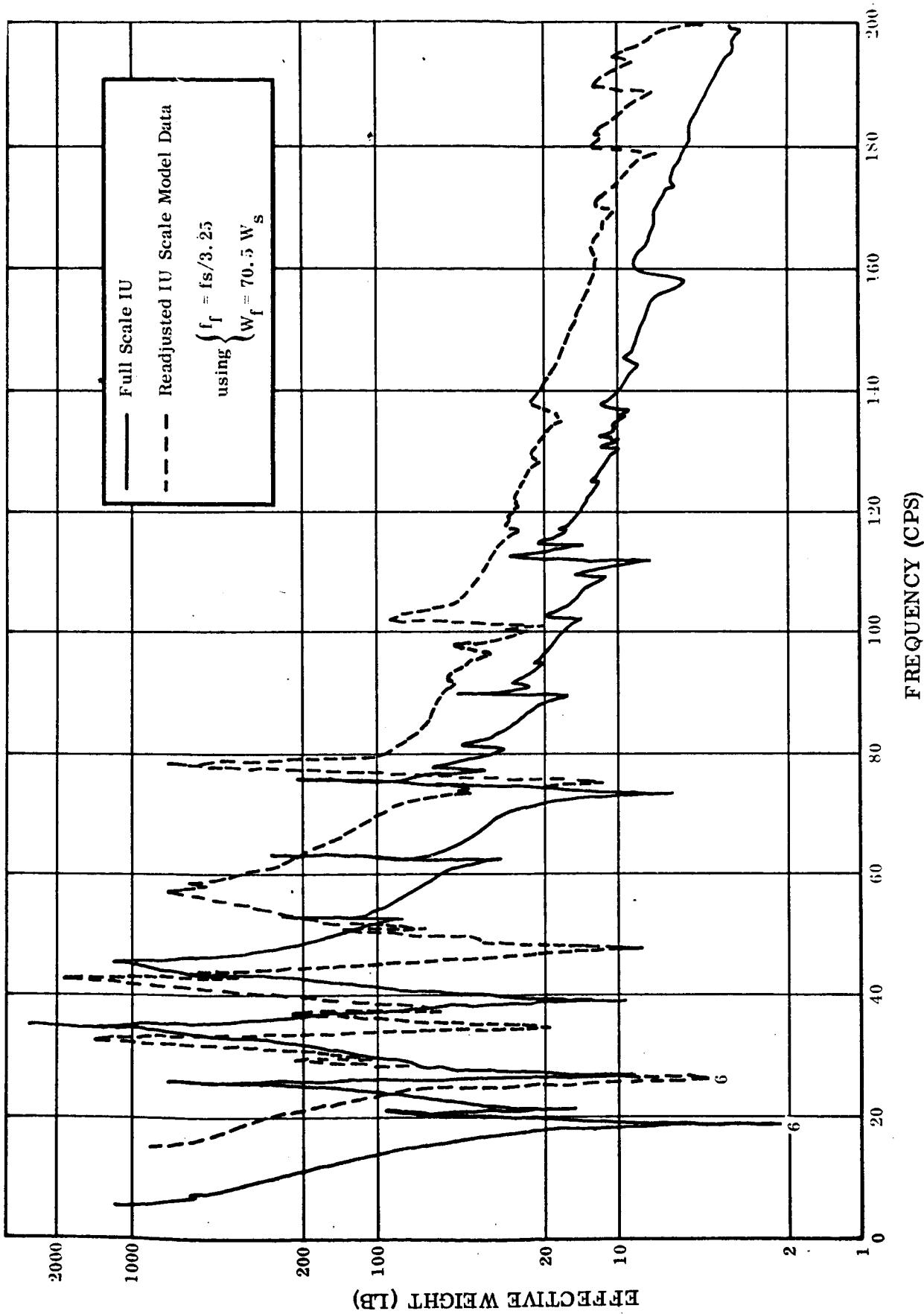


FIGURE 23. DRIVING POINT IMPEDANCE OF INSTRUMENT UNIT SEGMENT WITH COMPONENTS ATTACHED, SUPPORT CONFIGURATION (I)

SECTION VI

CONCLUSIONS AND RECOMMENDATIONS

REVIEW OF ACCOMPLISHED WORK

The shell structures in the various stages of the SATURN V system are large, massive, and intricate. To conduct vibration tests involving the components mounted on the shell structures, correspondingly large and expensive facilities are needed. To perform the tests economically on certain shell-mounted components, with no sacrifice in the validity of the test data, a rational shell segmentation procedure is an attractive approach.

The procedure described in the report involved the determination of the dynamic responses of the complete and uncut shell structure. Based on the response data, a part of the shell structure with the mounted components was determined and segmented. To compensate for the dynamic interaction between the segmented shell and the remaining structure, flexible supports were adapted in the test setup. A criterion in the segmentation and support design was to retain as many as possible of the major vibration modes of the uncut shell structure. The modal data were obtained in the vibration test using a frequency sweep. The segmentation procedure served to demonstrate the feasibility and desirability of the approach.

A shell model scaling technique developed in the contract period will be of special interest to testing engineers. Using the technique, scale models were designed to verify the segmentation procedure. The test data based on the scale model were scaled up to predict the vibration response of the full scale structure. In this manner, the segmented specimen of the full scale SATURN shell structure was designed and fabricated. Applying this technique, the engineer will have sufficient confidence that the designed specimen will have the desired dynamic responses under vibration.

To substantiate the test procedure, a parallel analytical investigation was conducted in the program. Specifically, the overall shell response was investigated using an integration procedure. For the segmented shell with mounted components, a finite difference technique was developed to generate the modal data.

The detailed work items on shell segmentation completed during the contract period are listed below:

1. A rectangular plate with various flexible supports was tested to evaluate the dynamic characteristics of the supports and the effect of point supports to the plate responses.
2. A finite difference computer program was developed to predict the modal data of the flexibly-supported rectangular plate.
3. Four (4) shell scale models were designed and fabricated based on the component-mounted SATURN shell structures. Vibration tests were performed to acquire the modal data. The shell scale model design procedure was formulated and documented in the final report.
4. Differential equations were established for the complete shell of revolution. Detailed formulation of the dynamic impedance of the ring stiffeners was carried out and documented in the first year progress report (Reference 1). A computer program was generated using stepwise integration and considering the ring impedances.
5. A segmentation procedure was established which was applied to two shell scale models. Vibration test data were acquired and documented for the segmented specimens.
6. A finite difference method was used to predict the modal data of the segmented shell. The mass and moment inertia of the attached components were considered, as well as the edge flexible supports. The method was mechanized in a computer program. In general, the analytical data generated by the computer program compared favorably with the test data.
7. A shell model scaling technique described above was applied to the SATURN S-4B instrument unit prototype (full scale) structure. Vibration tests were successfully conducted on the full scale segmented shell structure with mounted components making use of the scale model data acquired previously.

In addition to the work items described above, shell dynamic responses of a more general nature were investigated during the second year of the contract period. The investigation included the responses of the stiffened shell structures under transient and impulsive loadings, and the shell acoustic and blast overpressure

responses. The work conducted on these additional subjects is described in Sections III and IV of the present report. A number of computer programs were generated and checked out for the purpose which are documented in the Appendix.

CONCLUSIONS AND RECOMMENDATIONS

The investigation conducted in the contract demonstrated the feasibility of performing vibration tests for shell-mounted components using a segmented shell structure. A design procedure and related guidelines were formulated for this purpose. (See Reference 2 and the final report.) As is well known to the test engineers, the design and location of the flexible supports have a significant effect on the dynamic responses of the segmented structure. Since the SATURN shell structures are always large and expensive, it is well advisable to conduct a scale model investigation prior to segmentation design and fabrication. In the S-4B instrument unit segmentation test conducted in the contract, it was found that sufficient friction and damping existed in the segmented shell and supports. No additional damping devices were needed to control the vibration amplitudes. For other shell structures such as the S-1C oxidizer tank bulkhead, the inherent damping in the segmented piece may be relatively small. In this case, it may be necessary to adjust the vibration input to regulate the amplitudes. Otherwise, damping devices may be installed at the supporting stations to control the response amplitudes.

In the vibration tests performed, partially because of size and facility limitations, the vibration inputs (shaker heads) were connected to the test specimens. The inputs were regulated so that the desired amplitudes (as functions of the frequency) were reached at specified locations. In certain tests, it may be desirable to install the flexibly-supported specimen on a very rigid structure. The vibration inputs are then supplied through the rigid structure. For the latter case, the procedure reported in the contract may still apply with certain modifications such as the manner the vibration amplitudes are monitored and controlled, etc.

Analytical study on the modal responses of the complete and segmented shell supplied a guideline and gave insight to the problems involved. It is thus always advisable to apply the analytical technique described in the report or similar methods before or during the tests.

To conclude the report, the following specific recommendations are postulated:

1. Use the segmentation procedure for selected component qualification tests on SATURN structures. It may be advisable to start with some hardware where substantial experiences have been accumulated on the testing of a shell structure which was not segmented. When properly applied, substantial savings in time and expenses may be realized in conducting vibration tests using the segmented shells.
2. Explore the use of the finite element method in predicting the modal data of the segmented shell structures. In the contract work, the finite difference method was used to investigate the modal behavior of a segmented and flexibly-supported shell. The finite difference method possesses certain desirable features in dealing with singly or doubly curved shell structures. Using the method, the input data generation to the computer program is somewhat cumbersome. In order that the segmentation technique may be used by a large number of testing engineers, it is advisable to explore the use of the finite element method where the input data may be generated in a more mechanical fashion. It is recommended that the latter approach be explored in future investigations.
3. Further explore the acoustic responses of a stiffened shell structure of the SATURN type. During the contract period, random acoustic tests were performed on the complete instrument unit scale model. Power spectra of the acoustic input, the shell responses and the acoustic pressure inside the enclosed shell were generated. An analytical method was used to predict the shell acoustic response which was described in the present report. Since the preparations of the report draft, the principal investigator has further explored the subject. Using wave propagation equations together with the shell dynamic equations, it was possible to formulate the acoustic spectra inside the shell cavity considering the shell modal data, the shell structural damping as well as the air damping. Since the more delicate components in a SATURN system are located inside the shell structure which are subject to intense acoustic excitation during the launch phase, the analytical approach holds promise to predict the detailed acoustic distribution inside the structure as well as to define possible approaches of reducing the undesirable acoustic excitation. It is recommended that further investigation be carried out in the subject area.

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APPENDIX A

**KEYPUNCH INPUT FORMAT AND LISTINGS
OF THE FINITE DIFFERENCE
COMPUTER PROGRAM**

LISTINGS

The following listing contains the complete main program and its subroutines with the exception of a subroutine named "MITER"*. The MITER subroutine is a standard eigenvalue, eigenvector routine. Its physical package of the subroutine follows CHN5. Input data are also included here. The technique for their arrangements was explained in Section II previously.

Sample runs have been performed using 60 internal and boundary points for the scale mode of the shell panel as shown in Figure 6, Section II. Figure 7 shows the grid point arrangement. The computer output representing detail modal and frequency data were submitted to NASA Marshall Space Flight Center. The data listings are too voluminous to be included in the report. It is suggested that interested personnel contact NASA Marshall Space Flight Center Propulsion and Vehicle Engineering Laboratory for detail information on the program.

*The writeup and subroutine deck of "MITER" are available at IBM SHARE general library.

KEY PUNCH FORM - GENERAL PURPOSE
FORM 20-7-2 (H-7-B-4)

JOB TITLE		FINITE DIFFERENCE PROGRAM		ENGINEER		PAGE
DPW SERIAL NO.	PRE.	CRF. JOB NO.	LASH	FOR ORGN. NO.	ANALYST	DATE
1 2 3 4 5 6 7 8 9 0 1 2 3 4 5 6 7 8 9 0 1 2 3 4 5 6 7 8 9 0 1 2 3 4 5 6 7 8 9 0 1 2 3 4 5 6 7 8 9 0						1 OF 3
INPUT DATA FORMAT						
A		B		C		
H		CORE		BARM		
J1	K2	JWT				
SZX (1)		SZX (2)		SZX (J1)		SZP (1)
SIP (K2)						SZP (2)
SIX (1)		SIX (2)		SIX (J1)		SIP (1)
SIP (K2)						SIP (2)
SAX (1)		SAX (2)		SAX (J1)		SAP (1)
SAP (K2)						SAP (2)
J2	NEXT	FOR NEXT 0, REPEAT THIS AND THE FOLLOWING CARDS				
SAX (J2)		SAP (J2)		SIX (J2)		SIP (J2)
11	12	NXT1	FOR NXT1 0, REPEAT THIS AND THE FOLLOWING CARDS	SIX (J2)	SIP (J2)	SZX (J2)
FOR NXT1 1, THE FOLLOWING IS THE LAST READING FOR SAX, SAP, SIX, SIP, SZX, SZP						SZP (J2)
FOR NXT1 1, THE FOLLOWING IS THE LAST READING FOR BARM						

FINITE DIFFERENCE PROGRAM

FINITE DIFFERENCE PROGRAM

THIS IS TO READ THE ROW NUMBER OF "E TABLE"; THE CORRESPONDING ELEMENTS OF

WHICH ARE TO BE TRANSFERRED TO AA3

THE ELEMENTS ARE TRANSLATED TO ALL

THE ELEMENTS ARE ARRANGED IN ROWS AS ABOVE EACH ELEMENT IS THE SAME AS ABOVE IT.

* * * SAME AS ABOVE EXCEPT THE ELEMENTS ARE TRANSFERRED TO AA2

NC1 NC2 NC3

A diagram illustrating the intersection of three overlapping shaded regions. The regions are labeled AA1, AA2, and AA3. AA1 is at the top left, AA2 is in the center, and AA3 is at the bottom right. The regions overlap in a triangular area in the center.

21

```

$EXECUTE   IBJOB
$IBJOB    GO,MAP
$IBFTC SHELL
C FREE VIBRATION OF CURVED PANEL WITH GRID POINTS OF 60*60
C MAIN PROGRAM TO CALL ALL LINKS IN ORIGIN ALPHA          MAIN
C IN TAPE 9 A33INV*A31, A33INV*A32, AND (A12-A13*A33INV*A32)*A22*AM2
C ARE STORED
C IN TAPE 3 AM2, AM1, A12-A13*A33INV*A32, AND A11-A13*A33INV*A31
C ARE STORED
C IN TAPE 4 E TABLE AND THE FINAL MATRIX ARE STORED      MAIN
C
C CALL CHN1
C CALL CHN2
C CALL CHN3
C CALL CHN4
C CALL CHN5
C
C STOP
C
C SORIGIN
C $IBFTC MLMAX
C
C SUBROUTINE MLTMX (MDA,MDA,N1,M1,A,MDA,MDA,N2,M2,B,NDc,MDc,ABF)    MLTX
C
C **FOR MULTIPLICATION OF REAL MATRICES
C
C AB = A * B
C
C DIMENSION A(MDA,MDA),B(MDB,MDB),AB(77,60),ABF(NDc,MDc)
C
C
10  DO 1 I=1,N1
     DO 1 J=1,M2
        AB(I,J)=0.
        DO 1 K=1,M1
           PRODCT=A(I,K)*B(K,J)
1      AB(I,J)=AB(I,J)+PRODCT
        DO 2 I=1,N1
           DO 2 J=1,M2
2      ABF(I,J)=AB(I,J)
        RETURN
END
C
$IBFTC MATIV
C SUBROUTINE MXIV (A,N,B,M,DETERM)
C
C DIMENSION IPIVOT(77),A(N,N),B(N,1),INDEX(77,2),PIVOT(77)
C
C MATRIX INVERSION WITH ACCOMPANYING SOLUTION OF LINEAR EQUATIONS      MINV0040

```

```

C EQUIVALENCE (IROW,JROW), (ICOLUMN,JCOLUMN), (AMAX, T, SWAP)
C INITIALIZATION
C
C   10 DETERM=1.0
C   15 DO 20 J=1,N
C   20 IPIVOT(J)=0
C   30 DO 550 I=1,N
C
C SEARCH FOR PIVOT ELEMENT
C
C   40 AMAX=0.0
C   45 DO 105 J=1,N
C   50 IF (IPIVOT(J)-1) 60, 105, 60
C   60 DO 100 K=1,N
C   70 IF (IPIVOT(K)-1) 80, 100, 740
C   80 IF (ABS(AMAX)-ABS(A(J,K))) 85, 100, 100
C   85 IROW=J
C   90 ICOLUMN=K
C   95 AMAX=A(J,K)
C 100 CONTINUE
C 105 CONTINUE
C 110 IPIVOT(ICOLUMN)=IPIVOT(ICOLUMN)+1
C
C INTERCHANGE ROWS TO PUT PIVOT ELEMENT ON DIAGONAL
C
C   130 IF (IROW-ICOLUMN) 140, 260, 140
C   140 DETERM=-DETERM
C   150 DO 200 L=1,N
C   160 SWAP=A(IROW,L)
C   170 A(IROW,L)=A(ICOLUMN,L)
C   200 A(ICOLUMN,L)=SWAP
C   205 IF(M) 260, 260, 210
C   210 DO 250 L=1, N
C   220 SWAP=B(IROW,L)
C   230 B(IROW,L)=B(ICOLUMN,L)
C   250 B(ICOLUMN,L)=SWAP
C   260 INDEX(I,1)=IROW
C   270 INDEX(I,2)=ICOLUMN
C   310 PIVOT(I)=A(ICOLUMN,ICOLUMN)

```

```

      320 DETERM=DETERM*PIVOT(I)
C   DIVIDE PIVOT ROW BY PIVOT ELEMENT
C
      330 A(ICOLUMN,ICOLUMN)=1.0
      340 DO 350 L=1,N
      350 A(ICOLUMN,L)=A(ICOLUMN,L)/PIVOT(I)
      355 IF(M) 380, 380, 360
      360 DO 370 L=1,M
      370 B(ICOLUMN,L)=B(ICOLUMN,L)/PIVOT(I)

C   REDUCE NON-PIVOT ROWS
C
      380 DO 550 L1=1,N
      390 IF(L1-ICOLUMN) 400, 550, 400
      400 TA(A(L1,ICOLUMN))
      420 A(L1,ICOLUMN)=0.0
      430 DO 450 L=1,N
      450 A(L1,L)=A(L1,L)-A(ICOLUMN,L)*T
      455 IF(M) 550, 550, 460
      460 DO 500 L=1,M
      500 B(L1,L)=B(L1,L)-B(ICOLUMN,L)*T
      550 CONTINUE
      IF (DETERM.NE.0.) GO TO 600
      WRITE(6,555)
      555 FORMAT(1H0,17H ZERO DETERMINANT)
      STOP

C   INTERCHANGE COLUMNS
C
      600 DO 710 I=1,N
      610 L=N+1-I
      620 IF ((INDEX(L,1)-INDEX(L,2)) 630, 710, 630
      630 JROW=INDEX(L,1)
      640 JCOLUMN=INDEX(L,2)
      650 DO 705 K=1,N
      660 SWAP=A(K,JROW)
      670 A(K,JROW)=A(K,JCOLUMN)
      700 A(K,JCOLUMN)=SWAP
      705 CONTINUE

```

```

710 CONTINUE          MINV0850
740 RETURN           MINV0860
750 END              MINV0870
$ORIGIN ALPHA
$IBFTC CHAN1
      SUBROUTINE CHAN1
C   CHAIN 1 - SETUP FOR E TABLE
C
C   DIMENSION SAX(60),SIX(60),SZX(60),SAP(60),SIP(60),SZP(60),WT(60),  CHN1
C   1  BARKX(60),BARKP(60),WP(60),C2X(60),FX(60),DBXP(60),DBX(60),  CHN1
C   2  CZP(60),FP(60),DPPR(60),DP(60),SKP(60),BIGP(60),BIGR(60),  CHN1
C   3  ALFA(60),BETA(60),AM1(60,60),AM2(35,60),EL(60,80),WX(60),  CHN1
C   4  SKX(60),BIGH(60),IM1(20),IM2(20),AM3(20)  CHN1
COMMON NM,NMODE,E,RHO,H,D,ADP2,BARM,IOPT
C   CHN1
C   FOR SINGLE SHELL IOPT =1
C   FOR SANDWICHED SHELL WITH HONEYCOMB CORE IOPT =2
C
C   10 FORMAT (6E12.8/3E12.8*4I3)  CHN1
C   30 FORMAT (3I3)  CHN1
C   31 FORMAT (2I3)  CHN1
C   32 FORMAT (6E12.8)  CHN1
C   50 FORMAT (6E12.8)  CHN1
C   51 FORMAT (I12,E12.8,I12,E12.8,I12)  CHN1
C   62 FORMAT (2E12.8)  CHN1
C   1 STARTS=1000.  CHN1
C   REWIND 3  CHN1
C   REWIND 4  CHN1
C   READ (5,10) A,B,PHIO,PNU,E,RHO,H,CORE,BARM,N,M,NMODE,IOPT  CHN1
C   N =N-1  CHN1
C   M =M-1  CHN1
C   NM=(N+1)*(M+1)  MAIN
C   DO 15 I=1,NM  MAIN
C   SAX(I)=0.  MAIN
C   SIX(I)=0.  MAIN
C   SZX(I)=0.  MAIN
C   SAP(I)=0.  MAIN
C   SIP(I)=0.  MAIN
C   SZP(I)=0.  MAIN
C   WT(I)=0.  MAIN

```



```

53 READ (5,30) I5,I6,NXT3
      READ (5,62) WX(I5),WX(I6)
      IF (NXT3-1) 53,54,53
54   READ (5,30) I7,I8,NXT4
      READ (5,62) WP(I7),WP(I8)
      IF (NXT4-1) 54,55,54
55   READ (5,32) (WT(I),I=1,JWT)
      WRITE (6,330)
      FORMAT( 8H **AX** )
      WRITE (6,340) (SAX(I),I=1,NM)
      WRITE (6,350)
      FORMAT(// 9H **APHI** )
      WRITE (6,340) (SAP(I),I=1,NM)
      WRITE (6,360)
      FORMAT(// 9H ** ZX **)
      WRITE (6,340) (SZX(I),I=1,NM)
      WRITE (6,370)
      FORMAT(// 9H ** ZP **)
      WRITE (6,340) (SZP(I),I=1,NM)
      WRITE (6,380)
      FORMAT(// 9H ** IX **)
      WRITE (6,340) (SIX(I),I=1,NM)
      WRITE (6,390)
      FORMAT(// 9H ** IP **)
      WRITE (6,340) (SIP(I),I=1,NM)
      WRITE (6,400)
      FORMAT(// 9H ** WX **)
      WRITE (6,340) (WX(I),I=1,N1)
      WRITE (6,410)
      FORMAT(// 9H ** WP **)
      WRITE (6,340) (WP(I),I=1,NM)
      WRITE (6,420)
      FORMAT(// 9H ** WT **)
      WRITE (6,340) (WT(I),I=1,NM)
      WRITE (6,430)
      FORMAT(// 9H ** KX **)
      WRITE (6,340) (BARKX(I),I=1,N1)
      WRITE (6,440)
      FORMAT(// 9H ** KP **)
      WRITE (6,340) (BARKP(I),I=1,NM)

```



```

      DO 142 J=1,NM
142 AM2(I,J) =0.
C
BL3 =BLAM**3
ADP2 =ADP**ADP
APRG =ADP2*RGH
APRL =BLAM*(1.-PNU)*APRG
R65 =0.5*R6
R64 =0.5*R65
C
DO 150 I=2,N1
150 AM2(I,I)=-2.*(D/DBX(I))*((RG5*(SAP(I)*ADP+SAX(I)*DELX)+WX(I))/(BL3MAIN
1*APR6))
AM2(N+2,1) =2.*(RG4*(SAP(1)*ADP+SAX(1)*DELX)+WX(1))/APRL
DO 146 K=2,M1
I =N1*(K-1)+1
L =2*N+3+K
AM2(L,I) =-2.*(D/DP(I))*(BLAM*(R65*(SAX(I)*DELX+SAP(I)*ADP)+WP(I))MAIN
1)/(ADP2*R6H)
146 CONTINUE
WRITE (6,122)
122 FORMAT (1H125HMASS 1 MATRIX - INTERNAL )
DO 121 I=1,NM
WRITE (6,470) I
121 WRITE (6,340) (AM1(I,J),J=1,NM)
WRITE (6,126)
126 FORMAT (/26H MASS 2 MATRIX - BOUNDARY )
DO 152 I=1,M2
WRITE (6,470) I
152 WRITE (6,340) (AM2(I,J),J=1,NM)
C TO MAKE A STORAGE TABLE FOR ELEMENTS USED IN MATRICES
C
BL2 =BLAM*BLAM
BL4 =BL2*BL2
BLD4 =BL4/D
BLD2 =BL2/D
FAC14 =PNU/BL2
FAC19 =PNU*BL2
FAC21 =ADP2*BL2

```

```

      C          ADBL2 =ADP2/BL2
      C          JK=80
      C          DO 165 I=1,NM
      C          DO 165 J=1,JK
      C          165 EL(I,J) =0.
      C          ADBL2 =ADP2/BL2
      C          WRITE (3)((AM2(I,J),J=1,NM),I=1,M2)
      C          WRITE (3)((AM1(K,L),L=1,NM),K=1,NM)
      C          DO 170 J=1,NM
      C          EL(J,1) =1.
      C          EL(J,2) =-4.* (BLD4*DBX(J)+BLD2*BIGH(J))
      C          EL(J,3) =BLD4*DBX(J)
      C          EL(J,4) =-4.* (BLD2*BIGH(J)+DP(J)/D)
      C          EL(J,5) =2.*BLD2*BIGH(J)
      C          EL(J,6) =DP(J)/D
      C          EL(J,7) =2.*FAC21/(A*D)
      C          EL(J,8) =6.*BLD4*DBX(J)+8.*BLD2*BIGH(J)+6.*DP(J)/D
      C          EL(J,9) =-0.5*EL(1,7)
      C          EL(J,11) =ALFA(J)/(BL2*DRX(J))
      C          EL(J,12) =-2.* (1.+EL(J,11))
      C          EL(J,13) =(2.*BARK(1)*ADP2)/(D*BLAM*(1.-PNU))
      C          EL(J,14) =FAC14*DBXP(J)/DBX(J)
      C          EL(J,15) =-2.* (1.+EL(J,14))
      C          EL(J,17) =BL2*BEITA(J)/DP(J)
      C          EL(J,18) =-2.* (1.+EL(J,17))
      C          EL(J,19) =FAC19*DPPR(J)/DP(J)
      C          EL(J,20) =-2.* (1.+EL(J,19))
      C          EL(J,21) =-2.*FAC21*BIGR(J)
      C          EL(J,22) =-0.5*EL(J,21)
      C          EL(J,23) =6.*SKX(J)*BL4/SK+8.*BIGP(J)*BL2+6.*SKP(J)/SK
      C          EL(J,24) =-4.*BL2*(SKX(J)*BL2/SK+BIGP(J))
      C          EL(J,25) =SKX(J)*BL4/SK
      C          EL(J,26) =-4.* (SKP(J)/SK+BIGP(J)*BL2)
      C          EL(J,27) =2.*BIGP(J)*BL2
      C          EL(J,28) =SKP(J)/SK
      C          EL(J,29) =-2.
      C          EL(J,30) =D*(1.-PNU)/A
      C          EL(J,31) =SK*ADBL2*BIGR(J)/SKX(J)

```

```

MAIN          =-4.
EL(J,32)      ==FAC14*SK/SKX(J)
EL(J,33)      ==-(4.+EL(J,33))
EL(J,34)      ==2.*EL(J,31)
EL(J,35)      ==2.*EL(J,31)
EL(J,36)      ==EL(J,12)
EL(J,37)      ==-EL(J,17)
EL(J,38)      ==2.
EL(J,39)      ==2.*EL(J,2)
EL(J,40)      ==2.*EL(J,3)
EL(J,41)      ==2.*EL(J,4)
EL(J,42)      ==2.*EL(J,5)
EL(J,43)      ==2.*EL(J,6)
EL(J,44)      ==EL(J,3)+EL(J,8)
EL(J,45)      ==EL(J,6)+EL(J,8)
EL(J,46)      ==EL(J,3)+EL(J,6)+EL(J,8)
EL(J,47)      ==2.*EL(J,11)
EL(J,48)      ==-EL(J,47)
EL(J,49)      ==4.*EL(J,5)
EL(J,50)      ==4.*EL(J,27)
EL(J,51)      ==2.*EL(J,14)
EL(J,52)      ==EL(J,25)+EL(J,23)
EL(J,53)      ==EL(J,23)+EL(J,28)
EL(J,54)      ==EL(J,17)*2.
EL(J,55)      ==-EL(J,54)
EL(J,56)      ==EL(J,23)+EL(J,25)+EL(J,28)
EL(J,57)      ==2.*EL(J,22)
EL(J,58)      ==2.*EL(J,24)
EL(J,59)      ==2.*EL(J,25)
EL(J,60)      ==2.*EL(J,26)
EL(J,61)      ==2.*EL(J,27)
EL(J,62)      ==2.*EL(J,28)
EL(J,63)      ==2.
EL(J,64)      ==-EL(1,30)
EL(J,65)      ==2.*EL(1,9)
EL(J,66)      ==-1.
EL(J,67)      ==-EL(J,11)
EL(J,69)      ==-EL(J,18)
EL(J,70)      ==2.*EL(J,19)
EL(J,71)      ==4.

MAIN          170 CONTINUE

```



```

      DO 600 I=1,NR
      DO 600 J=1,NC
 600  AA3(I,J) =0.
      READ (4) ((EL(I,J),J=1,71),I=1,NM)
C     READ (5,575) (ETBL(J),J=1,NR)
C
      DO 610 J=1, NR
      L3 =ETBL(J)
      DO 603 K1=1,NC
 603  AROW(K1) =0.
      READ (5,535) NZRO, (NAC(I),NEC(I),I=1,NZRO)
      DO 609 K=1,NZRO
L1=NAC(K)
L2=NEC(K)
 609  AROW(L1) =EL(L3,L2)
      WRITE (4) (AROW(JX),JX=1,NC)
 610  CONTINUE
      REWIND 4
C
      C
      615  WRITE (6,630)
 630  FORMAT (1H1 16H** A31 MATRIX **//)
      READ (4) ((EL(I,J),J=1,71),I=1,NM)
      DO 640 I=1, NR
 640  READ (4) (AA3(I,J),J=1, NC)
      WRITE (6,560) I
      WRITE (6,565) (A31(I,J),J=1,NC1)
 640  WRITE (6,650)
      WRITE (6,650)
 650  FORMAT (1H1 16H** A32 MATRIX **//)
      DO 660 I=1, NR
      WRITE (6,560) I
 660  WRITE (6,565) (A32(I,J),J=1,NC2)
      WRITE (6,670)
 670  FORMAT (1H1 16H** A33 MATRIX **//)
      DO 680 I=1, NR
      WRITE (6,560) I
 680  WRITE (6,565) (A33(I,J),J=1,NC3)
C
      REWIND 4
C

```

```

DO 690 I=1,NR
690 B1(I,1) =1.
CALL MXIV (A33,NR,B1,1,DETR)
CALL MLTMX(NR,NR,A33,NR,NC1,NR,NC1,A31,NR,NC1,A31)
CALL MLTMX (NR,NR,NR,A33,NR,NC2,NR,NC2,A32,NR,NC2,A32)
WRITE (9) ((A31(I,J),J=1,NC1),I=1,NR)
WRITE (9) ((A32(I,J),J=1,NR),I=1,NC2)
RETURN
END
SORGIN ALPHA
$IBFTC CHAN3
C CHAIN 3 - DEFINE AA1 AND SOME INTERMEDIATE ALGEBRA
DIMENSION EXT3(77,60),AA1(60,172),A11(60,60),A12(60,35),A13(60,77)CHN3
DIMENSION EL(60,80),IETBL(60),NAC(50),NEC(50),AROW(172)
EQUIVALENCE (AA1(1,1),A11(1),EL(1)),(AA1(3601),A12(1)),
1 (AA1(5701),A13(1))
COMMON NM
READ (5,316) NR,NC,NC1,NC2,NC3
NR3 =NC3
DO 720 I=1,NR
DO 720 J=1,NC
720 AA1(I,J) =0.
READ (4) ((EL(I,J),J=1,71),I=1,NM)
316 FORMAT (5I3)
READ (5,575) (IETBL(J),J=1,NR)
575 FORMAT (36I2)
DO 736 J=1,NR
L3 =IETBL(J)
C
DO 732 J1=1,NC
732 AROW(J1) =0.
READ (5,535) NZRO,(NAC(I),NEC(I),I=1,NZRO)
DO 730 K=1,NZRO
L1=NAC(K)
L2=NEC(K)
AROW(L1) =EL(L3,L2)
730 CONTINUE
WRITE (4) (AROW(K2),K2=1,NC)
736 CONTINUE

```

```

REWIND 4
      WRITE (6,740)
      FORMAT(1H1 16H** A11 MATRIX **//)
740   READ (4) ((EL(I,J),J=1,71),I=1,NM)
      DO 741 I=1,NR
      READ (4) (AA1(I,J),J=1,NC)
      WRITE (6,745) I
      WRITE (6,750) (A11(I,J),J=1,NC1)
741   REWIND 9
C      535  FORMAT(24I3)
      745  FORMAT(8H ** ROW 12,2H**)
      750  FORMAT(8E15.4)
      WRITE (6,755)
      755  FORMAT(1H1 16H** A12 MATRIX **//)
      DO 751 I=1,NR
      WRITE (6,745) I
      WRITE (6,750) (A12(I,J),J=1,NC2)
      WRITE (6,765)
      765  FORMAT(1H1 16H** A13 MATRIX **//)
      DO 761 I=1,NR
      WRITE (6,745) I
      WRITE (6,750) (A13(I,J),J=1,NC3)
      761  WRITE (9) ((EXT3(I,J),J=1,NC1),I=1,NR3)
      READ (9) ((EXT3(I,J),J=1,NC1),I=1,NR3)
      CALL MLTMX (NR,NC3,NR,NC3,A13,NR3,NC1,EXT3,NC1,EXT3)
      DO 772 I=1,NR
      DO 772 J=1,NC1
      772 A11(I,J) =A11(I,J)-EXT3(I,J)
      READ (9) ((EXT3(I,J),J=1,NC2),I=1,NR3)
      CALL MLTMX (NR,NC3,NR,NC3,A13,NR3,NC1,EXT3,NR3,NC1,EXT3)
      REWIND 4
      DO 746 I=1,NR
      DO 746 J=1,NC2
      746 A12(I,J) =A12(I,J)-EXT3(I,J)
      WRITE (3) ((A12(I,J),J=1,NC2),I=1,NR)
      WRITE (3) ((A11(I,J),J=1,NC1),I=1,NR)
      RETURN
      END
      SORIGIN ALPHA
      SIBFTC CHAN4

```

```

C   SUBROUTINE CHN4
C   CHAIN 4 - DEFINE AA2, AND REDUCTION OF MATRIX WILL BE COMPLETED   CHN4
C   DIMENSION EL(60,80),A21(35,60),A22(35,35),EXT4(60,60),EXT5(60,60)   CHN4
C   IAM1(60,60),AA2(35,95),B1(35,1),B2(60,1),NAC(50),NEC(50),IETBL(35)   CHN4
C   DIMENSION AROW(100)   CHN4
C   EQUIVALENCE (AA2(1),A21(1),EL(1)),(AA2(2101),A22(1))   CHN4
C   COMMON NM   CHN4
C   READ (5,300) NR,NC,NC1,NC2   CHN4
C   300 FORMAT (4I3)   CHN4
C   DO 501 I=1,NR   CHN4
C   DO 501 J=1,NC   CHN4
C   501 AA2(I,J)=0.   CHN4
C   READ (4) ((EL(I,J),J=1,71),I=1,NM)   CHN4
C   575 FORMAT (36I2)   CHN4
C   READ (5,575) (IETBL(J),J=1,NR)   CHN4
C   DO 511 J=1,NR   CHN4
C   L3 =IETBL(J)   CHN4
C   DO 513 J=1,NC   CHN4
C   513 AROW(J1)=0.   CHN4
C   552 READ (5,535) NZRO,(NAC(I),NEC(I),I=1,NZRO)   CHN4
C   DO 510 K=1,NZRO   CHN4
C   L1=NAC(K)   CHN4
C   L2=NEC(K)   CHN4
C   AROW(L1)=EL(L3,L2)   CHN4
C   510 CONTINUE   CHN4
C   WRITE (4) (AROW(J1),J1=1,NC)   CHN4
C   511 CONTINUE   CHN4
C   REWIND 4   CHN4
C   509 WRITE (6,550)   CHN4
C   550 FORMAT (1H1 16H** A21 MATRIX **//)   CHN4
C   READ (4) ((EL(I,J),J=1,71),I=1,NM)   CHN4
C   DO 520 I=1,NR   CHN4
C   READ (4) (AA2(I,J),J=1,NC)   CHN4
C   WRITE (6,560) I   CHN4
C   520 WRITE (6,565) (A21(I,J),J=1,NC1)   CHN4
C   WRITE (6,570)   CHN4
C   560 FORMAT (8H ** ROW I2,2H**)   CHN4
C   565 FORMAT (8E15.4)   CHN4
C   570 FORMAT (1H1 16H** A22 MATRIX **//)   CHN4
C   DO 521 I=1,NR   CHN4

```

```

      WRITE (6,560) I
521      WRITE (6,565) (A22(I,J),J=1,NC2)
      DO 522 I=1,NC2
522      B1(I,1)=1.
      REWIND 3
      FORMAT(24I3)
      CALL MXIV (A22,NC2,B1,1,DETER)
      M2 =NR

      READ(3)((EXT4(I,J),J=1,NM),I=1,M2)
      CALL MLTMX (NC2,NC2,NC2,A22,NM,NM,M2,NM,EXT4,NM,NM,EXT4)
      READ ((3)) ((AM1(I,J),J=1,NM),I=1,NM)
      READ ((3)) ((EXT5(I,J),J=1,NC2),I=1,NM)
      CALL MLTMX (NM,NM, NM,NC2,EXT5,NM,NM,NC2,NM,EXT4,NM,NM,EXT4)
      DO 320 I=1,NM
      DO 320 J=1,NM
      AM1(I,J)=AM1(I,J)-EXT4(I,J)
      WRITE (9) ((EXT4(I,J),J=1,NM),I=1,NM)
      READ(3)((EXT4(I,J),J=1,NM),I=1,NM)
      CALL MLTMX (NC2,NC2,NC2,A22,NR,NR,NC1,NR,NC1,A21)
      CALL MLTMX (NM,NM, NM,NC2,EXT5,NR,NC1,NR,NC1,A21,NM,NM,EXT5)
      DO 420 I=1,NM
      DO 420 J=1,NM
      EXT4(I,J)=EXT4(I,J)-EXT5(I,J)
      DO 780 I=1,NM
      780 B2(I,1)=1.
      CALL MXIV (EXT4,NM,B2,1,DET)
      CALL MLTMX (NM,NM,NM,NM,EXT4,NM,NM,NM,NM,AM1)
      DO 430 I=1,NM
      DO 430 J=1,NM
      EL(I,J)=AM1(I,J)
      WRITE (4) ((AM1(I,J),J=1,NM),I=1,NM)
      WRITE (6,782)
      782 FORMAT(1H1 13HFINAL MATRIX /)
      DO 785 I=1,NM
      WRITE (6,560) I
      WRITE (6,565) (EL(I,J),J=1,NM)
      785 WRITE (6,565) (EL(I,J),J=1,NM)
      RETURN
      END
      SORIGIN          BETA
      SIBFTC         CHAN5

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```

SUBROUTINE CHNS
COMMON NM,NMODE,E,RHO,H,D,ADP2,BARM,IOPT
DIMENSION GUESS(60,1),VECTOR(60,5),EIGVAL(5),TEMP(650),
1 EIGM(60,120),NAKSR(5),NAKDR(5),ITRS(15)
N2 =NM*2
BACKSPACE 4
DO 12 I=1,NM
DO 12 J=1,N2
12 EIGM(I,J) =0.
READ (4) ((EIGM(I,J),J=1,NM),I=1,NM)
C
NC =1
NGUESS =0
N =NM
MAXR =NM
EPSR =0.
EPDP =0.
NITRDP =300
RDP =0.8
RSP =0.8
NITRSP =60
NTAPE =4
NTAPEI =6
C
CALL MITERS (EIGM,GUESS,NGUESS,N,NMODE,MAXR,NC,EPSP,EPDP,NAKSR,
1 NAKDR,NITRSP,NITRDP,RSP,RDP,IR,TEMP,VECTOR,EIGVAL,ITRS,
2 NTAPE,NTAPEI)
RETURN
END
C
SDATA
+195 +02+54 +01+7854 -00+3 -00+103 +08+2588 -0-3PANL0001
+71 -01+1 +01+1712 -04 12 5 5 1 0002
1 12 1 +00+1413 +00+1413 +00+1413 +00+1413 +00+1413 +00
+1413 +00+1413 +00+1413 +00+1413 +00+1413 +00+1413 +00+1413 +00
+1413 +00 +00+1413 +00+1413 +00+1413 +00+1413 +00+1413 +00+1413 +00
+616 -03+616 -03+616 -03+616 -03+616 -03+616 -03+616 -03+616 -03
+616 -03+616 -03+616 -03+616 -03+616 -03+616 -03+616 -03+616 -03
+46 -01+46 -01+46 -01+46 -01+46 -01+46 -01+46 -01+46 -01+46

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22127	24128	27139	25163	25
9	22 2 1 21 33	22103	27104	24105 27114 28115 26116 23117 26118 28127AA30008A
24129	27140	25164	25	AA30008B
10	22 22 21 34	22104	27105	24106 27115 28116 26117 23118 26119 28128AA30009A
24130	27141	25165	25	AA30009B
11	22 23 21 35	22105	27106	24107 27116 28117 26118 53119 26129 27130AA30010A
24131	27142	25166	25	AA30010B
12	22 24 21 36	22106	61107	24117 62118 60119 23130 61131 24143 25167AA30011A
24132	27143	25167	25	AA30011B
13	22 26 21 38	22107	22107	22107 61119 24129 62130 60131 23142 61143 24155AA30012A
24134	27145	25170	28	AA30012B
14	22 27 21 39	22108	25109	27110 24111 27120 28121 26122 23123 26124 28134 28135 28136 28137 28138 27149 25
24135	27146	25171	25	AA30013B
15	22 28 21 40	22109	25110	27111 24112 27121 28122 26123 23124 26125 23125 26126 23127 26127AA30014A
24136	27147	25172	25	AA30014B
16	22 29 21 41	22110	25111	27112 24113 27122 28123 26124 23126 26125 23126 26126 23127 26127AA30015A
24137	27148	25173	25	AA30015B
17	22 30 21 42	22111	25112	27113 24114 27123 28124 26125 23126 26126 23127 26127AA30016A
24138	27149	25174	25	AA30016B
18	22 31 21 43	22112	25113	27114 24115 27124 28125 26126 23128 23129 26129AA30017A
24139	27150	25175	25	AA30017B
19	22 32 21 44	22113	25114	27115 24116 27125 28126 26127 23128 23129 26129AA30018A
24140	27151	25176	25	AA30018B
20	22 33 21 45	22114	25115	27116 24117 27126 28127 26128 23129 26129AA30019A
24141	27152	25177	25	AA30019B
21	22 34 21 46	22115	25116	27117 24118 27127 28128 26129 23129 26129AA30020A
24142	27153	25178	25	AA30020B
22	22 35 21 47	22116	25117	27118 24119 27128 28129 26130 53131 26141AA30021A
24143	27154	25179	25	AA30021B
23	22 36 21 48	22117	25118	27119 24120 62130 60131 23142 61143 24155AA30022A
24144	27155	25180	25	AA30022B
24	22 37 21 49	22118	25119	27120 24122 27123 26133 28134 28135 28136 26137 52138 26138 26139AA30027A
24145	27156	25181	28	AA30023B
25	22 38 21 50	22119	25120	27121 24122 27123 27132 28133 26134 52135 52136 26135 52137 26138AA30026B
24146	27157	25182	28	AA30024B
26	22 39 21 51	22120	25121	27122 24123 24124 27133 28134 28135 28136 26136 52138 26138AA30025A
24147	27158	25183	27	AA30025B
27	22 40 21 52	22121	25122	27123 24125 27134 28135 28136 26137 52139 26139AA30026A

AA30027B	28149	24150	27	22114	25125	27126	24127	27136	28137	26138	52139	26104A	30028A
AA30028B	28150	24151	27	22115	25126	27127	24128	27137	28138	26139	52140	26111A	30029A
AA30029B	28151	24152	27	22117	25128	27129	24130	27139	28140	26141	52142	26143	M30031A
AA30031B	28152	24153	27	22147	21159	22118	25129	27130	24131	27140	28141	26142	M30032A
AA30032B	27154	24155	27	22148	21160	22119	25130	26131	24141	26142	28143	26143	M30033A
AA30033B	28156	24156	27	22157	21162	22121	25132	26133	24143	26144	28145	26145	24
AA30034B	28157	24157	27	22158	21163	22122	25133	26134	24144	26146	28147	26147	28
AA30035B	28159	24159	27	22159	21164	22123	25134	26135	24145	26146	28147	26148	28
AA30036B	28160	24160	27	22160	21165	22124	25135	26136	24146	26146	28148	26149	28
AA30037B	28161	24161	27	22161	21166	22125	25136	26137	24147	26147	28149	26149	28
AA30038B	28162	24162	27	22162	21167	22126	25137	26138	24148	26148	28149	26149	28
AA30039B	28163	24163	27	22163	21168	22127	25138	26139	24149	26149	28150	26150	28
AA30040B	28164	24164	27	22164	21169	22128	25139	26140	24150	26150	28151	26152	28
AA30041B	28165	24165	27	22165	21170	22129	25140	26141	24151	26151	28152	26153	28
AA30042B	28166	24166	27	22166	21171	22130	25141	26142	24152	26152	28153	26154	28
AA30043B	28167	24167	27	22167	21172	22131	25142	26143	24153	26153	28154	26155	28
AA30044B	28168	24168	27	22168	21173	22132	25143	26144	24154	26154	28155	26155	28
AA30045B	28169	24169	27	22169	21174	22133	25144	26145	24155	26155	28156	26156	28
AA30046B	28170	24170	27	22170	21175	22134	25145	26146	24156	26156	28157	26157	28
AA30047B	28171	24171	27	22171	21176	22135	25146	26147	24157	26157	28158	26158	28
AA30048B	28172	24172	27	22172	21177	22136	25147	26148	24158	26158	28159	26159	28
AA30049B	28173	24173	27	22173	21178	22137	25148	26149	24159	26159	28160	26160	28
AA30050B	28174	24174	27	22174	21179	22138	25149	26150	24160	26150	28161	26161	28
AA30051B	28175	24175	27	22175	21180	22139	25150	26151	24161	26151	28162	26162	28
AA30052B	28176	24176	27	22176	21181	22140	25151	26152	24162	26152	28163	26163	28
AA30053B	28177	24177	27	22177	21182	22141	25152	26153	24163	26153	28164	26164	28
AA30054B	28178	24178	27	22178	21183	22142	25153	26154	24164	26154	28165	26165	28
AA30055B	28179	24179	27	22179	21184	22143	25154	26155	24165	26155	28166	26166	28

16	6	7	4	8	8	9	4	10	6	19	5	20	2	21	5	32	3	67	5	68	2	69AA100	8A	
5	85	3103	7115	9163	9																		AA100	8B
16	7	6	8	4	9	8	10	4	11	6	20	5	21	2	22	5	33	3	68	5	69	2	70AA100	9A
5	86	3104	7116	9164	9																		AA100	9B
16	8	6	9	4	10	8	11	4	12	6	21	5	22	2	23	5	34	3	69	5	70	2	71AA10010A	
5	87	3105	7117	9165	9																		AA10010B	
15	9	6	10	4	11	45	12	4	22	5	23	2	24	5	35	3	70	5	71	2	72	5	88AA10011A	
3106	7118	9166	9																			AA10011B		
13	10	43	11	41	12	8	23	42	24	2	36	3	24	2	71	42	72	2	89	3107	7119AA10012A			
9	9167	9																				AA10012B		
16	1	2	2	5	13	8	14	4	15	6	25	2	26	5	37	3	61	3	73	5	74	4	75AA10013A	
5	92	6	96	9108	7120	9																	AA10013B	
16	1	5	2	2	3	5	13	4	14	8	15	4	16	6	25	5	26	2	27	5	38	3	62AA10014A	
13	74	6	97	9109	7121	9																	AA10014B	
16	2	5	3	2	4	5	13	6	14	4	15	8	16	4	17	6	26	5	27	2	28	5	39AA10015A	
3	63	3	98	9110	7122	9																	AA10015B	
16	3	5	4	2	5	5	14	6	15	4	16	8	17	4	18	6	27	5	28	2	29	5	40AA10016A	
3	64	3	99	9111	7123	9																	AA10016B	
16	4	5	5	2	6	5	15	6	16	4	17	8	18	4	19	6	28	5	29	2	30	5	41AA10017A	
3	65	3100	9112	7124	9																		AA10017B	
16	5	6	2	7	5	16	6	17	4	18	8	19	4	20	6	29	5	30	2	31	5	42AA10018A		
3	66	3101	9113	7125	9																		AA10018B	
16	6	5	7	2	8	5	17	6	18	4	19	8	20	4	21	6	30	5	31	2	32	5	43AA10019A	
3	67	3102	9114	7126	9																		AA10019B	
16	7	5	8	2	9	5	18	6	19	4	20	8	21	4	22	6	31	5	32	2	33	5	44AA10020A	
3	68	3103	9115	7127	9																		AA10020B	
16	8	5	9	2	10	5	19	6	20	4	21	8	22	4	23	6	32	5	33	2	34	5	45AA10021A	
3	69	3104	9116	7128	9																		AA10021B	
16	9	5	10	2	11	5	20	6	21	4	22	8	23	4	24	6	33	5	34	2	35	5	46AA10022A	
3	70	3105	9117	7129	9																		AA10022B	
16	10	5	11	2	12	5	21	6	22	4	23	45	24	4	34	5	35	2	36	5	47	3	71AA10023A	
3	7106	9118	7130	9																			AA10023B	
12	11	42	12	2	22	43	23	41	24	8	35	42	36	2	48	3	72	3107	9119	7131AA10024A				
9																						AA10024B		
16	1	3	13	2	14	5	25	8	26	4	27	6	37	2	38	5	49	3	74	5	75	4	76AA10025A	
5	93	6108	9120	7132	9																		AA10025B	
16	2	3	13	5	14	2	15	5	25	4	26	8	27	4	28	6	37	5	38	2	39	5	50AA10026A	
3	75	6109	9121	7133	9																		AA10026B	
16	3	14	5	15	2	16	5	25	6	26	4	27	8	28	4	29	6	38	5	39	2	40AA10027A		
5	51	3110	9122	7134	9																		AA10027B	

16	4	3 15	5 16	2 17	5 26	6 27	4 28	8 29	4 30	6 39	5 40	2 41AA10028A		
5	52	3111	9123	7135	9	2 18	5 27	6 28	4 29	8 30	4 31	6 40	5 41	2 AA10028B
16	5	3 16	5 17	2 18	5 27	6 28	4 29	8 30	4 31	6 40	5 41	2 42AA10029A		
15	53	3112	9124	7136	9	2 19	5 28	6 29	4 30	8 31	4 32	6 41	5 42	2 43AA10030A
16	6	3 17	5 18	2 19	5 28	6 29	4 30	8 31	4 32	6 41	5 42	2 43AA10030B		
15	54	3113	9125	7137	9	2 20	5 29	6 30	4 31	8 32	4 33	6 42	5 43	2 44AA10031A
16	7	3 18	5 19	2 20	5 29	6 30	4 31	8 32	4 33	6 42	5 43	2 44AA10031B		
5	55	3114	9126	7138	9	2 21	5 30	6 31	4 32	8 33	4 34	6 43	5 44	2 45AA10032A
16	8	3 19	5 20	2 21	5 30	6 31	4 32	8 33	4 34	6 43	5 44	2 45AA10032B		
15	56	3115	9127	7139	9	2 22	5 31	6 32	4 33	8 34	4 35	6 44	5 45	2 46AA10033A
16	9	3 20	5 21	2 22	5 31	6 32	4 33	8 34	4 35	6 44	5 45	2 46AA10033B		
15	57	3116	9128	7140	9	2 23	5 32	6 33	4 34	8 35	4 36	6 45	5 46	2 47AA10034A
16	10	3 21	5 22	2 23	5 32	6 33	4 34	8 35	4 36	6 45	5 46	2 47AA10034B		
5	58	3117	9129	7141	9	2 24	5 33	6 34	4 35	45 36	4 46	5 47	2 48	5 59AA10035A
15	11	3 22	5 23	2 24	5 33	6 34	4 35	45 36	4 46	5 47	2 48	5 59AA10035B		
12	12	3 23	42 24	2 34	43 35	41 36	8 47	42 48	2 60	3119	9131	7143AA10036A		
9	15	13	3 25	2 26	5 37	44 38	4 39	6 49	2 50	5 75	5 76	4 77	5 94AA10037A	
6120	9132	7144	9	2 27	5 37	4 38	44 39	4 40	6 49	5 50	2 51	5 76AA10038A		
15	14	3 25	5 26	2 27	5 37	4 38	44 39	4 40	6 49	5 50	2 51	5 AA10038B		
6121	9133	7145	9	2 28	5 37	6 38	4 39	44 40	4 41	6 50	5 51	2 52AA10039A		
15	15	3 26	5 27	2 28	5 37	6 38	4 39	44 40	4 41	6 50	5 51	2 52AA10039B		
6122	9134	7146	9	2 29	5 38	6 39	4 40	44 41	4 42	6 51	5 52	2 53AA10040A		
15	16	3 27	5 28	2 29	5 38	6 39	4 40	44 41	4 42	6 51	5 52	2 53AA10040B		
6123	9135	7147	9	2 30	5 39	6 40	4 41	44 42	4 43	6 52	5 53	2 54AA10041A		
15	17	3 28	5 29	2 30	5 39	6 40	4 41	44 42	4 43	6 52	5 53	2 54AA10041B		
6124	9136	7148	9	2 31	5 40	6 41	4 42	44 43	4 44	6 53	5 54	2 55AA10042A		
15	18	3 29	5 30	2 31	5 40	6 41	4 42	44 43	4 44	6 53	5 54	2 55AA10042B		
5125	9137	7149	9	2 32	5 41	6 42	4 43	44 44	4 45	6 54	5 55	2 56AA10043A		
15	19	3 30	5 31	2 32	5 41	6 42	4 43	44 44	4 45	6 54	5 55	2 56AA10043B		
5126	9138	7150	9	2 33	5 42	6 43	4 44	44 45	4 46	6 55	5 56	2 57AA10044A		
15	20	3 31	5 32	2 33	5 42	6 43	4 44	44 45	4 46	6 55	5 56	2 57AA10044B		
5127	9139	7151	9	2 34	5 43	6 44	4 45	44 46	4 47	6 56	5 57	2 58AA10045A		
15	21	3 32	5 33	2 34	5 43	6 44	4 45	44 46	4 47	6 56	5 57	2 58AA10045B		
5128	9140	7152	9	2 35	5 44	6 45	4 46	44 47	4 48	6 57	5 58	2 59AA10046A		
15	22	3 33	5 34	2 35	5 44	6 45	4 46	44 47	4 48	6 57	5 58	2 59AA10046B		
5129	9141	7153	9	2 36	5 45	6 46	4 47	46 48	4 58	5 59	2 60	5130AA10047A		
14	23	3 34	5 35	2 36	5 45	6 46	4 47	46 48	4 58	5 59	2 60	5130AA10047B		
9142	7154	9										AA10047B		

11 24	3 35	42 36	2 46	43 47	41 48	44 59	42 60	2131	9143	7155	9	AA10049	
11 25	40	37 39	38 42	49 50	451	6 76	42 77	4 95	6132	65144	7	AA10049	
11 26	40	37 42	38 39	39 42	49 450	8 51	4 52	6 77	6133	65145	7	AA10050	
11 27	40	38 42	39 39	40 42	49 650	4 51	8 52	4 53	6134	65146	7	AA10051	
11 28	40	39 42	40 39	41 42	50 651	4 52	8 53	4 54	6135	65147	7	AA10052	
11 29	40	40 42	41 39	42 42	51 652	4 53	8 54	4 55	6136	65148	7	AA10053	
11 30	40	41 42	42 39	43 42	52 653	4 54	8 55	4 56	6137	65149	7	AA10054	
11 31	40	42 42	42 42	43 44	42 55	6 54	4 55	8 56	6138	65150	7	AA10055	
11 32	40	43 42	43 42	44 45	42 54	6 55	4 56	8 57	6139	65151	7	AA10056	
11 33	40	40 44	42 45	45 39	46 42	55 656	4 57	8 58	4 59	6140	65152	7	AA10057
11 34	40	40 45	42 42	46 39	47 42	56 657	4 58	8 59	4 60	6141	65153	7	AA10058
10 35	40	46 46	42 47	39 47	48 39	59 41	60 6143	65155	7	AA10059			
8 36	40	47 47	49 48	39 58	43 59	41 60			AA10060				
35 95	60	35							AA20000				
1 12	3 4	5 6	7 8	9 10 11	12 1	1 2	3 4	5 6 7 8	9 10 11 12	113253749	113253749	AA20000	
9	1 10	13	36 14	67 25	66 61	12 62	11 74	67 78	1 90	11	AA2000		
9	2 10	13	67 14	36 15	67 26	66 61	11 62	12 63	11 79	1	AA2000		
9	3 10	14	67 15	36 16	67 27	66 62	11 63	12 64	11 80	1	AA2000		
9	4 10	15	67 16	36 17	67 28	66 63	11 64	12 65	11 81	1	AA2000		
9	5 10	16	67 17	36 18	67 29	66 64	11 65	12 66	11 82	1	AA2000		
9	6 10	17	67 18	36 19	67 30	66 65	11 66	12 67	11 83	1	AA2000		
9	7 10	18	67 19	36 20	67 31	66 66	11 67	12 68	11 84	1	AA2000		
9	8 10	19	67 20	36 21	67 32	66 67	11 68	12 69	11 85	1	AA2000		
9	9 10	20	67 21	36 22	67 33	66 68	11 69	12 70	11 86	1	AA2000		
9	10 10	21	67 22	36 23	67 34	66 69	11 70	12 71	11 87	1	AA2000		
9	11 10	22	67 23	36 24	67 35	66 70	11 71	12 72	11 88	1	AA2000		
7	12 10	23	48 24	36 36	66 71	47 72			AA20012				
5	1 13	14	1 1	62	66 74	66 90	1		AA20013				
5	2 13	15	2 14	15	1 61	73 14			AA20014				
5	3 14	15	2 15	1 5	14 14	1 62	1		AA20015				
5	4 14	15	3 15	4 14	1 63	1			AA20016				
5	5 14	15	5 15	6 14	1 64	1			AA20017				
5	6 14	7 15	8 15	9 14	1 65	1			AA20018				
5	7 14	8 15	9 14	10 20	1 66	1			AA20019				
5	8 14	9 15	10 15	11 14	22 1	1 70			AA20020				
5	9 14	10 15	11 15	12 14	23 1	1 71			AA20021				
5	10 14	11 15	12 15	24 1	1 72	1			AA20022				
4	11 51	12 15							AA20023				
									AA20024				
									AA20025				

9 1 16 2 69 3 66 14 37 62 37 73 18 74 17 90 17 91 1
9 2 37 13 16 14 69 15 66 26 37 73 17 74 18 75 17 92 1
9 14 37 25 16 26 69 27 66 38 37 74 17 75 18 76 17 93 1
9 26 37 37 16 38 69 39 66 50 37 75 17 76 18 77 17 94 1
7 38 55 49 16 50 69 51 66 76 54 77 18 95 1
5 1 20 2 1 13 19 61 19 73 1
5 1 19 13 20 14 1 25 19 74 1
5 13 19 25 20 26 1 37 19 75 1
5 25 19 37 20 38 1 49 19 76 1
4 37 70 49 20 50 1 77 1

AA200026
AA200027
AA200028
AA200029
AA200030
AA200031
AA200032
AA200033
AA200034
AA200035

APPENDIX B

KEYPUNCH INPUT FORMAT AND LISTINGS OF THE SHELL TRANSIENT RESPONSE COMPUTER PROGRAMS

This appendix includes the input formats, listings and sample run data of three (3) related computer programs. The analyses of the programs are described in Section III of the subject report. The detail data are arranged according to the following order:

1. Natural Frequencies and Modes of Simply Supported Cylindrical Shell

This program provides circumferential harmonic numbers, n , axial half wave numbers, k , and the corresponding natural frequencies and modes in either print or punched cards. For the sample run, thirty lowest natural frequencies for the shell are computed and printed out in sequence. These frequencies are in the range of $n = 2 \sim 30$ and $k = 1, 3, 5, 7, \dots, 15$.

The program also provides the normalized displacement components, u , v , and w for each of the natural modes and the corresponding natural frequencies. For the sample run, values of u , v and w are printed out for 21 axial stations.

2. Transient Response for the General Stiffened Shell

This program provides the transient response solution for displacement components, u , v , and w at a given location of the shell. Two sample runs are included here.

- a. Cylindrical Shell - From the data obtained by using the program, "Natural Frequencies and Modes of Simply Supported Cylindrical Shell," 18 modes for odd values of k are selected for input to this program. The print out is shown for the response of the first mode, as well as the response for 12 modes and for 18 modes.
- b. Instrument Unit Structure - Data obtained from the program "General Stiffened Shell," NOR-66-201, May 1966 of this contract, are directly used as input data to the present program. Final results shown in the output data sheets are the modal summation based on nine natural modes, of the nine modes, two modes are corresponding to $n = 1$; two modes, $n = 2$; three modes, $n = 4$; and two modes, $n = 6$.

3. Natural Frequency and Modes of Mass Attached Cylindrical Shell

This program computes the natural frequencies and mode shapes of a mass attached shell structure. Natural frequencies and mode shapes are computed by using 30, 40 and 50 modes of the unloaded shell. Frequencies and mode shapes that are converged for the 50 mode case are shown here.

**NATURAL FREQUENCIES AND MODES OF SIMPLY
SUPPORTED CYLINDRICAL SHELL**

KEY PUNCH FORM - GENERAL PURPOSE

FORM 20-70B (R.7-63)

```

SEEXECUTE    IBJOB
SIBJOB      GO+MAP
SIBFTC  SAMPL
           DIMENSION B(50,50),C(50,50),OM(50,50),KSTO(50),NSTO(50),
1  OSTO(2000)
REAL  NU,LAMBDA,L,NU2
READ  (5,10) A,H,E,NU,RHO,L,KF,NF,KS,NKF,KINC,NS
10 FORMAT(6E12.8/6I3)
ITER =1
NU2 =NU*NU
LAMBDA =SQRT(E/((1.-NU*NU)*RHO))
A0H4 =(A/H)**4
HOL2 =(H/LAMBDA)**2
CON1 =1./(12.*A0H4*HOL2)
PAOL =3.1415927*A/L
PAOL2 =PAOL*PAOL
PAOL4 =PAOL2*PAOL2
CON2 =12.*((1.-NU2)*(A/H)**2
STAT =NS
DO 90 N=2,NF
XN =N
XN2=XN*XN
DO 90 K=KS,KF,KINC
XK =K
XK2=XK*XK
FAC1 =((XN2+XK2*PAOL2-1.)***2
XK4=XK2*XK2
FAC2 =CON2*XK4*PAOL4/(XN2+XK2*PAOL2)**2
OM(N,K)=SQRT((CON1*(FAC1+FAC2)))
PKAOL =XK*PAOL
PKAOL2=PKAOL*PAOL2
PKN2=(PKAOL2+XN2)**2
B(N,K)=( -NU*PKAOL2+XN2)*PKAOL/PKN2
C(N,K)=((2.+NU)*PKAOL2+XN2)/PKN2
90 CONTINUE
WRITE(6,20)
20 FORMAT(16H1 * INPUT DATA * /)
WRITE(6,25)
25 FORMAT(6X,7H RADIUS,9X,9HTHICKNESS,17X,1HE, 7X,15HPOISSON'S RATIO,CILY

```

C COMPUTATION OF DEFLECTION AT EACH STATION

```

      WRITE (6,36) ITER
 36 FORMAT(//15H MODE NUMBER * I2,2H * /)
      WRITE (6,123)
 123 FORMAT(7X,8H STATION,17X,1HU,17X,1HV,17X,1HW/)
      SX =0.
      UK=K1
      DO 120 IS =1,NS1
      QUANT =UK*POL*SX
      SQ =SIN(QUANT)
      U =B(N1,K1)*COS(QUANT)
      V =C(N1,K1)*SQ
      W =SQ
      WRITE (6,125) SX,U,V,W
      WRITE (7,128) N1,K1,U,V,W
      SX =SX+DX
 120  CONTINUE
 125  FORMAT(4E18.6)
 128  FORMAT (2I10,3E16.6)
      KSTO(ITER) =K1
      NSTO(ITER) =N1

```

```

      111X,3HRHO,12X,6HLENGTH )
C HUNT FOR THE SMALLEST OMEGA AND REARRANGE THEM IN SEQUENCE
      WRITE (6,30) A,H,E,NU,RHO,L
 30  FORMAT (6E18.5)
      SX =L/STAT
      DX =SX
      POL =3.1415927/L
      NS1 =NS+1
 32  SMALL =OM(2,1)
      DO 100 N=2,NF
      DO 100 K=KS,KF,KINC
      IF (OM(N,K).LE.SMALL) GO TO 91
      GO TO 100
 91  N1=N
      K1=K
      SMALL=OM(N,K)
 100  CONTINUE
      WRITE (6,50) N1,K1,OM(N1,K1)
 50  FORMAT(1H1, 7HFOR N =12,3X,3HK =12,3X,11HFREQUENCY =E15.6/)
```

```

OSTO(ITER) = OM(N1,K1)/6.283185
OM(N1,K1) = 5.E+25
ITER = ITER+1
CILY
CILY
CILY
IF(ITER.LE.NKFF) GO TO 32
WRITE (7,138) (NSTO(J),KSTO(J),OSTO(J),J=1,NKFF)
138 FORMAT (2I8,E18.7)
WRITE (6,142)
142 FORMAT (1H1 29HSEQUENCE OF FREQUENCY IN CPS /)
WRITE (6,148)
148 FORMAT (10X,2H N 11X,1HK 11X,4HFREQ,5X,5HORDER /)
DO 145 I=1,NKF
145 WRITE (6,147) NSTO(I),KSTO(I),OSTO(I),I
147 FORMAT (2I12,E18.7,I6)
STOP
END
CILY
CILY
CILY
SDATA
*195   +02+5      -01+103    +08+3      -00+2591    -03+534    +02
15 30  1 30  2 20

```

* INPUT DATA *
RADIUS THICKNESS
0.19500E 02 0.50000E-01
* INPUT DATA *
F POISSON'S RATIO
0.10300E 0.8 0.30000F 00
RHO LENGTH
0.25910E-03 0.53400F 77

FOR N = 6 K = 1 FREQUENCY = 0.461578E 03

NODE NUMBER *	STATION	1 *	U	V	W
0.			0. 293335E-01	0.	0.
0.267000E 01			0.289724E-01	0.438435E-02	0.156434E 00
0.534000E 01			0.278978E-01	0.8666075E-02	0.309017E 00
0.801000E 01			0.261363E-01	0.127239E-01	0.453990E 00
0.106800E 02			0.237313E-01	0.164737E-01	0.587785E 00
0.133500E 02			0.207419E-01	0.198179E-01	0.707107E 00
0.160200E 02			0.172418E-01	0.226741E-01	0.809017E 00
0.186900E 02			0.133171E-01	0.249720E-01	0.891007F 00
0.213600E 02			0.906455E-02	0.266550E-01	0.951057E 00
0.240300E 02			0.458877E-02	0.276817E-01	0.987688F 00
0.267000E 02			0.134034E-08	0.280268E-01	0.100000E 01
0.293700E 02			-0.458877E-02	0.276817E-01	0.987688E 00
0.320400E 02			-0.906455E-02	0.266550E-01	0.951057E 00
0.347100E 02			-0.133171E-01	0.249720E-01	0.891007F 00
0.373800E 02			-0.172418E-01	0.226741E-01	0.809017E 00
0.400500E 02			-0.207419E-01	0.198179E-01	0.707107E 00
0.427200E 02			-0.237313E-01	0.164737E-01	0.587785F 00
0.453900E 02			-0.261363E-01	0.127239E-01	0.453991E 00
0.480600E 02			-0.278978E-01	0.8666075E-02	0.309017E 00
0.507300E 02			-0.289724E-01	0.438436E-02	0.156435F 00
0.534000E 02			-0.293335E-01	0.673757E-08	0.240398E-06

FOR N =11 K = 3 FREQUENCY = 0.138754E 04

MODE NUMBER * 12 *

STATION	U	V	W
0.	0.229042E-01	0.	0.
0.-267000E 01	0.-204078E-01	0.-381357E-02	0.-453990E 00
0.-534000E 01	0.-134628E-01	0.-679584E-02	0.-809017E 00
0.-801000E 01	0.-358301E-02	0.-829670E-02	0.-987688E 00
0.-106800E 02	-0.-707779E-02	0.-798898E-02	0.-951057E 00
0.-133500E 02	-0.-161957E-01	0.-593978E-02	0.-707107E 00
0.-160200E 02	-0.-217832E-01	0.-259578E-02	0.-309017E 00
0.-186900E 02	-0.-226222E-01	-0.-131407E-02	-0.-156434E 00
0.-213600E 02	-0.-185299E-01	-0.-493746E-02	-0.-587785E 00
0.-240300E 02	-0.-103983E-01	-0.-748456E-02	-0.-891006E 00
0.-267000E 02	-0.-382229E-08	-0.-840011E-02	-0.-100000E 01
0.-293700E 02	0.-103983E-01	-0.-748456E-02	-0.-891007E 00
0.-320400E 02	0.-185299E-01	-0.-493747E-02	-0.-587785E 00
0.-347100E 02	0.-226222E-01	-0.-131407E-02	-0.-156435E 00
0.-373800E 02	0.-217832E-01	0.-259578E-02	0.-309017E 00
0.-400500E 02	0.-161957E-01	0.-593978E-02	0.-707106E 00
0.-427200E 02	0.-707780E-02	0.-798898E-02	0.-951056F 00
0.-453900E 02	-0.-358300E-02	0.-829670E-02	0.-987688F 00
0.-480600E 02	-0.-134627E-01	0.-679584E-02	0.-809017E 00
0.-507300E 02	-0.-204078E-01	0.-381358E-02	0.-453991F 00
0.-534000E 02	-0.-229042E-01	0.-680913E-08	0.-810600E-06

FOR N = 13 K = 5 FREQUENCY = 0.230582E 04

MODE NUMBER * 26 *

STATION	U	V	W
0.	0.223914E-01	0.	0.424417E-02
0.267000E 01	0.158331E-01	0.424417E-02	0.707107E 00
0.534000E 01	0.221569E-10	0.600216E-02	0.100000E 01
0.801000E 01	-0.158331E-01	0.424417E-02	0.707107E 00
0.106800E 02	-0.223914E-01	0.190757E-09	0.317814E-07
0.133500E 02	-0.158331E-01	-0.424417E-02	-0.707107E 00
0.160200E 02	-0.106744E-08	-0.600216E-02	-0.100000E 01
0.186900E 02	0.158331E-01	-0.424417E-02	-0.707107E 00
0.213600E 02	0.223914E-01	-0.739271E-09	-0.123167E-06
0.240300E 02	0.158331E-01	0.424417E-02	0.707107E 00
0.267000E 02	0.444833E-08	0.600216E-02	0.100000E 01
0.293700E 02	-0.158331E-01	0.424417E-02	0.707107E 00
0.320400E 02	-0.223914E-01	0.271881E-08	0.452972E-06
0.347100E 02	-0.158331E-01	-0.424417E-02	-0.707106E 00
0.373800E 02	-0.145024E-07	-0.600216E-02	-0.100000E 01
0.400500E 02	0.158331E-01	-0.424417E-02	-0.707107E 00
0.427200E 02	0.223914E-01	-0.505611E-08	-0.842381E-06
0.453900E 02	0.158331E-01	0.424416E-02	0.707106E 00
0.480600E 02	0.232218E-07	0.600216E-02	0.100000F 01
0.507300E 02	-0.158331E-01	0.424417E-02	0.707108E 00
0.534000E 02	-0.223914E-01	0.739340E-08	0.123179E-05

SEQUENCE OF FREQUENCY IN CPS	N	K	ORDER	FREQ
				1
6	1	1	1	0.7346245E 02
7	1	1	2	0.7542803E 02
5	1	1	3	0.8743618E 02
8	1	1	4	0.8758063E 02
9	1	1	5	0.1059216E 03
4	1	1	6	0.1253863E 03
10	1	1	7	0.1284192E 03
11	1	1	8	0.1541813E 03
12	1	1	9	0.1828206E 03
3	1	1	10	0.2079398E 03
13	1	1	11	0.2141621E 03
11	3	3	12	0.2208343E 03
10	3	3	13	0.2220137E 03
12	3	3	14	0.2313555E 03
9	3	3	15	0.2378022E 03
14	1	1	16	0.2481226E 03
13	3	3	17	0.2508561E 03
8	3	3	18	0.2711430E 03
14	3	3	19	0.2771556E 03
15	1	1	20	0.2846603E 03
15	3	3	21	0.3087176E 03
16	1	1	22	0.3237531E 03
7	3	3	23	0.3256807E 03
16	3	3	24	0.3445400E 03
17	1	1	25	0.3653891E 03
13	5	5	26	0.3669834E 03
14	5	5	27	0.3708424E 03
12	5	5	28	0.3754184E 03
17	3	3	29	0.3839920E 03
15	5	5	30	0.3851326E 03

KEY PUNCH FORM - GENERAL PURPOSE

FORM 2-0-708 (R-7-63)

TRANSIENT RESPONSE PROGRAM		
JOB TITLE	DPWA SERIAL NO.	PRE:
		DPD JOB NO.
		DASH
		FOR ORGN. NO.
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		ENGINEER		PAGE 1 OF 2																																																																																																																																																							
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		ALFA		THO																																																																																																																																																							
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KEY PUNCH FORM - GENERAL PURPOSE
FORM 20-708 (R.7-63)

JOB TITLE				TRANSIENT RESPONSE PROGRAM										ENGINEER		PAGE		
DPWA SERIAL NO.		PRE	DPO JOB NO.	DASH		FOR ORGN. NO.		ANALYST				2 OF 2						
W (2) U (2) V (2) BETA (2) *2																		
W (NSTAT) U (NSTAT) V (NSTAT) BETA (NSTAT) *2																		
MATERIAL	AC (1)		AC (2)		—		—		—		—		—		AC (7) *2			
	XA		XB		HA		RHOS		NSEG		KT2		*2					
	RHO		EN		EPhi		IX		IV		RRAD		*3		*2			
	AREA																*3	
																	*2	
*1 FOR SIMPLY SUPPORTED CYLINDRICAL SHELL (NR = 0)																		
*2 FOR GENERAL STIFFENED SHELL (NR ≠ 0)																		
*3 THESE DATA MUST BE PROVIDED FOR EACH INDIVISUAL RING STIFFENER -																		
KT2 = 1, 2 OR NS < 0																		

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      EXECUTE    IBJOB
      SIBJOB
      SIBFTC SHOCK
      C DYNAMIC RESPONSE OF THE INSTRUMENT UNIT DUE TO IMPACT LOADING
      C
      DIMENSION U(50),AC(7),V(50),W(50),BETA(50),DELS(100),FNCT(100),
      1   0(100),FU(20,100),FV(20,100),FW(20,100),LREP(10),SUMU(20,100),
      2   SUMV(20,100),SUMW(20,100),NX(50),NY(50),FFCT(100),QF(100)
      EQUIVALENCE INCH,NX(1)
      COMMON AC,NEGR
      REAL LAMDA,LENS,IX,IY
      READ (5,2) TIN,DELT,TEND,LAMDA,ALFA,THO,OMFC,TEA
      2 FORMAT (6E12.8)
      C NUMBER OF INCREMENT FOR TIME MUST BE LARGER THAN THAT OF SEGMENT CUTS
      READ (5,7) MM,NS,NRESP,NSTAT,LOCFC,MT,ITAO
      READ (5,7) (LREP(I),I =1,NRESP)
      7 FORMAT (10I5)
      7 WRITE (6,214) (LREP(I),I=1,NRESP)
      214 FORMAT (//32H1RESPONSE SOUGHT AT SEGMENT NO. 516 /)
      OMFC =OMFC*6.2831854
      NT 3MT
      IMOD =1
      IFT =ITAO+1
      READ (5,2) (FFCT(I),I=1,IFT)
      T =TIN
      C
      DO 200 J=1,NT
      QF(J) =0.
      DO 200 I=1,NRESP
      SUMU(I,J) =U.
      SUMV(I,J) =U.
      SUMW(I,J) =U.
      200 CONTINUE
      WRITE (6,210)
      210 FORMAT (1H 11HINPUT DATA '/')
      WRITE (6,212) TIN,DELT,LAMDA,TEND,ALFA,THO,OMFC
      212 FORMAT (2X,11H INITIAL =E14.5,3X, 6HDELT =E14.5,3X,7HLAMDA =E14.5,
      13X,6HTEND =E14.5,3X,7HALPHA =E14.5,//2X,16HTHETA OF FORCE =E14.5,
      23X,15HFORCING OMEGA =E14.5/)
      WRITE (6,610)
      C

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610 FORMAT (/1H 29H FORCING FUNCTIONS ARE /)
WRITE (6,401) (FFCT(I),I=1,IFT)

```
C C NM = NUMBER OF MODES TO BE ADDED
C C NS = NUMBER OF SECTIONS
C C NRESP = NUMBER OF POINTS RESPONSE IS SOUGHT
C C NSEG = NUMBER OF SEGMENTS
C C NCH = CIRCUMFERENTIAL WAVE NUMBER
C C NT = NUMBER OF INCREMENTATION OF TIME
C C ITAO = NUMBER OF INCREMENTAL TIME FOR THE FORCING FUNCTION
      5 CONTINUE
      NT =NT
      IF (INR.NE.0) GO TO 400
      READ (5,6) OM,NCH
      OM =OM*6.2831854
      GO TO 402
401 FORMAT (SEG15.6)
409 READ (5,401) Z1,Z2,Z3,Z4,FQ
      OM =FQ*6.2831854
      NCH =Z1
      6 FORMAT (1E12.0,15)
402 CONTINUE
      IF (INR) 6,9,8
      9 READ (5,10) (NX(I),NY(I),U(I),V(I),W(I),I=1,NSTAT)
      GO TO 17
      8 READ (5,32) UMAX,BMAX
      32 FORMAT (3E16.8)
      READ (5,4) (W(I),U(I),V(I),BETA(I),I=1,NSTAT)
      10 FORMAT (2I10,3E16.6)
      4 FORMAT (4E16.8)
      17 KT1 =1
      WRITE (6,326)
      326 FORMAT (1H1)
      IF (INR.NE.0) GO TO 319
      WRITE (6,218)
      218 FORMAT (/17H INPUT U V AND W '/')
      WRITE (6,222) (U(I),V(I),W(I),I=1,NSTAT)
      222 FORMAT (3E18.6)
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      GO TO 324
319  WRITE (6,321)
      WRITE (6,323) (U(I),V(I),W(I),BETA(I)),I=1,NSTAT
321  FORMAT(//25H INPUT U, V, W, AND BETA /,
323  FORMAT(4E18.6))
324  CONTINUE
      IF (UR.EQ.0) GO TO 320
      DO 322 IZ1,NSTAT
      U(I1)=U(I1)*WMAX
      V(I1)=V(I1)*WMAX
      W(I1)=W(I1)*WMAX
      BETA(I1)=BETA(I1)*WMAX
322  CONTINUE
      NSECT=NS
      N1=21
      IF (NSECT) 83,500,50
83   K11=22
      NSECT=NSECT
      50  K1=81
      R3=20.
      DO 12 IZ1,NSECT
      READ (5,85) NDEGR, (AC(I2),12E1.7)
85   FORMAT (I3,7E11.7)
      READ (5,76) XA,XB,HA,RHOS,NSEG,KT2
76   FORMAT (4E12.8,2I3)
      RESV=XB
      DELTX=(XB-XA)/FSEG
      XA=-DELTX
      RH=RHOS*HA
      DO 14 I1=1,NSEG
      XA=XA+DELTX
      XB=XA+DELTX/2.
      CALL YDIF (XB,RB,YPRIME)
      ANGL=ATAN(YPRIME)
      DELS(K1)=DELTX/COS(ANGL)
      RRH=RB*RH
      FNCT(K1)=(U(K1)*U(K1)+V(K1)*V(K1)+W(K1)*W(K1))*RRH
      IF (I.EQ.1) =NSECT.AND.I1.EQ.NSEG.AND.KT1.EQ.2) GO TO 15

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IF (KT2.EQ.0) GO TO 13
IF (KT2.EQ.1.AND.I1.EQ.1) GO TO 15
 10 READ (5,2) RHO,EN,EPHI,IY,IY,RRAD,AREA
 11 R1 = (U(K1)+BETA(K1)*EN)*2+(W(K1)-BETA(K1))*EPHI)**2+V(K1)*V(K1)
 12 R2 = (IX+IY*BETA(K1)*BETA(K1))
 13 R3=R3+(RHO*RRA)*(AREA*R1+R2))
 14 CONTINUE
 15 K1=K1+1
 16 CONTINUE
 17 WRITE (6,142) (FNCT(I),I=1,K1)
 18 WRITE (6,143) (FNCT(I),I=18,6)
 19 NSTA1=NSTAT-1
 20 NSTA1=NSTA1
 21 CALL SHINT (FNCT,DELS,TII,NSTA1)
 22 FORMAT (H18.8) INTERMEDIATE PRINT OUT **//9H FUNCTION)
 23 WRITE (6,144) TII
 24 FORMAT (/1H 13HINTERGRATION =E15.5)
 25 HFL =0.5*ALAM0
 26 SGN =SIGN(ONEON-HFL)
 27 BTCH =ALFA-HFL
 28 COMPUTATION OF Q(T) FOR A FIXED TIME
 29 IT =1
 30 TA0 =0.
 31 FCH =NCH
 32 IF (NR.NE.0) FCH =Z21
 33 Q =LOCFC(COS(FCH*T0)/(TII*GAMA))
 34 EXP =EXP(-0.5*(T-TA0))
 35 SINF =SIN(GAMA*(T-TA0))
 36 IMPT
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IF (I6.6T,IFT) GO TO 412
FST =FFCT(I6)
GO TO 411
412 FST =0.
411 CONTINUE
OF (I6) #FST*SNT*EX
DELS(I6) =DELT
TAO =TAO+DELT
101 CONTINUE
C
      IF (IT.EQ.1) GO TO 107
      IF (IT.LT.2) GO TO 106
      CALL SHINT (OF,DELS,OSUM,IT)
      Q(1T) =Q1*OSUM
      GO TO 107
106 Q(1T) =0.
107 IT =IT+1
      T =T+DELT
      IF (IT.LE.NT) GO TO 100
      NT =NT-1
      WRITE (6,223)
223 FORMAT(//12H VALUES OF Q/)
      WRITE (6,224) (Q(J),J=1,NT)
224 FORMAT(6E18.6)
      CNT =COS (FCM*TETA)
      SNT =SIN (FCM*TETA)
      DO 120 IZ1,NRESP
      K =LREP(I1)
      DO 120 J=1,NT
      FU(I,J) =U(K)*CNT*Q(J)
      FV(I,J) =V(K)*SNT*Q(J)
      FW(I,J) =W(K)*CNT*Q(J)
120 CONTINUE
      WRITE (6,125) NCH
125 FORMAT(1H120HFOR THE WAVE NUMBER 12/)
      WRITE (6,126)
126 FORMAT(14X,2H T,19X,1HU,19X,19X,1HW//)
      T =TIN
      DO 130 I=1,NRESP
      K =LREP(I1)

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      WRITE (6,132) K,TETA
      DO 130 J=1,NT
      WRITE (6,133) T, FV(I,J),FV(I,J),FW(I,J)
      T ET+DELT
      C
      DO 250 I=1,NRESP
      DO 250 J=1,NT
      SUM(I,J)=SUM(I,J)+FU(I,J)
      SUM(I,J)=SUM(I,J)+FV(I,J)
      SUM(I,J)=SUM(I,J)+FW(I,J)
      250 CONTINUE
      T ETIN
      WRITE (6,300)
      300 FORMAT(//45H RESPONSES SUMMED UP WITH THE PREVIOUS MODES //)
      DO 330 I=1,NRESP
      K XLRSP(I)
      WRITE (6,132) K,TETA
      DO 330 J=1,NT
      WRITE (6,133) T,SUM(I,J),SUM(I,J),SUM(I,J)
      T ET+DELT
      330 CONTINUE
      IF (IMOD.GE.NM) GO TO 500
      IMOD=IMOD+1
      60 TO 5
      500 STOP
      END
      SUBFTC DIFFR
      SUBROUTINE YDIFF (X,Y,YPRIME)
      DIMENSION AC(7),COEF(2)
      COMMON AC,NDEGR
      C
      Y JAC(1)
      IF (NDEGR.GE.2) GO TO 651
      YSECND=0.
      IF (NDEGR.EQ.1) GO TO 650
      YPRIME=0.
      RETURN

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```

      650 YPRIME = AC(12)
      651 GO TO 5200
      652 YPRIME = AC(12)
      653 IF(X.EQ.0.0) RETURN
      654 DO 780 I2=2,N2
      655 COEF(I2)=I2
      656 COEF(1)=1
      657 COEF(2)=COEF(1)+2.*AC(12+2)*X***(12-1)
      658 YPRIME=YPRIME+COEF(1)*AC(12+1)*X***(12-1)
      659 Y=Y+AC(12+1)*X**12
      660 DO 649 I2=2,N2
      661 COEF(2)=COEF(1)+2.*FLOAT(I2)
      662 COEF(1)=COEF(2)
      663 COEF(2)=YSECND=YSECND+COEF(2)*AC(12+2)*X***(12-1)
      664 SUM=0.
      665 K1=K-1
      666 DO 667 I=1,K1
      667 SUM=SUM+(F(I1)-F(I1)+F(I1)-F(I1))/DH(I1))
      668 F(I1)=F(I1)+F(I1)+F(I1)-F(I1)
      669 RETURN
      670 END
      671 DO 781 I2=2,N2
      672 COEF(I2)=COEF(12)
      673 COEF(1)=1
      674 COEF(2)=COEF(12)
      675 YSECND=YSECND+COEF(2)*AC(12+2)*X***(12-1)
      676 Y=Y+AC(12+1)*X**12
      677 SUM=SUM+ADU+CORE
      678 ADU=(DH(I1)*DH(I1)/12.)*(FP(I1)-FP(IP1))
      679 FP(I1)=0.5*(FP(I1)-FP(I1))+(FP(I1)+FP(I1))/DH(I1))
      680 FP(I1)=FP(I1)+FP(I1)+FP(I1)-FP(IP1)
      681 SUM=SUM+F(I1)-F(I1)
      682 SUM=SUM+F(I1)+F(I1)
      683 SUM=SUM+F(I1)-F(I1)
      684 SUM=SUM+F(I1)+F(I1)
      685 SUM=SUM+F(I1)-F(I1)
      686 SUM=SUM+F(I1)+F(I1)
      687 SUM=SUM+F(I1)-F(I1)
      688 SUM=SUM+F(I1)+F(I1)
      689 SUM=SUM+F(I1)-F(I1)
      690 SUM=SUM+F(I1)+F(I1)
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      698 SUM=SUM+F(I1)+F(I1)
      699 SUM=SUM+F(I1)-F(I1)
      700 SUM=SUM+F(I1)+F(I1)
      701 SUM=SUM+F(I1)-F(I1)
      702 SUM=SUM+F(I1)+F(I1)
      703 SUM=SUM+F(I1)-F(I1)
      704 SUM=SUM+F(I1)+F(I1)
      705 SUM=SUM+F(I1)-F(I1)
      706 SUM=SUM+F(I1)+F(I1)
      707 SUM=SUM+F(I1)-F(I1)
      708 SUM=SUM+F(I1)+F(I1)
      709 SUM=SUM+F(I1)-F(I1)
      710 FP(I1)=(F(I1)-F(I1))/DH(I1)
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15 GO TO 7
15 FP(IP1) = (F(IP1)-F(I))/DH(I)
15 GO TO 6
2 CONTINUE
RETURN
END

C

SAMPLE RUN

TRANSIENT RESPONSE OF CYLINDRICAL SHELL

RESPONSE SOUGHT AT SEGMENT NO. 11

INPUT DATA T INITIAL = 0. DELT = 0.10000E-02 LAMDA = 0.

THETA OF FORCE = 0. FORCING GMEGA = 0.46158E 03

FORCING FUNCTIONS ARE
0.10000E 01 0.820152E 00 0.422184E 00 -0.645859E -01 -0.495665E 00
-0.753422E 00 -0.778285E 00 -0.581129E 00 -0.234781E 00 0.151338E 00
0.464970E 00 0.622990E 00 0.592672E 00 C.397071E 00 0.104212E 00
-0.195603E 00 -0.417048E 00 -0.503522E 00 -C.440946E 00 -0.258476E 00
-0.169355E -01 0.210824E 00 0.361450E 00 0.398272E 00 0.319628E 00
0.156636E 00 -0.381236E -01 -0.207127E 00 -C.304559E 00 -0.308368E 00
-0.224673E 00 -0.838736E -01 0.698214E -01 0.191953E 00 0.250409E 00
0.233552E 00 0.151939E 00 0.336317E -01 -C.850916E -01 -0.170622E 00
-0.201323E 00

FUNDAMENTAL MODE (1ST MODE)

INPUT U V AND W	0.293335E-01	0.	0.
0.289724E-01	0.438435E-02	0.156434E 00	
0.278978E-01	0.866075E-02	0.309017E 00	
0.261363E-01	0.127239E-01	0.453990E 00	
0.237313E-01	0.164737E-01	0.587785E 00	
0.207419E-01	0.198179E-01	0.707107E 00	
0.172418E-01	0.226741E-01	0.809017E 00	
0.133171E-01	0.249720E-01	0.891007E 00	
0.906455E-02	0.266550E-01	0.951057E 00	
0.458877E-02	0.276817E-01	0.987688E 00	
0.134034E-08	0.280268E-01	0.100000E 01	
-0.458877E-02	0.276817E-01	0.987688E 00	
-0.906455E-02	0.266550E-01	0.951057E 00	
-0.133171E-01	0.249720E-01	0.891007E 00	
-0.172418E-01	0.226741E-01	0.809017E 00	
-0.207419E-01	0.198179E-01	0.707107E 00	
-0.237313E-01	0.164737E-01	0.587785E 00	
-0.261363E-01	0.127239E-01	0.453991E 00	
-0.278978E-01	0.866075E-02	0.309017E 00	
-0.289724E-01	0.438436E-02	0.156435E 00	
-0.293335E-01	0.673757E-08	0.240358E-06	

** INTERMEDIATE PRINT OUT **

FUNCTION	0.217345E-06	0.639824E-05	0.243360E-04	0.522747E-04	0.874795E-04	0.126505E-03
0.165529E-03	0.200734E-03	0.228673E-03	0.246611E-03	0.252792E-03	0.246611E-03	0.246611E-03
0.228673E-03	0.200734E-03	0.165529E-03	0.165529E-03	0.126505E-03	0.874795E-04	0.522749E-04
0.243360E-04	0.639832E-05	-0.000000E-19				

INTEGRATION = 0.21232E-01

VALUES OF Q	-0.000000E-19	0.932778E-05	0.773718E-04	0.146131E-03	0.179613E-03	0.151204E-03
0.556460E-04	-0.873513E-04	-0.237994E-03	-0.348315E-03	-0.377374E-03	-0.377374E-03	-0.304992E-03
-0.139675E-03	0.818599E-04	0.303648E-03	0.465484E-03	0.519755E-03	0.445296E-03	
0.254294E-03	-0.978583E-05	-0.282822E-03	-0.496372E-03	-0.595391E-03	-0.552612E-03	
-0.375823E-03	-0.106154E-03	0.192100E-03	0.447449E-03	0.598830E-03	0.610441E-03	
0.480140E-03	0.239476E-03	-0.544203E-04	-0.332900E-03	-0.532210E-03	-0.608265E-03	
-0.546311E-03	-0.363423E-03	-0.103550E-03	0.173404E-03	0.401408E-03	0.541057E-03	
0.567741E-03	0.475249E-03					

FUR THE WAVE NUMBER 6

T	U	V	W
AT LOCATION OF SEGMENT 11 AND THETA -0.			
0.	-0.	0.	-0.
C.100000E-02	0.125024E-13	-0.	0.932778E-05
0.200000E-02	0.103705E-12	-0.	0.773718E-04
0.300000E-02	0.195866E-12	-0.	0.145131E-03
0.400000E-02	0.240742E-12	-0.	0.179612E-03
0.500000E-02	0.202665E-12	-0.	0.151204E-03
0.600000E-02	0.745846E-13	-0.	0.556460E-04
0.700000E-02	-0.117080E-12	0.	-0.373513E-04
0.800000E-02	-0.318993E-12	0.	-0.237994E-03
C.900000E-02	-0.466860E-12	0.	-0.348315E-03
0.100000E-01	-0.505809E-12	0.	-0.377374E-03
C.110000E-01	-0.408793E-12	0.	-0.304992E-03
0.120000E-01	-0.187212E-12	0.	-0.139675E-03
0.130000E-01	0.109720E-12	-0.	0.818599E-04
0.140000E-01	0.406991E-12	-0.	0.303648E-03
C.150000E-01	0.623907E-12	-0.	0.465484E-03
0.160000E-01	0.696649E-12	-0.	0.519755E-03
0.170000E-01	0.596847E-12	-0.	0.445296E-03
0.180000E-01	0.340841E-12	-0.	0.254294E-03
C.190000E-01	-0.131163E-13	0.	-0.378583E-05
0.200000E-01	-0.379078E-12	0.	-0.282822E-03
0.210000E-01	-0.665308E-12	0.	-0.496372E-03
0.220000E-01	-0.798026E-12	0.	-0.595391E-03
0.230000E-01	-0.740688E-12	0.	-0.552612E-03
0.240000E-01	-0.503730E-12	0.	-0.375823E-03
C.250000E-01	-0.142282E-12	0.	-0.106154E-03
0.260000E-01	0.257480E-12	-0.	0.192100E-03
0.270000E-01	0.599734E-12	-0.	0.447449E-03
C.280000E-01	0.802636E-12	-0.	0.598830E-03
0.290000E-01	0.818199E-12	-0.	0.610441E-03
0.300000E-01	0.643551E-12	-0.	0.480140E-03
C.310000E-01	0.320980E-12	-0.	0.239476E-03
0.320000E-01	-0.729416E-13	0.	-0.544203E-04
0.330000E-01	-0.446199E-12	0.	-0.332900E-03
0.340000E-01	-0.713342E-12	0.	-0.532210E-03
0.350000E-01	-0.815282E-12	0.	-0.608265E-03
0.360000E-01	-0.732242E-12	0.	-0.546311E-03
0.370000E-01	-0.487110E-12	0.	-0.363423E-03
0.380000E-01	-0.138792E-12	0.	-0.103550E-03
C.390000E-01	0.232420E-12	-0.	0.173404E-03
C.400000E-01	0.538023E-12	-0.	0.401408E-03
0.410000E-01	0.725201E-12	-0.	0.541057E-03
0.420000E-01	0.760966E-12	-0.	0.567741E-03
0.430000E-01	0.636995E-12	-0.	0.475249E-03

RESPONSES SUMMED UP WITH THE PREVIOUS MODES
 AT LOCATION OF SEGMENT 11 AND THETA -0.

0.	-0.	0.	-0.
0.100000E-02	0.125024E-13	-0.	0.932778E-05
0.200000E-02	0.103705E-12	-0.	0.773718E-04
0.300000E-02	0.195866E-12	-0.	0.146131E-03
0.400000E-02	0.240742E-12	-0.	0.179613E-03
0.500000E-02	0.202665E-12	-0.	0.151204E-03
0.600000E-02	0.745846E-13	-0.	0.556460E-04
0.700000E-02	-0.117080E-12	0.	-0.873513E-04
0.800000E-02	-0.318993E-12	0.	-0.237994E-03
0.900000E-02	-0.466860E-12	0.	-0.348315E-03
0.100000E-01	-0.505809E-12	0.	-0.377374E-03
0.110000E-01	-0.408793E-12	0.	-0.304992E-03
0.120000E-01	-0.187212E-12	0.	-0.139675E-03
0.130000E-01	0.109720E-12	-0.	0.818599E-04
0.140000E-01	0.406991E-12	-0.	0.303648E-03
0.150000E-01	0.623907E-12	-0.	0.465484E-03
0.160000E-01	0.696649E-12	-0.	0.519755E-03
0.170000E-01	0.596847E-12	-0.	0.445296E-03
0.180000E-01	0.340841E-12	-0.	0.254294E-03
0.190000E-01	-0.131163E-13	0.	-0.978583E-05
0.200000E-01	-0.379078E-12	0.	-0.282822E-03
0.210000E-01	-0.665308E-12	0.	-0.496372E-03
0.220000E-01	-0.798026E-12	0.	-0.595391E-03
0.230000E-01	-0.740688E-12	0.	-0.552612E-03
0.240000E-01	-0.503730E-12	0.	-0.375823E-03
0.250000E-01	-0.142282E-12	0.	-0.106154E-03
0.260000E-01	0.257480E-12	-0.	0.192100E-03
0.270000E-01	0.599734E-12	-0.	0.447449E-03
0.280000E-01	0.802636E-12	-0.	0.598830E-03
0.290000E-01	0.818199E-12	-0.	0.610441E-03
0.300000E-01	0.643551E-12	-0.	0.480140E-03
0.310000E-01	0.320980E-12	-0.	0.239476E-03
0.320000E-01	-0.729416E-13	0.	-0.544203E-04
0.330000E-01	-0.446199E-12	0.	-0.332200E-03
0.340000E-01	-0.713342E-12	0.	-0.532210E-03
0.350000E-01	-0.815282E-12	0.	-0.608265E-03
0.360000E-01	-0.732242E-12	0.	-0.546311E-03
0.370000E-01	-0.487110E-12	0.	-0.363423E-03
0.380000E-01	-0.138792E-12	0.	-0.103550E-03
0.390000E-01	0.232420E-12	-0.	0.173404E-03
0.400000E-01	0.538023E-12	-0.	0.401408E-03
0.410000E-01	0.725201E-12	-0.	0.541057E-03
0.420000E-01	0.760966E-12	-0.	0.567741E-03
0.430000E-01	0.636995E-12	-0.	0.475249E-03

12 TH MODE

INPUT U V AND W

0. 422492E-01	0.	0. 934975E-02	0. 453990E 00
0. 376443E-01	0.	0. 166614E-01	0. 809017E 00
0. 248334E-01	0.	0. 203410E-01	0. 987688E 00
C. 660922E-02	0.	0. 195866E-01	0. 951057E 00
-0. 130557E-01	0.	0. 145626E-01	0. 707107E 00
-0. 298747E-01	0.	0. 636408E-02	0. 309017E 00
-0. 401813E-01	0.	-0. 322170E-02	-0. 156434E 00
-0. 417290E-01	0.	-0. 121052E-01	-0. 587785E 00
-0. 341803E-01	0.	-0. 183499E-01	-0. 891006E 00
-0. 191807E-01	0.	-0. 205946E-01	-0. 100000E 01
-0. 705060E-08	0.	-0. 183499E-01	-0. 891007E 00
0. 191807E-01	0.	-0. 121052E-01	-0. 587785E 00
0. 341803E-01	0.	-0. 322171E-02	-0. 156435E 00
0. 417290E-01	0.	0. 636407E-02	0. 309017E 00
0. 401813E-01	0.	0. 145626E-01	0. 707106E 00
0. 298747E-01	0.	0. 195866E-01	0. 951056E 00
0. 130557E-01	0.	0. 203410E-01	0. 987688E 00
-0. 660920E-02	0.	0. 166614E-01	0. 809017E 00
-0. 248334E-01	0.	0. 934976E-02	0. 453991E 00
-0. 376443E-01	0.	0. 166940E-07	0. 810600E-06

** INTERMEDIATE PRINT OUT **

FUNCTION	0.450878E-06	0.524412E-04	0.165550E-03	0.246527E-03	0.228613E-03	0.126576E-03
0.245386E-04	0.662382E-05	0.876009E-04	0.200710E-03	0.252700E-03	0.200710E-03	0.200710E-03
0.876009E-04	0.662390E-05	0.245386E-04	0.126575E-03	0.228612E-03	0.246527E-03	0.246527E-03
0.165550E-03	0.524415E-04	-0.000000E-19				

INTEGRATION = 0.21308E-01

VALUES OF Q	-0.000000E-19	-0.422568E-05	-0.113635E-04	0.382142E-05	-0.586285E-05	-0.103500E-04
0.858525E-05	0.304905E-05	-0.424718E-05	-0.122251E-04	0.112251E-04	0.444557E-05	-0.752619E-05
0.575545E-05	0.916374E-06	-0.123152E-04	0.180270E-05	0.214209E-05	-0.971702E-05	
0.515085E-05	0.786973E-05	-0.667758E-05	0.489908E-05	0.807086E-05	-0.865679E-05	
0.580247E-06	0.567461E-05	-0.107361E-04	-0.152811E-05	0.749353E-05	-0.755542E-05	
0.658119E-07	0.102515E-04	-0.551244E-05	-0.109428E-05	0.965211E-05	-0.628136E-05	
-0.45567CE-05	0.780866E-05	-0.6155505E-05	-0.562301E-05	0.109379E-04	-0.213606E-05	
-0.867100E-05	0.102398E-04					

FOR THE WAVE NUMBER 7

T	U	V	W
AT LOCATION OF SEGMENT 11 AND THETA -0.			
0.	0.	-0.	0.
0.100000E-02	0.297936E-13	-0.	0.422568E-05
0.200000E-02	0.801194E-13	-0.	0.113635E-04
0.300000E-02	-0.269433E-13	0.	-0.382142E-05
0.400000E-02	0.413366E-13	-0.	0.586285E-05
0.500000E-02	0.729738E-13	-0.	0.103500E-04
0.600000E-02	-0.605312E-13	0.	-0.858525E-05
0.700000E-02	-0.214976E-13	0.	-0.304905E-05
0.800000E-02	0.299451E-13	-0.	0.424718E-05
0.900000E-02	-0.791440E-13	0.	-0.112251E-04
0.100000E-01	-0.313440E-13	0.	-0.444557E-05
0.110000E-01	0.530642E-13	-0.	0.752619E-05
0.120000E-01	-0.405794E-13	0.	-0.575545E-05
0.130000E-01	-0.646096E-14	0.	-0.916374E-06
0.140000E-01	0.868298E-13	-0.	0.123152E-04
0.150000E-01	-0.127101E-13	0.	-0.180270E-05
0.160000E-01	-0.151030E-13	0.	-0.214209E-05
0.170000E-01	0.685108E-13	-0.	0.971702E-05
0.180000E-01	-0.363166E-13	0.	-0.515085E-05
0.190000E-01	-0.554864E-13	0.	-0.786973E-05
0.200000E-01	0.470810E-13	-0.	0.667758E-05
0.210000E-01	-0.345415E-13	0.	-0.489908E-05
0.220000E-01	-0.569044E-13	0.	-0.807096E-05
0.230000E-01	0.610355E-13	-0.	0.865679E-05
0.240000E-01	-0.409109E-14	0.	-0.590247E-06
0.250000E-01	-0.400094E-13	0.	-0.567461E-05
0.260000E-01	0.756957E-13	-0.	0.107361E-04
0.270000E-01	0.107741E-13	-0.	0.152811E-05
0.280000E-01	-0.528339E-13	0.	-0.749353E-05
0.290000E-01	0.532703E-13	-0.	0.755542E-05
0.300000E-01	-0.464013E-15	0.	-0.658119E-07
0.310000E-01	-0.722789E-13	0.	-0.102515E-04
0.320000E-01	0.388660E-13	-0.	0.551244E-05
0.330000E-01	0.771530E-14	-0.	0.109428E-05
0.340000E-01	-0.680531E-13	0.	-0.965211E-05
0.350000E-01	0.442874E-13	-0.	0.628136E-05
0.360000E-01	0.321275E-13	-0.	0.455670E-05
0.370000E-01	-0.550558E-13	0.	-0.780866E-05
0.380000E-01	0.433968E-13	-0.	0.615505E-05
0.390000E-01	0.396456E-13	-0.	0.562301E-05
0.400000E-01	-0.771189E-13	0.	-0.109379E-04
0.410000E-01	0.150605E-13	-0.	0.213606E-05
0.420000E-01	0.611358E-13	-0.	0.857100E-05
0.430000E-01	-0.721970E-13	0.	-0.102398E-04

RESPONSES SUMMED UP WITH THE PREVIOUS MODES
 AT LOCATION OF SEGMENT 11 AND THETA -0.

0.	0.	-0.	0.
0.100000E-02	0.237106E-12	-0.	0.936246E-04
0.200000E-02	0.135517E-11	-0.	0.648561E-03
0.300000E-02	0.137948E-11	0.	0.941250E-03
0.400000E-02	0.862897E-12	-0.	0.861419E-03
0.500000E-02	0.380282E-12	-0.	0.455855E-03
0.600000E-02	-0.322487E-12	0.	-0.232754E-03
0.700000E-02	-0.130185E-11	0.	-0.103806E-02
0.800000E-02	-0.220627E-11	-0.	-0.165528E-02
0.900000E-02	-0.238690E-11	0.	-0.175555E-02
0.100000E-01	-0.145750E-11	0.	-0.121734E-02
0.110000E-01	-0.681997E-13	-0.	-0.280986E-03
0.120000E-01	0.101823E-11	0.	0.680098E-03
0.130000E-01	0.176112E-11	0.	0.141761E-02
0.140000E-01	0.209425E-11	-0.	0.177933E-02
0.150000E-01	0.195912E-11	0.	0.170735E-02
0.160000E-01	0.155218E-11	0.	0.124892E-02
0.170000E-01	0.659437E-12	-0.	0.467264E-03
0.180000E-01	-0.654193E-12	0.	-0.461653E-03
0.190000E-01	-0.151686E-11	0.	-0.122921E-02
0.200000E-01	-0.185853E-11	-0.	-0.168648E-02
0.210000E-01	-0.223067E-11	0.	-0.182861E-02
0.220000E-01	-0.212898E-11	0.	-0.153800E-02
0.230000E-01	-0.111689E-11	-0.	-0.779191E-03
0.240000E-01	0.161600E-12	0.	0.218803E-03
0.250000E-01	0.137791E-11	0.	0.118348E-02
0.260000E-01	0.248906E-11	-0.	0.187061E-02
0.270000E-01	0.288257E-11	-0.	0.202060E-02
0.280000E-01	0.221831E-11	0.	0.154734E-02
0.290000E-01	0.837802E-12	-0.	0.631260E-03
0.300000E-01	-0.685051E-12	0.	-0.398635E-03
0.310000E-01	-0.176393E-11	0.	-0.121212E-02
0.320000E-01	-0.233297E-11	-0.	-0.163347E-02
0.330000E-01	-0.245724E-11	-0.	-0.160823E-02
0.340000E-01	-0.161469E-11	0.	-0.111434E-02
0.350000E-01	-0.414237E-13	-0.	-0.343661E-03
0.360000E-01	0.997425E-12	-0.	0.337047E-03
0.370000E-01	0.133834E-11	0.	0.783028E-03
0.380000E-01	0.158366E-11	-0.	0.104177E-02
0.390000E-01	0.143434E-11	-0.	0.106687E-02
0.400000E-01	0.650088E-12	0.	0.808989E-03
0.410000E-01	0.909030E-13	-0.	0.473098E-03
0.420000E-01	-0.136445E-12	-0.	0.128117E-03
0.430000E-01	-0.593298E-12	0.	-0.299708E-03

18 TH MODE

INPUT U	V AND W
0.330657E-01	0.
0.233810E-01	0.105181E-01
0.327195E-10	0.148748E-01
-C.233810E-01	0.105181E-01
-0.330657E-01	0.472742E-09
-0.233810E-01	-0.105181E-01
-0.157631E-08	-0.148748E-01
0.233810E-01	-0.105181E-01
0.330657E-01	-0.183209E-08
0.233810E-01	0.105181E-01
0.656892E-08	0.148748E-01
-0.233810E-01	0.105181E-01
-0.330657E-01	0.673787E-08
-0.233810E-01	-0.105181E-01
-0.214159E-07	-0.148748E-01
0.233809E-01	-0.105181E-01
0.330657E-01	-0.125303E-07
0.233810E-01	0.105181E-01
0.342919E-07	0.148748E-01
-0.233809E-01	0.105181E-01
-0.330657E-01	0.183226E-07

0.	0.105181E-01	0.707107E 00
	0.148748E-01	0.100000E 01
	0.105181E-01	0.707107E 00
	0.472742E-09	0.317814E-07
	-0.105181E-01	-0.707107E 00
	-0.148748E-01	-0.100000E 01
	-0.105181E-01	-0.707107E 00
	-0.183209E-08	-0.123167E-06
	0.105181E-01	0.707107E 00
	0.148748E-01	0.100000E 01
	0.105181E-01	0.707107E 00
	0.673787E-08	0.452972E-06
	-0.105181E-01	-0.707106E 00
	-0.148748E-01	-0.100000E 01
	-0.105181E-01	-0.707107E 00
	-0.125303E-07	-0.842381E-06
	0.105181E-01	0.707106E 00
	0.148748E-01	0.100000E 01
	0.105181E-01	0.707108E 00
	0.183226E-07	0.123179E-05

** INTERMEDIATE PRINT OUT **

FUNCTION				
0.276170E-06	0.126463E-03	0.252649E-03	0.126463E-03	0.276170E-06
0.252649E-03	0.126463E-03	0.276170E-06	0.126463E-03	0.252649E-03
0.276170E-06	0.126462E-03	0.252649E-03	0.126463E-03	0.276170E-06
0.252649E-03	0.126463E-03	-0.000000E-19		

INTEGRATION = 0.21347E-01

VALUES CF Q				
-0.000000E-19	-0.109097E-05	-0.102131E-05	-0.276692E-06	-0.342510E-05
-0.295834E-05	0.253376E-05	-0.444216E-06	0.144589E-05	0.180027E-05
0.232468E-05	-0.283119E-05	0.145677E-05	-0.267079E-05	0.231377E-06
-0.817434E-06	0.210306E-05	-0.158893E-05	0.322126E-05	-0.177054E-05
-0.931259E-06	-0.663195E-06	0.732295E-06	-0.266138E-05	0.216892E-05
0.224829E-05	-0.923296E-06	0.756972E-06	0.13768E-05	-0.135891E-05
-0.265106E-05	0.200324E-05	-0.217502E-05	0.741259E-06	0.302031E-06
0.105215E-05	-0.120876E-05			-0.674293E-06

FOR THE WAVE NUMBER 8

T

U

V

W

AT LOCATION OF SEGMENT 11 AND THETA -0.

0.	-0.	0.	-0.
C.100000E-02	-0.716653E-14	0.	-0.109097E-05
0.200000E-02	-0.670891E-14	0.	-0.102131E-05
0.300000E-02	-0.181757E-14	0.	-0.276692E-06
0.400000E-02	-0.224992E-13	0.	-0.342510E-05
0.500000E-02	0.103817E-13	-0.	0.158042E-05
0.600000E-02	-0.194331E-13	0.	-0.295834E-05
0.700000E-02	0.166441E-13	-0.	0.253376E-05
0.800000E-02	-0.291802E-14	0.	-0.444216E-06
0.900000E-02	0.949797E-14	-0.	0.144589E-05
0.100000E-01	0.118258E-13	-0.	0.180027E-05
0.110000E-01	-0.635653E-14	0.	-0.967668E-06
0.120000E-01	0.152706E-13	-0.	0.232468E-05
0.130000E-01	-0.185979E-13	0.	-0.283119E-05
0.140000E-01	0.956942E-14	-0.	0.145677E-05
0.150000E-01	-0.175442E-13	0.	-0.267079E-05
0.160000E-01	0.151990E-14	-0.	0.231377E-06
0.170000E-01	-0.337987E-14	0.	-0.514525E-06
0.180000E-01	-0.536966E-14	0.	-0.817434E-06
0.190000E-01	0.138148E-13	-0.	0.210306E-05
0.200000E-01	-0.104376E-13	0.	-0.158893E-05
0.210000E-01	0.211602E-13	-0.	0.322126E-05
0.220000E-01	-0.116306E-13	0.	-0.177054E-05
0.230000E-01	0.129564E-13	-0.	0.197237E-05
0.240000E-01	-0.611737E-14	0.	-0.931259E-06
0.250000E-01	-0.435648E-14	0.	-0.663195E-06
0.260000E-01	0.481039E-14	-0.	0.732295E-06
0.270000E-01	-0.174824E-13	0.	-0.266138E-05
0.280000E-01	0.142469E-13	-0.	0.216883E-05
0.290000E-01	-0.174433E-13	0.	-0.265544E-05
0.300000E-01	0.147689E-13	-0.	0.224829E-05
0.310000E-01	-0.606506E-14	0.	-0.923296E-06
0.320000E-01	0.497249E-14	-0.	0.756972E-06
0.330000E-01	0.747332E-14	-0.	0.113768E-05
0.340000E-01	-0.892660E-14	0.	-0.135891E-05
0.350000E-01	0.148167E-13	-0.	0.225557E-05
0.360000E-01	-0.174146E-13	0.	-0.265106E-05
0.370000E-01	0.131591E-13	-0.	0.200324E-05
0.380000E-01	-0.142875E-13	0.	-0.217502E-05
0.390000E-01	0.486927E-14	-0.	0.741259E-06
0.400000E-01	0.198402E-14	-0.	0.302031E-06
0.410000E-01	-0.443072E-14	0.	-0.674498E-06
0.420000E-01	0.691151E-14	-0.	0.105215E-05
0.430000E-01	-0.794023E-14	0.	-0.120876E-05

RESPONSES SUMMED UP WITH THE PREVIOUS MODES
 AT LOCATION OF SEGMENT 11 AND THETA -0.

0.	-0.	0.	-0.
0.100000E-02	0.265296E-12	C.	0.977082E-04
0.200000E-02	0.139653E-11	0.	0.654515E-03
0.300000E-02	0.136591E-11	0.	0.939422E-03
0.400000E-02	0.959723E-12	0.	0.875360E-03
0.500000E-02	0.354656E-12	-0.	0.452133E-03
0.600000E-02	-0.290475E-12	0.	-0.227206E-03
0.700000E-02	-0.130792E-11	-0.	-0.103972E-02
0.800000E-02	-0.226630E-11	0.	-0.166290E-02
0.900000E-02	-0.235576E-11	-0.	-0.175228E-02
0.100000E-01	-0.155016E-11	-0.	-0.122951E-02
0.110000E-01	-0.321309E-13	C.	-0.276787E-03
0.120000E-01	0.994186E-12	-C.	0.677177E-03
0.130000E-01	0.175917E-11	0.	0.141705E-02
0.140000E-01	0.215848E-11	-0.	0.178861E-02
0.150000E-01	0.193019E-11	C.	0.170336E-02
0.160000E-01	0.160804E-11	-C.	0.125693E-02
0.170000E-01	0.661157E-12	C.	0.467344E-03
0.180000E-01	-0.676882E-12	0.	-0.464326E-03
0.190000E-01	-0.148538E-11	-0.	-0.122564E-02
0.200000E-01	-0.192863E-11	C.	-0.169588E-02
0.210000E-01	-0.220659E-11	-C.	-0.182640E-02
0.220000E-01	-0.215538E-11	0.	-0.154080E-02
0.230000E-01	-0.114785E-11	-0.	-0.734294E-03
0.240000E-01	0.226316E-12	C.	0.228203E-03
0.250000E-01	0.130559E-11	C.	0.117352E-02
0.260000E-01	0.258800E-11	-C.	0.188408E-02
0.270000E-01	0.283310E-11	C.	0.201447E-02
0.280000E-01	0.225690E-11	-C.	0.155201E-02
0.290000E-01	0.843425E-12	C.	0.632784E-03
0.300000E-01	-0.714869E-12	-0.	-0.403358E-03
0.310000E-01	-0.174243E-11	0.	-0.120884E-02
0.320000E-01	-0.236335E-11	-C.	-0.163793E-02
0.330000E-01	-0.247826E-11	-0.	-0.161120E-02
0.340000E-01	-0.157436E-11	0.	-0.110875E-02
0.350000E-01	-0.115915E-12	-0.	-0.354015E-03
0.360000E-01	0.108740E-11	C.	0.349484E-03
0.370000E-01	0.127115E-11	-C.	0.773893E-03
0.380000E-01	0.164208E-11	0.	0.104983E-02
0.390000E-01	0.142499E-11	-C.	0.106562E-02
0.400000E-01	0.627075E-12	-0.	0.805935E-03
0.410000E-01	0.114650E-12	0.	0.475950E-03
0.420000E-01	-0.149737E-12	-0.	0.126941E-03
0.430000E-01	-0.599976E-12	C.	-0.301424E-03

SAMPLE RUN FOR INSTRUMENT UNIT STRUCTURE

RESPONSE SOUGHT AT SEGMENT NO.

12

INPUT DATA				
T INITIAL =	0.	DELT =	0.10000E-02	LAMDA = 0.
THETA CF FORCE =	0.	FORCING OMEGA =	0.54789E 03	
FORCING FUNCTIONS ARE				
0.10000E 01	0. 820152E 00	0.422184E 00	-C.645859E-01	-0.495665E 00
-0.753422E 00	-0. 778285E 00	-0.581129E 00	-C.234781E 00	0.151338E 00
0.464970E 00	0. 622990E 00	0.592672E 00	C.397071E 00	0.104212E 00
-0.195603E 00	-0. 417048E 00	-0.503522E 00	-C.440946E 00	-0.258476E 00
-0.169355E-01	0. 210824E 00	0.361450E 00	C.398272E 00	0.319623E 00
0.156636E 00	-0. 381236E-01	-0.207127E 00	-C.304559F 00	-0.309368E 00
-0.224673E 00	-0. 338736E-01	0.698214E-01	C.191953F 00	0.250409E 00
0.233552E 00	0. 151939E 00	0.336317E-C1	-C.350916E-01	-0.179522E 00
-0.2C1323E 00				

INPUT U, V, W, AND BETA

-0.364019E-01	-0.975416E-01	0.377846E 00	0.310211E 00
-0.355704E-01	-0.118718E 00	0.474544E 00	-0.764732E 00
-0.344444E-C1	-0.139687E 00	0.556454E 00	-0.688052E 00
-0.326332E-01	-0.159656E 00	0.635008E 00	-0.651468E 00
-0.302061E-01	-0.178290E 00	0.708289E 00	-0.601011E 00
-0.272199E-01	-0.195265E 00	0.775066E 00	-0.540247E 00
-0.237383E-01	-0.210291E 00	0.834189E 00	-0.470234E 00
-0.198305E-01	-0.223114E 00	0.884654E 00	-0.392262E 00
-0.155702E-01	-0.233519E 00	0.925611E 00	-0.307610E 00
-0.110331E-01	-0.241333E 00	0.956258E 00	-0.219447E 00
-0.634338E-02	-0.246398E 00	0.979124E 00	-0.997624E-01
-0.378789E-C2	-0.248586E 00	0.987975E 00	-0.758307E-01
-0.121137E-02	-0.249623E 00	0.992633E 00	-0.522090E-01
0.125540E-02	-0.249393E 00	0.100000E 01	-0.485778E-01
0.624893E-02	-0.241340E 00	0.974052E 00	0.330556E 00
0.112177E-01	-0.230217E 00	0.929688E 00	0.422164E 00
0.160670E-01	-0.216042E 00	0.873348E 00	0.525092E 00
0.206871E-01	-0.198860E 00	0.804944E 00	0.625006E 00
0.249614E-01	-0.178788E 00	0.724932E 00	0.719894E 00
0.287661E-01	-0.156017E 00	0.634075E 00	0.806941E 00
0.319702E-C1	-0.130830E 00	0.533490E 00	0.983007E 00
0.344364E-01	-0.103603E 00	0.424665E 00	0.944598E 00
0.360109E-01	-0.748163E-01	0.309989E 00	0.100000E 01
0.368266E-01	-0.454092E-01	0.188251E 00	0.924387E-01

** INTERMEDIATE PRINT OUT **

FUNCTION	0.763144E-14	0.119510E-13	0.164140E-13	0.213527E-13	0.265485E-13	0.317761E-13
0.367970E-13	0.413740E-13	0.452862E-13	0.493298E-13	0.719190E-13	0.732218E-13	0.732218E-13
0.739082E-13	0.521627E-13	0.483005E-13	0.429504E-13	0.369766E-13	0.306272E-13	0.306272E-13
0.242087E-13	0.180412E-13	0.124378E-13	0.767816E-14	0.399487E-14	-0.000000E-19	-0.000000E-19

INTEGRATION = 0.60324E-11

VALUES OF Q	0.448227E 00	0.3642C8E 01	0.659503F 01	0.744444E 01	0.502172E 01
-0.000000E-19	-0.754830E 00	-0.136128E 02	-0.161694E 02	-0.136416E 02	-0.610530E 01
-0.503088E 00	0.151483E 01	0.221350E 02	0.28423E 02	0.164250E 02	0.429778E 01
0.455630E 01	0.225768E 02	-0.289159E 02	-0.268080E 02	-0.163901E 02	-0.400464E 00
-0.101620E 02	0.253190E 02	0.338235E 02	0.283317E 02	0.140992E 02	-0.490518E 01
0.165482E 02	-0.350247E 02	-0.368418E 02	-0.277167E 02	-0.100702E 02	0.110470E 02
-0.221178E 02	0.294111E 02	0.380368E 02	0.252828E 02	0.456695E 01	-0.176655E 02
0.346908E 02	-0.415665E 02				

FOR THE WAVE NUMBER 4

T

U

V

W

AT LOCATION OF SEGMENT 12 AND THETA -0.

0.	0.	-0.	-0.
0.100000E-02	-0.238113E-07	C.	0.621056E-05
0.200000E-02	-0.193479E-C6	0.	0.504640E-04
0.300000E-02	-0.350350E-C6	0.	0.913797E-04
0.400000E-02	-0.395473E-C6	C.	0.103149E-03
0.500000E-02	-0.266770E-C6	0.	0.695801E-04
0.600000E-02	0.267257E-C7	-C.	-0.697070E-05
0.700000E-02	0.400990E-06	-0.	-0.104588E-03
0.800000E-02	0.723157E-C6	-0.	-0.188617E-03
0.900000E-02	0.858970E-C6	-C.	-0.224040E-03
0.100000E-01	0.724684E-C6	-0.	-0.189015E-03
0.110000E-01	0.324334E-C6	-C.	-0.345941E-04
0.120000E-01	-0.242152E-06	0.	0.631591E-04
0.130000E-01	-0.804727E-06	C.	0.209892E-03
0.140000E-01	-0.117588E-C5	C.	0.306699E-03
0.150000E-01	-0.121346E-C5	C.	0.316499E-03
0.160000E-01	-0.872552E-C6	C.	0.227583E-03
0.170000E-01	-0.228312E-C6	0.	0.595494E-04
0.180000E-01	0.539841E-06	-0.	-0.140804E-03
0.190000E-01	0.119935E-C5	-0.	-0.312820E-03
0.200000E-01	0.153611E-05	-0.	-0.400654E-03
0.210000E-01	0.142413E-05	-C.	-0.371448E-03
0.220000E-01	0.870697E-06	-0.	-0.227099E-03
0.230000E-01	0.212740E-C7	-0.	-0.554876E-05
0.240000E-01	-0.879093E-C6	0.	0.229239E-03
0.250000E-01	-0.155752E-C5	0.	0.406239E-03
0.260000E-01	-0.179682E-C5	C.	0.468653E-03
0.270000E-01	-0.150507E-05	0.	0.392550E-03
0.280000E-01	-0.748996E-06	0.	0.195356E-03
0.290000E-01	0.260579E-C6	-0.	-0.679654E-04
0.300000E-01	0.122809E-05	-C.	-0.320316E-03
0.310000E-01	0.186062E-05	-C.	-0.485296E-03
0.320000E-01	0.195716E-05	-0.	-0.510474E-03
0.330000E-01	0.147240E-05	-0.	-0.384039E-03
0.340000E-01	0.534963E-06	-C.	-0.139531E-03
0.350000E-01	-0.586851E-C6	0.	0.153065E-03
0.360000E-01	-0.156241E-05	C.	0.407515E-03
0.370000E-01	-0.209687E-05	0.	0.546913E-03
0.380000E-01	-0.202064E-05	0.	0.527032E-03
0.390000E-01	-0.134310E-C5	0.	0.350314E-03
0.400000E-01	-0.242611E-06	0.	0.632789E-04
0.410000E-01	0.938448E-06	-C.	-0.244770E-03
0.420000E-01	0.184289E-05	-0.	-0.480670E-03
0.430000E-01	0.220815E-05	-C.	-0.575938E-03

RESPONSES SUMMED UP WITH THE PREVIOUS MODES
AT LOCATION OF SEGMENT 12 AND THETA -0.

0.	0.	-0.	-0.
0.100000E-02	-0.238113E-07	0.	0.621056E-05
0.200000E-02	-0.193479E-06	0.	0.504640E-04
0.300000E-02	-0.350350E-06	0.	0.913797E-04
0.400000E-02	-0.395473E-06	0.	0.103149E-03
0.500000E-02	-0.266770E-06	0.	0.695901E-04
0.600000E-02	0.267257E-07	-0.	-0.697070E-05
0.700000E-02	0.400990E-06	-0.	-0.104588E-03
0.800000E-02	0.723157E-06	-0.	-0.188617E-02
0.900000E-02	0.858970E-06	-0.	-0.224040E-02
0.100000E-01	0.724684E-06	-0.	-0.189015E-03
0.110000E-01	0.324334E-06	-0.	-0.845941E-04
0.120000E-01	-0.242152E-06	0.	0.631591E-04
0.130000E-01	-0.804727E-06	0.	0.209892E-03
0.140000E-01	-0.117588E-05	0.	0.306699E-03
0.150000E-01	-0.121346E-05	0.	0.316499E-03
0.160000E-01	-0.872552E-06	0.	0.227543E-03
0.170000E-01	-0.228312E-06	0.	0.595494E-04
0.180000E-01	0.539841E-06	-0.	-0.140804E-03
0.190000E-01	0.119935E-05	-0.	-0.312820E-03
0.200000E-01	0.153611E-05	-0.	-0.400654E-03
0.210000E-01	0.142413E-05	-0.	-0.371448E-03
0.220000E-01	0.870697E-06	-0.	-0.227099E-03
0.230000E-01	0.212740E-07	-0.	-0.554876E-05
0.240000E-01	-0.879093E-06	0.	0.229289E-03
0.250000E-01	-0.155752E-05	0.	0.406239E-03
0.260000E-01	-0.179682E-05	0.	0.468653E-03
0.270000E-01	-0.150507E-05	0.	0.392560E-03
0.280000E-01	-0.748996E-06	0.	0.195356E-03
0.290000E-01	0.260579E-06	-0.	-0.679654E-04
0.300000E-01	0.122809E-05	-0.	-0.320316E-03
0.310000E-01	0.186062E-05	-0.	-0.485296E-03
0.320000E-01	0.195716E-05	-0.	-0.510474E-03
0.330000E-01	0.147240E-05	-0.	-0.384039E-03
0.340000E-01	0.534963E-06	-0.	-0.139531E-03
0.350000E-01	-0.586851E-06	0.	0.153065E-03
0.360000E-01	-0.156241E-05	0.	0.407515E-03
0.370000E-01	-0.209687E-05	0.	0.546913E-03
0.380000E-01	-0.202064E-05	0.	0.527032E-03
0.390000E-01	-0.134310E-05	0.	0.350314E-03
0.400000E-01	-0.242611E-06	0.	0.632789E-04
0.410000E-01	0.938448E-06	-0.	-0.244770E-03
0.420000E-01	0.184289E-05	-0.	-0.480670E-03
0.430000E-01	0.220815E-05	-0.	-0.575938E-03

INPUT U, V, W, AND BETA

0.161827E 00	-0.340274E 00	0.666679E 00	0.946573E 00
0.162036E 00	-0.300335E 00	0.601662E 00	0.732761E 00
0.162018E 00	-0.260442E 00	0.521205E 00	0.743098E 00
0.162007E 00	-0.220513E 00	0.441333E 00	0.743863E 00
0.161989E 00	-0.180558E 00	0.361390E 00	0.744251E 00
0.161964E 00	-0.140584E 00	0.281415E 00	0.744487E 00
0.161927E 00	-0.100598E 00	0.201422E 00	0.744588E 00
0.161880E 00	-0.606097E-01	0.121426E 00	0.744547E 00
0.161819E 00	-0.206263E-01	0.414426E-01	0.744351E 00
0.161744E 00	0.193432E-01	-0.385090E-01	0.744160E 00
0.161659E 00	0.592895E-01	-0.118511E 00	0.740505E 00
0.161608E 00	0.891996E-01	-0.178557E 00	0.752898E 00
0.161571E 00	0.119081E 00	-0.238818E 00	0.707998E 00
0.161373E 00	0.148970E 00	-0.282115E 00	0.381904E 00
0.161306E 00	0.186015E 00	-0.347808E 00	0.702705E 00
0.161179E 00	0.223010E 00	-0.423534E 00	0.688010E 00
0.161048E 00	0.259943E 00	-0.498720E 00	0.687372E 00
0.160913E 00	0.296805E 00	-0.573783E 00	0.686004E 00
0.160778E 00	0.333588E 00	-0.648691E 00	0.684535E 00
0.160648E 00	0.370286E 00	-0.723432E 00	0.682958E 00
0.160529E 00	0.406893E 00	-0.797998E 00	0.681346E 00
0.160430E 00	0.443406E 00	-0.872350E 00	0.678874E 00
0.160370E 00	0.479822E 00	-0.947450E 00	0.688331E 00
0.159982E 00	0.516293E 00	-0.100000E 01	0.100000E 01

** INTERMEDIATE PRINT OUT **

FUNCTION
0.866682E-17 0.707098E-17 0.540512E-17
0.113666E-17 0.659474E-18 0.418658E-18
0.204234E-17 0.186726E-17 0.259081E-17
0.719900E-17 0.862096E-17 0.1012C6E-16

0.398506E-17 0.279976E-17 0.185017E-17
0.414078E-18 0.916952E-18 0.138415E-17
0.355249E-17 0.464923E-17 0.586995E-17
0.116820E-16 0.133113E-16 -0.000000E-19

INTEGRATION = 0.18332E-14

VALUES OF Q
-0.000000E-19 -0.469783E 01 -0.400516E 02 -0.809710E 02 -0.113121E 03 -0.123092E 03
-0.107047E 03 -0.634651E 02 -0.165019E 01 -0.652268E 02 -0.122870E 03 -0.153609E 03
0.169234E 03 0.152177E C3 0.114852E 03 0.574352E 02 0.209439E 02 -0.162025E 02
-0.392012E 02 -0.483220E 02 -0.450561E 02 -0.446557E 02 -0.507164E 02 -0.675374E 02
-C.633302E 02 -0.791591E 02 -C.926049E 02 -0.976555E 02 -0.998145E 02 -0.124328E 03
-0.327798E 02 0.944988E 01 0.523503E 02 0.889981E 02 0.113946E 03 0.617775E 01
0.120223E 03 0.104257E 03 0.806336E 02 0.541774E 02 0.301859E 02 0.301859E 02

FOR THE WAVE NUMBER 2

T	U	V	W
AT LOCATION OF SEGMENT 12 AND THETA -0.			
0.	-0.	0.	0.
0.100000E-02	-0.183249E-06	0.	0.202469E-06
0.200000E-02	-0.156563E-05	0.	0.172984E-05
0.300000E-02	-0.316519E-05	0.	0.349716E-05
0.400000E-02	-0.442197E-05	0.	0.488575E-05
0.500000E-02	-0.484692E-05	0.	0.535528E-05
0.600000E-02	-0.418453E-05	0.	0.462341E-05
0.700000E-02	-0.248088E-05	0.	0.274108E-05
0.800000E-02	-0.645066E-07	0.	0.712722E-07
0.900000E-02	0.254975E-05	-0.	-0.281717E-05
0.100000E-01	0.480303E-05	-0.	-0.530679E-05
0.110000E-01	0.623920E-05	-0.	-0.689358E-05
0.120000E-01	0.661542E-05	-0.	-0.730926E-05
0.130000E-01	0.594867E-05	-0.	-0.657258E-05
0.140000E-01	0.448960E-05	-0.	-0.496048E-05
0.150000E-01	0.263607E-05	-0.	-0.291255E-05
0.160000E-01	0.814796E-06	-0.	-0.900254E-06
0.170000E-01	-0.633362E-06	0.	0.699791E-06
0.180000E-01	-0.153239E-05	0.	0.169311E-05
0.190000E-01	-0.189496E-05	0.	0.209371E-05
0.200000E-01	-0.188893E-05	0.	0.208704E-05
0.210000E-01	-0.176126E-05	0.	0.194599E-05
0.220000E-01	-0.174561E-05	0.	0.192870E-05
0.230000E-01	-0.198253E-05	0.	0.219046E-05
0.240000E-01	-0.247561E-05	0.	0.273525E-05
0.250000E-01	-0.309436E-05	0.	0.341841E-05
0.260000E-01	-0.361997E-05	0.	0.399964E-05
0.270000E-01	-0.381740E-05	0.	0.421778E-05
0.280000E-01	-0.351089E-05	0.	0.387912E-05
0.290000E-01	-0.264007E-05	0.	0.291696E-05
0.300000E-01	-0.128138E-05	0.	0.141577E-05
0.310000E-01	0.369400E-06	-0.	-0.408144E-06
0.320000E-01	0.204640E-05	-0.	-0.226103E-05
0.330000E-01	0.347898E-05	-0.	-0.384386E-05
0.340000E-01	0.445420E-05	-0.	-0.492137E-05
0.350000E-01	0.486004E-05	-0.	-0.536977E-05
0.360000E-01	0.469958E-05	-0.	-0.519248E-05
0.370000E-01	0.407547E-05	-0.	-0.450291E-05
0.380000E-01	0.315200E-05	-0.	-0.348259E-05
0.390000E-01	0.211782E-05	-0.	-0.233994E-05
0.400000E-01	0.117998E-05	-0.	-0.130374E-05
0.410000E-01	0.241491E-06	-0.	-0.266819E-06
0.420000E-01	-0.721480E-06	0.	0.797151E-06
0.430000E-01	-0.162457E-05	0.	0.179496E-05

RESPONSES SUMMED UP WITH THE PREVIOUS MODES
AT LOCATION OF SEGMENT 12 AND THETA -0.

0.	-0.	0.	0.
C.100000E-02	-0.415181E-05	0.	0.611102E-04
0.200000E-02	-0.322999E-04	0.	0.464978E-03
0.300000E-02	-0.317425E-04	0.	0.548518E-03
0.400000E-02	-0.298135E-05	0.	0.311864E-03
0.500000E-02	0.187780E-04	0.	0.421008E-04
C.600000E-02	0.963398E-05	0.	-0.426495E-04
0.700000E-02	-0.120270E-04	C.	-0.194055E-04
0.800000E-02	-0.135489E-04	0.	-0.149841E-03
0.900000E-02	0.100376E-04	-0.	-0.427159E-03
C.100000E-01	0.320723E-04	-0.	-0.528054E-03
0.110000E-01	0.282822E-04	-0.	-0.262617E-03
0.120000E-01	0.663380E-05	-C.	0.209933E-03
0.130000E-01	-0.492269E-05	-0.	0.522500E-03
0.140000E-01	0.640699E-05	-0.	0.468305E-03
0.150000E-01	0.230399E-04	-0.	0.199137E-03
0.160000E-01	0.217552E-04	-0.	-0.597909E-04
0.170000E-01	0.457778E-05	C.	-0.227191E-03
0.180000E-01	-0.742641E-05	0.	-0.385899E-03
0.190000E-01	-0.125831E-05	C.	-0.604851E-03
0.200000E-01	0.131694E-04	0.	-0.725534E-03
0.210000E-01	0.153612E-04	0.	-0.545714E-03
0.220000E-01	0.103850E-05	C.	-0.119708E-03
0.230000E-01	-0.134434E-04	0.	0.290490E-03
0.240000E-01	-0.129886E-04	0.	0.480064E-03
0.250000E-01	-0.199843E-05	0.	0.407981E-03
C.260000E-01	0.281475E-05	0.	0.227362E-03
0.270000E-01	-0.504779E-05	0.	0.992351E-04
0.280000E-01	-0.154600E-04	0.	-0.358341E-04
0.290000E-01	-0.155218E-04	0.	-0.239582E-03
0.300000E-01	-0.497434E-05	0.	-0.416731E-03
0.310000E-01	0.303650E-05	-0.	-0.449529E-03
0.320000E-01	-0.849160E-06	-C.	-0.290859E-03
0.330000E-01	-0.103444E-04	-0.	0.760680E-05
0.340000E-01	-0.131029E-04	-0.	0.227525E-03
0.350000E-01	-0.679363E-05	-0.	0.312369E-03
C.360000E-01	-0.584448E-06	-0.	0.340206E-03
0.370000E-01	-0.267319E-05	-C.	0.384811E-03
0.380000E-01	-0.111832E-04	-0.	0.432683E-03
0.390000E-01	-0.600869E-05	-0.	0.283047E-03
C.400000E-01	-0.614722E-05	-0.	0.147601E-03
C.410000E-01	-0.595933E-05	-0.	0.184913E-04
0.420000E-01	-0.630696E-05	C.	-0.936695E-04
0.430000E-01	-0.692474E-05	0.	-0.175731E-03

NATURAL FREQUENCY AND MODES OF MASS ATTACHED CYLINDRICAL SHELL

KEY PUNCH FORM - GENERAL PURPOSE

Form 20-708 (R.7-63)

JOB TITLE				NATURAL FREQUENCY AND MODE OF MASS ATTACHED CYLINDER				ENGINEER				PAGE	
DPWA SERIAL NO.	PRI.	DED. JOB NO.	DASH	FOR ORGN. NO.		ANALYST							OF
1	2	3	4	5	6	7	8	9	0	1	2	3	4
2	3	4	5	6	7	8	9	0	1	2	3	4	5
3	4	5	6	7	8	9	0	1	2	3	4	5	6
4	5	6	7	8	9	0	1	2	3	4	5	6	7
5	6	7	8	9	0	1	2	3	4	5	6	7	8
6	7	8	9	0	1	2	3	4	5	6	7	8	9
7	8	9	0	1	2	3	4	5	6	7	8	9	0
8	9	0	1	2	3	4	5	6	7	8	9	0	1
9	0	1	2	3	4	5	6	7	8	9	0	1	2
0	1	2	3	4	5	6	7	8	9	0	1	2	3
1	2	3	4	5	6	7	8	9	0	1	2	3	4
2	3	4	5	6	7	8	9	0	1	2	3	4	5
3	4	5	6	7	8	9	0	1	2	3	4	5	6
4	5	6	7	8	9	0	1	2	3	4	5	6	7
5	6	7	8	9	0	1	2	3	4	5	6	7	8
6	7	8	9	0	1	2	3	4	5	6	7	8	9
7	8	9	0	1	2	3	4	5	6	7	8	9	0
8	9	0	1	2	3	4	5	6	7	8	9	0	1
9	0	1	2	3	4	5	6	7	8	9	0	1	2
0	1	2	3	4	5	6	7	8	9	0	1	2	3
1	2	3	4	5	6	7	8	9	0	1	2	3	4
2	3	4	5	6	7	8	9	0	1	2	3	4	5
3	4	5	6	7	8	9	0	1	2	3	4	5	6
4	5	6	7	8	9	0	1	2	3	4	5	6	7
5	6	7	8	9	0	1	2	3	4	5	6	7	8
6	7	8	9	0	1	2	3	4	5	6	7	8	9
7	8	9	0	1	2	3	4	5	6	7	8	9	0
8	9	0	1	2	3	4	5	6	7	8	9	0	1
9	0	1	2	3	4	5	6	7	8	9	0	1	2

A	C	H	RHO	XL	THL								
THI	AMASS	INC	N MODE										
IMOD	I INC												
N (1)	K (1)		QM (1)										
N (2)	K (2)		OM (2)										
—	—	—	—										
N (N MODE)	K (N MODE)		OM (N MODE)										

IBJOB
60,MAP
EXECUTE
IBJOB
STRETCH MAIN


```

NGUES =0
NTAPEI =6
CALL MITERS (TM,GUESS,NGUES,N1,KMODE,MAXR,NC,EPSP,EPDP,NAKSR,
1 NAKDR,NITRSP,NITRDP,RSP,RDP,IR,TEMP,VECTR,EIGV,ITRS,NTAPE,
2 NTAPEI)
176 IF(LEIG.0T.1) GO TO 176
      WRITE (6,155)
155 FORMAT (1H151)REFLECTIONS OF CYLINDRICAL SHELL WITH MASS ATTACHED ) IMP2
      WRITE (6,150)
150 FORMAT (17W ** INPUT DATA ** /) IMP2
      WRITE (6,160)
160 FORMAT (6.165) A,CL,H,RHO,XL,THL,AMASS,NMODE IMP2
      WRITE (6,165) A,CL,H,RHO,XL,THL,AMASS,NMODE IMP2
165 FORMAT (7E15.5,8X,12) IMP2
160 FORMAT (9X,1H1 10X,6HLENSTH13X,1HHL12X,3HRHO 14X,2HXL12X,3HTHL11X, IMP2
1   4HMASS 12X,5HMMODE /) IMP2
      WRITE (6,170)
170 WRITE (6,175) (N(I),K(I),OM(I),I=1,NMODE) IMP2
      FORMAT (//18H INPUT N, K, OMEGA /) IMP2
175 FORMAT (3X,2I4,E14.5,3X,2I4,E14.5,3X,2I4,E14.5,3X,2I4,E14.5) IMP2
NSTA =INIC
STA =NSTA
KSTA =NSTA+1
176 CRIT =CCL/A IMP2
      J2 =1
      PAL =PI*A/CL IMP2
      XINC =CL/(A*STA) IMP2
      X =0. IMP2
      THETA =THL IMP2
      KST =1
      KY=0 IMP2
179 CONTINUE IMP2
      IF(J2.GE.2) GO TO 166
      WRITE (6,200) THETA IMP2
      DO 62 KJ =1,KMODE IMP2
62 EIGV(KJ) =SORT(OM2/EIGV(KJ))/(2.*PI) IMP2
      WRITE (6,64)(EIGV(KJ)*KJ=1,KMODE) IMP2
64 FORMAT (/16H ** FREQ. IN CPS E14.5 /16X,E14.5 /) IMP2
      GO TO 168 IMP2
168 IF(KST.EQ.1) WRITE(6,276) X IMP2
      IF(KST.EQ.1) WRITE(6,277) X IMP2

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277 FORMAT(6X,EM THETA 16X,1HJ 17X,1HW 16X,1HW / )
148 CONTINUE
      XS(KST) =THETA
      WRITE(6,200) SO TO 180
200 FORMAT(7/78H ** DEFLECTIONS AT THETA =E14.5 /)
180 CONTINUE
      IF(JN.EQ.2) XS(KST) =X
      PALX =XS*ALGX
      COS(JN) =COS(TN)*SIN(PALX)
      SIN(JN) =SIN(TN)*SIN(PALX)*COS(JN)
      X(KST,JN) =XS(KST,JN)*COS(PALX)+B(JN)
      Y(KST,JN) =XS(KST,JN)*VECTR(JN,JN)*CSK(JN)
      Z(KST,JN) =XS(KST,JN)*VECTR(JN,JN)*CSN(JN)
      U(KST,JN) =X(KST,JN)+VECTR(JN,JN)*CSK(JN)
      V(KST,JN) =Y(KST,JN)+VECTR(JN,JN)*CSN(JN)
      W(KST,JN) =Z(KST,JN)+VECTR(JN,JN)*C(JN)
      DO 190 JN=1,LMODE
      U(KST,JN)=0.00.
      V(KST,JN)=0.00.
      W(KST,JN)=0.00.
190   CONTINUE
      195 CONTINUE
      196 CONTINUE
      197 IF(ABS(W(KST,KSTA)) .GE. .016E-6) GO TO 191
      191 192 KST=KST+1
      193 192 IF(X(KST,KSTA) .GT. 1.0) X(KST,KSTA)=X(KST,KSTA)
      194 DO 193 KX=1,KST
          WIXX,ILUP,IWIKX,ILUP,IIGN
          VIIX,ILUP,VIWIKX,ILUP,BIGN
          S193=ABS(W(KST,KSTA))
          195 196 CONTINUE
          197 198 ZABS(W(KST,KSTA))

```

```

      WRITE (6,212) ILUP
      WRITE (6,210) (XS(I),U(I,ILUP),V(I,ILUP),W(I,ILUP)),I=1,KST)
      ILUP = ILUP+1
      IF (ILUP.LE.KMODE) GO TO 217
      212 FORMAT(1W 3X,15H * MODE NUMBER 12,2H */)
      213 FORMAT(1W 3X,15H * MODE NUMBER 12,2H */)
      214 IF(IJG.EQ.3) GO TO 500
      IF(IJG.EQ.2) GO TO 300
      205 FORMAT(1ZX,2W X 16X,1W 17X,1W 16X,1W /)
      218 FORMAT(1E18.7)
      X 2X+XINC
      KST = KST+1
      IF(KST.LE.KSTA) GO TO 180
      KST = 1
      X 2X
      THETA = THI
      X 2X
      180 X 2X
      X 2X
      KST = KST+1
      THETA = THAP1/STA
      KST = KST+1
      IF(KST.EQ.KSTA) GO TO 350
      J2 = J2+1
      350 GO TO 179
      300 X 2X
      X 2X
      LEIG = LEIG+1
      LMODE = LMODE+1INC
      KST = 1
      IF(LMODE.LE.MODE) GO TO 625
      276 FORMAT(1//35H CIRCUMFERENTIAL DEFLECTIONS AT X =E14.6)
      STOP
      END

```

DEFLECTIONS OF CYLINDRICAL SHELL WITH MASS ATTACHED

** INPUT DATA **

A	LNGTH	H	RHO	XL	THL	MASS	NODE
0.19500E 02	0.53400E 02	0.50000F-01	0.25910E-03	0.13700E 01	0.	0.25900E-02	50

INPUT N, K, OMEGA

N	K	OMEGA	N	K	OMEGA	N	K	OMEGA	N	K	OMEGA
6	1	0.46158E 03	7	1	0.47393E 03	5	1	0.54938E 03	8	1	0.55029E 03
9	1	0.66553E 03	4	1	0.78783E 03	10	1	0.80688E 03	11	1	0.96875E 03
12	1	0.11487E 04	3	1	0.13065E 04	13	1	0.13456E 04	11	3	0.13875E 04
10	3	0.13950E 04	12	3	0.14536E 04	9	3	0.14942E 04	14	1	0.15590E 04
13	3	0.15762E 04	8	3	0.17036E 04	14	3	0.17144E 04	15	1	0.17886E 04
15	1	0.19397E 04	16	1	0.20342E 04	7	3	0.20463E 04	16	3	0.21644E 04
17	1	0.22958E 04	13	5	0.23058E 04	14	5	0.23301E 04	12	5	0.23588E 04
17	3	0.24127E 04	15	5	0.24199E 04	11	5	0.24999E 04	2	1	0.25315E 04
6	2	0.25584E 04	16	5	0.25638E 04	18	1	0.25734E 04	18	3	0.26809E 04
10	5	0.27391E 04	17	5	0.27521E 04	19	1	0.28668E 04	19	3	0.29672E 04
18	5	0.29768E 04	9	5	0.30864E 04	20	1	0.31762E 04	15	7	0.32290E 04
19	5	0.32320E 04	14	7	0.32628E 04	16	7	0.32646E 04	20	3	0.32727E 04
5	3	0.32993E 04	17	7	0.33616E 04						

** EIGENMATRIX **

ROW NO.	1	2	3	4	5
ROW NO.	1				
0.111P4E 01	0.11877E 00				
-0.11877E 00	0.11877E 00	-0.11877E 00	-0.11877E 00	-0.11877E 00	-0.11877E 00
-0.11877E 00	-0.11877E 00	0.11877E 00	-0.11877E 00	0.11877E 00	-0.11877E 00
0.11877E 00	0.11876E 00				
-0.11877E 00	0.11876E 00	0.11877E 00	-0.11877E 00	0.11876E 00	-0.11877E 00
0.11876E 00	0.11876E 00	0.11877E 00	0.11876E 00	0.11876E 00	-0.11876E 00
-0.11877E 00	-0.11876E 00	-0.11876E 00	-0.11877E 00	-0.11876E 00	-0.11877E 00
ROW NO.	2				
0.11354E 00	0.10621E 01	0.11354E 00	0.11354E 00	0.11354E 00	0.11354E 00
0.11354E 00	0.11354E 00	0.11354E 00	-0.11353E 00	-0.11353E 00	-0.11353E 00
-0.11353E 00	-0.11353E 00	-0.11353E 00	0.11354E 00	-0.11353E 00	0.11354E 00
0.11354E 00	0.11353E 00	0.11353E 00	0.11353E 00	-0.11353E 00	-0.11353E 00
-0.11353E 00	0.11353E 00	0.11354E 00	-0.11353E 00	0.11353E 00	-0.11354E 00
0.11353E 00	-0.11353E 00	-0.11354E 00	-0.11353E 00	-0.11353E 00	-0.11353E 00
-0.11353E 00					
ROW NO.	3				
0.82755E-01	0.82755E-01	0.79866E 00	0.82755E-01	0.82755E-01	0.82755E-01
0.82755E-01	0.82755E-01	0.82755E-01	-0.82755E-01	-0.82755E-01	-0.82755E-01
-0.82755E-01	-0.82755E-01	-0.82755E-01	0.82755E-01	0.82755E-01	-0.82755E-01
0.82755E-01	0.82754E-01	0.82754E-01	0.82754E-01	-0.82755E-01	0.82754E-01
-0.82755E-01	0.82754E-01	0.82755E-01	-0.82755E-01	0.82754E-01	0.82755E-01
0.82754E-01	0.82754E-01	0.82755E-01	-0.82755E-01	0.82754E-01	-0.82755E-01
-0.82755E-01	0.82753E-01	0.82753E-01	-0.82753E-01	0.82753E-01	-0.82753E-01
0.82753E-01	-0.82753E-01	-0.82753E-01	-0.82753E-01	-0.82753E-01	-0.82753E-01
ROW NO.	4				
0.84635E-01	0.84635E-01	0.84635E-01	0.78922E 00	0.84635E-01	0.84635E-01
0.84635E-01	0.84635E-01	0.84635E-01	-0.84634E-01	-0.84634E-01	-0.84634E-01
-0.84634E-01	-0.84634E-01	-0.84634E-01	0.84635E-01	-0.84634E-01	0.84635E-01
0.84634E-01	0.84634E-01	0.84634E-01	0.84634E-01	-0.84634E-01	-0.84634E-01
-0.84635E-01	0.84634E-01	0.84635E-01	-0.84634E-01	0.84634E-01	0.84635E-01
0.84634E-01	-0.84634E-01	-0.84635E-01	-0.84634E-01	-0.84633E-01	-0.84634E-01
-0.84634E-01	-0.84633E-01	-0.84635E-01	-0.84634E-01	-0.84633E-01	-0.84634E-01
ROW NO.	5				
0.58059E-01	0.58059E-01	0.58059E-01	0.58059E-01	0.58059E-01	0.58059E-01
0.58059E-01	0.58059E-01	0.58059E-01	-0.58059E-01	-0.58059E-01	-0.58059E-01
-0.58059E-01	-0.58059E-01	-0.58059E-01	0.58059E-01	0.58059E-01	-0.58059E-01
0.58059E-01	0.58059E-01	0.58059E-01	0.58059E-01	-0.58059E-01	0.58059E-01
-0.58059E-01	0.58059E-01	0.58059E-01	-0.58059E-01	0.58059E-01	-0.58059E-01
0.58059E-01	-0.58059E-01	-0.58059E-01	-0.58059E-01	-0.58058E-01	-0.58059E-01
-0.58059E-01	-0.58058E-01	-0.58059E-01	-0.58059E-01	-0.58058E-01	-0.58059E-01

SOLUTION FOR EIGENMATRIX OF SORRY 50

MODE	EIGENVALUE	ITERATIONS	S.P.	D.P.	AITKENS S.P.	D.P.
1	0.14632137E 01	14	0	3	0	0
2	0.97404109E 00	60	15	0	0	0

EIGENVECTORS

	COLUMN 1	COLUMN 2	COLUMN
1	0.10000000E 01	0.10000000E 01	
2	0.85982065E 00	-0.97101007E 00	
3	0.42622349E 00	-0.67465298E-01	
4	0.43191709E 00	-0.67748424E-01	
5	0.23050754E 00	-0.25732379E-01	
6	0.13593487E 00	-0.13472417E-01	
7	0.17561053E 00	-0.13370225E-01	
8	0.85974974E-01	-0.79274047E-02	
9	0.56730844E-01	-0.52746073E-02	
10	0.38199861E-01	-0.33061244E-02	
11	0.40627330E-01	-0.35396921E-02	
12	-0.39158454E-01	0.46642019E-02	
13	-0.38214273E-01	0.33569345E-02	
14	-0.34644440E-01	0.30142553E-02	
15	-0.32893230E-01	0.28734552E-02	
16	0.29442091E-01	-0.25232467E-02	
17	-0.29535093E-01	0.25686657E-02	
18	-0.25102125E-01	0.21883519E-02	
19	-0.23944507E-01	0.20695918E-02	
20	0.22725602E-01	-0.19658848E-02	
21	-0.19025182E-01	0.16313144E-02	
22	0.17306819E-01	-0.14850397E-02	
23	-0.15375293E-01	0.13850636E-02	
24	-0.14635256E-01	0.12244624E-02	
25	0.13435072E-01	-0.11446725E-02	
26	0.13655878E-01	-0.11818676E-02	
27	0.15621589E-01	-0.14478206E-02	
28	0.15277757E-01	-0.14169677E-02	
29	-0.86642038E-02	0.50305144E-03	
30	0.15803813E-01	-0.15128574E-02	
31	0.11706947E-01	-0.10190189E-02	
32	0.89413845E-02	-0.77101984E-03	
33	-0.10728772E-01	0.92195591E-03	
34	0.10784300E-01	-0.91996184E-03	
35	0.11154525E-01	-0.97227283E-03	
36	-0.99543558E-02	0.83978637E-03	
37	0.94495283E-02	-0.83424302E-03	
38	0.93360423E-02	-0.79516845E-03	
39	0.10823664E-01	-0.10232199E-02	
40	-0.81609217E-02	0.70091221E-03	
41	0.80926457E-02	-0.69333031E-03	
42	0.73963042E-02	-0.63154771E-03	
43	0.72938953E-02	-0.63426694E-03	
44	-0.63016087E-02	0.58036949E-03	
45	0.70082628E-02	-0.60794386E-03	
46	-0.66554364E-02	0.56781142E-03	
47	-0.68277984E-02	0.59077388E-03	
48	-0.66335159E-02	0.56591677E-03	
49	-0.64572991E-02	0.55881572E-03	
50	-0.62752842E-02	0.53515856E-03	

CHECK EIGENVALUES AND EIGENVECTORS

	0.14632137E 01	0.97404118F 00	
	COLUMN 1	COLUMN 2	COLUMN
1	0.10000000E 01	0.10000000E 01	
2	0.85882071E 00	-0.97100934E 00	
3	0.42622355F 00	-0.67464978E-01	
4	0.43191713E 00	-0.67748056E-01	
5	0.23050755E 00	-0.25732264E-01	
6	0.13593488F 00	-0.13472338E-01	
7	0.13561059E 00	-0.13330164F-01	
8	0.85974979E-01	-0.79273517E-02	
9	0.58739849E-01	-0.52745784E-02	
10	0.38199863E-01	-0.33061051E-02	
11	0.40627333F-01	-0.35396767E-02	
12	-0.39158456F-01	0.34641835E-02	
13	-0.38214281E-01	0.33569197F-02	
14	-0.34644443E-01	0.30142359F-02	
15	-0.32893232E-01	0.28734409E-02	
16	0.29442093E-01	-0.25232298F-02	
17	-0.29535094E-01	0.25686522F-02	
18	-0.25102126F-01	0.21883388F-02	
19	-0.23944509E-01	0.20695803E-02	
20	0.22725603E-01	-0.19658701F-02	
21	-0.19025183E-01	0.16313064F-02	
22	0.17306820E-01	-0.14850294E-02	
23	-0.16375300E-01	0.13850569F-02	
24	-0.14635257E-01	0.12244538E-02	
25	0.13435074E-01	-0.11446651E-02	
26	0.13656879F-01	-0.11818598E-02	
27	0.15621590E-01	-0.14476129F-02	
28	0.15277758E-01	-0.14169592E-02	
29	-0.80664216E-02	0.50304671E-03	
30	0.15803814E-01	-0.15128495F-02	
31	0.11706948E-01	-0.10190128E-02	
32	0.89413851E-02	-0.77101562E-03	
33	-0.10728773E-01	0.92195057F-03	
34	0.10784301E-01	-0.91995630F-03	
35	0.11154526E-01	-0.97226799E-03	
36	-0.98543565F-02	0.83978193F-03	
37	0.96405295E-02	-0.83423829E-03	
38	0.93360430E-02	-0.79516370E-03	
39	0.10823665E-01	-0.10232152F-02	
40	-0.81609221F-02	0.70090812F-03	
41	0.80926461E-02	-0.69332590F-03	
42	0.73963046E-02	-0.63154398E-03	
43	0.72938961F-02	-0.63426283F-03	
44	-0.68016092E-02	0.58036559E-03	
45	0.70082632E-02	-0.60793958F-03	
46	-0.66554368E-02	0.56780809F-03	
47	-0.68277990E-02	0.59076954E-03	
48	-0.66335163E-02	0.56591348E-03	
49	-0.64573996E-02	0.55881156E-03	
50	-0.62752846E-02	0.53515533F-03	

** DEFLECTIONS AT THETA = 0.

** FREQ. IN CPS 0.60731E 02
0.74435E 02

X	U	V	W
* MODE NUMBER 1 *			
0.	0.2156827E-01	-0.	-0.
0.1521368E 00	0.2132365E-01	-0.	0.1306088E 00
0.3042735E 00	0.2059060E-01	-0.	0.2585745E 00
0.4564103E 00	0.1941309E-01	0.	0.3798555E 00
0.6085470E 00	0.1789512E-01	0.	0.4938636E 00
0.7606838E 00	0.1605713E-01	0.	0.6082210E 00
0.9128205F 00	0.1365120E-01	-0.	0.7319357E 00
0.1064957E 01	0.1023243E-01	-0.	0.8589744E 00
0.1217094E 01	0.5562546E-02	0.	0.9605347E 00
0.1369231E 01	0.1459937E-08	0.	0.1000000E 01
0.1521367E 01	-0.5562544E-02	0.	0.9605347E 00
0.1673504E 01	-0.1023243E-01	-0.	0.8589745E 00
0.1825641F 01	-0.1365120E-01	-0.	0.7319359E 00
0.1977778E 01	-0.1605713E-01	0.	0.6082211E 00
0.2129914E 01	-0.1789512E-01	0.	0.4938637E 00
0.2282051F 01	-0.1941310E-01	0.	0.3798556E 00
0.2434188E 01	-0.2059060E-01	-0.	0.2585746E 00
0.2586325E 01	-0.2132365E-01	-0.	0.1306089E 00
0.2738461E 01	-0.2156827E-01	-0.	0.1169375E-06
* MODE NUMBER 2 *			
0.	0.1131616E-01	0.	0.
0.1521368E 00	0.1102895E-01	0.	-0.1096557E 00
0.3042735F 00	0.1017769E-01	0.	-0.2171141E 00
0.4564103E 00	0.8737139E-02	-0.	-0.3186764E 00
0.6085470E 00	0.6634509E-02	-0.	-0.4165223E 00
0.7606838F 00	0.3974061E-02	-0.	-0.5253181E 00
0.9128205F 00	0.1310564E-02	0.	-0.6611493E 00
0.1064957E 01	-0.4752302E-03	0.	-0.8162974E 00
0.1217094E 01	-0.8043363E-03	-0.	-0.9476982E 00
0.1369231E 01	-0.2073886E-09	0.	-0.1000000E 01
0.1521367E 01	0.8043355E-03	-0.	-0.9476982E 00
0.1673504F 01	0.4752310E-03	0.	-0.8162975E 00
0.1825641F 01	-0.1310562E-02	0.	-0.6611494E 00
0.1977778F 01	-0.3974059E-02	-0.	-0.5253182E 00
0.2129914E 01	-0.6634506E-02	-0.	-0.4165223E 00
0.2282051F 01	-0.8737135E-02	-0.	-0.3186764E 00
0.2434188E 01	-0.1017769E-01	0.	-0.2171142E 00
0.2586325E 01	-0.1102895E-01	0.	-0.1096558E 00
0.2738461E 01	-0.1131616E-01	0.	-0.1007634E-06

CIRCUMFERENTIAL DEFLECTIONS AT X = 0.137000E 01
 THETA U V W

* MODE NUMBER	1 *		
-0.3141593E 01	-0.7887064E-06	-0.4359693E-08	0.2674892E-01
-0.2967060E 01	-0.4923795E-06	0.4152535E-02	0.1792333E-01
-0.2792527E 01	0.3222766E-06	0.5623501E-02	-0.3730739E-02
-0.2617994E 01	0.7047845E-06	0.2369230E-02	-0.3198064E-01
-0.2443461E 01	0.1075790E-05	-0.4953857E-02	-0.4568009E-01
-0.2268928E 01	-0.6605414E-06	-0.9985683E-02	-0.1385671E-02
-0.2094395E 01	-0.2028895E-05	-0.4200095E-02	0.6532119E-01
-0.1919862E 01	-0.1449150E-05	0.1054190E-01	0.8595974E-01
-0.1745329E 01	0.1100597E-05	0.1928176E-01	-0.1167383E-03
-0.1570796E 01	0.3867311E-05	0.7589414E-02	-0.1344260E 00
-0.1396263E 01	0.2599485E-05	-0.2175842E-01	-0.1610198E 00
-0.1221730E 01	-0.2997460E-05	-0.3610435E-01	0.2158377E-01
-0.1047197E 01	-0.6821299E-05	-0.9338764E-02	0.2745675E 00
-0.8726645E 00	-0.4464778E-05	0.4606423E-01	0.2907929E 00
-0.6981315E 00	0.5478416E-05	0.7053747E-01	-0.6636030E-01
-0.5235986E 00	0.1339179E-04	0.1485559E-01	-0.5253977E 00
-0.3490657E 00	0.1139171E-04	-0.8478165E-01	-0.5114933E 00
-0.1745328E 00	-0.6128298E-05	-0.1235744E 00	0.2120690E 00
0.1639128E-06	-0.2899005E-04	0.1652384E-06	0.1000000E 01

* MODE NUMBER	2 *		
-0.3141593E 01	-0.2320664E-04	-0.4980232E-07	0.1000000E 01
-0.2967060E 01	-0.1032787E-04	0.1412406E 00	0.4335212E 00
-0.2792527E 01	0.1372274E-04	0.1232216E 00	-0.6123379E 00
-0.2617994E 01	0.2208688E-04	-0.3026290E-01	-0.9423715E 00
-0.2443461E 01	0.6021088E-05	-0.1433687E 00	-0.2111234E 00
-0.2268928E 01	-0.1536175E-04	-0.9344190E-01	0.7060234E 00
-0.2094395E 01	-0.1865430E-04	0.5311170E-01	0.7781323E 00
-0.1919862E 01	-0.1954501E-05	0.1282026E 00	-0.2837560F-02
-0.1745329E 01	0.1482582E-04	0.5832441E-01	-0.6900052E 00
-0.1570796E 01	0.1381539E-04	-0.6317139E-01	-0.5413830E 00
-0.1396263E 01	-0.1095975E-05	-0.9845424E-01	0.1619371E 00
-0.1221730E 01	-0.1189993E-04	-0.2544019E-01	0.5607061E 00
-0.1047197E 01	-0.8516657E-05	0.5684808E-01	0.2778478E 00
-0.8726645E 00	0.2296305E-05	0.5915394E-01	-0.2280920E 00
-0.6981315E 00	0.7320015E-05	0.1175383E-02	-0.3416823E 00
-0.5235986E 00	0.3628047E-05	-0.3558204E-01	-0.4337646E-01
-0.3490657E 00	-0.2012483E-05	-0.1921754E-01	0.1841333E 00
-0.1745328E 00	-0.2592258E-05	0.8002378E-02	0.6921634E-01
0.1639128E-06	0.6055350E-06	-0.1910805E-07	-0.1166164E 00

JOB 5587 COMP/ASSMBL TIME 0.006 HRS., EXECUTION TIME 0.024 HRS.

APPENDIX C

**INPUT DATA FORMAT, PROGRAM LISTING,
AND TYPICAL OUTPUT DATA OF THE
ACOUSTIC RESPONSE COMPUTER PROGRAM**

DATA INPUT FORMAT

The program predicts the power spectral density of the acoustic response of a shell structure. The input data to the program include the shell modal data in the form of the generalized masses and the generalized forces, the acoustic input power spectrum, and the frequencies at which the response spectral density is to be computed. If the responses at more than one station are sought, an identical number of sets of W data [$= \bar{w}(s) \cos n\theta$] is to be provided to the program. Responses can be sought for different values of damping constant "GAMMA" by providing a number of input data sets with the desired values of "GAMMA". The following is a table defining the input data. The balance of the Appendix consists of the input format, computer program listing, and typical output data. The detail analysis of the program is presented in Section IV of the subject report.

DEFINITIONS

GAMMA	Damping constant, (Non-dimensional)
L	Total number of stations in the frequency domain where spectral density is computed, (ND)
N	Number of natural modes. (ND)
NSTAT	Number of sets of deflections w, (ND)
OMNAT	Natural frequency (cps)
B	Generalized force (lb.)
AMAS	Generalized mass (lb. in. sec ²)
OMEGA	Forcing frequency (cps)
PHIP	Input spectral density (lb ² /in ⁴ /cps)
W	$= \bar{w}(s) \cos n\theta$

KEY PUNCH FORM - GENERAL PURPOSE

FORM 20-708 (R.7-63)

JOB TITLE	Acoustic Response Program				ENGINEER	PAGE 1 OF 1
	DPWA SERIAL NO.	PRE.	DPD JOB NO.	DASH		
GAMMA	L	N	*	* NSTAT		
OMNAT (1)	OMNAT (2)				OMNAT (N)	
B (1)	B (2)				B (N)	
AMAS (1)	AMAS (2)				AMAS (N)	
OMEGA (1)	OMEGA (2)				OMEGA (L)	
PHIP (1)	PHIP (2)				PHIP (L)	
W (1)	W (2)				W (N)	*

```

      EXECUTE      1BJ08
      SISJOB YAHANE  BO,MAP
      SIGHTIC MAIN
      C   ACUSTIC RESPONSE PROGRAM

      C   N = TOTAL NUMBER OF NATURAL MODES
      C   L = TOTAL NUMBER OF STATIONS IN THE FREQUENCY DOMAIN WHERE PHIC ARE
      C   COMPUTED
      C   PHIP = INPUT SPECTRAL DENSITY
      C   GAMMA = DAMPING COEFFICIENT
      C   AMAS = GENERALIZED MASS
      C   OMAT = NATURAL FREQUENCY

      DIMENSION OMAT(10),OMEGA(45),AMAS(10),B(10),BIGC(45,10,10),
      1  BIGO(45,10,10),APACT(45,10),M(10),PHIP(45),BIGA(45,10),PHIG(45)

      NGAM = 1
      3  READ (15,10) GAMMA,L,N,MSTAT
      WRITE (6,210)
      210 FORMAT(1BN1* INPUT DATA 2N1*
      INPUT 15,215) GAMMA,L,N,MSTAT
      215 FORMAT(1BN1* GAMMA =E14.6,10H
      1S =12,19H NO. OF STAT. =12,/1
      10 FORMAT(1E12.8,313)
      15 FORMAT(1E12.8)
      READ (15,15) OMAT(I),I=1,M)
      READ (15,15) B(I),I=1,M)
      READ (15,15) (AMAS(I),I=1,M)
      READ (15,15) (OMEGA(I),I=1,L)
      IF(NGAM.GT.1) GO TO 242
      WRITE (6,225) (B(I),I=1,M)
      225 FORMAT(10W ** DS ** /6E18.6/6E18.6/6E18.6)
      WRITE (6,230) (AMAS(I),I=1,M)
      230 FORMAT(19H GENERALIZED MASS /6E18.6/6E18.6/6E18.6)
      WRITE (6,220) (OMAT(I),I=1,M)
      220 FORMAT(19H NATURAL FREQUENCY /6E18.6/6E18.6/6E18.6)
      WRITE (6,240) (OMEGA(I),I=1,L)
      240 FORMAT(114H FORCING FREQ /6E18.6/6E18.6/6E18.6/6E18.6/6E18.6)
      1/6E18.6/6E18.6)
      GBAR =306.064

      C FIRSTLY TO COMPUTE BIGC AND BIGD
      C 242 CONTINUE

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```

      GAM2 =GAMMA+GAMMA
      DO 100 J=1,L
      DO 100 J=1,N
      DO 100 K=1,N
      RDMK =DMAT(K)/DMEGA(1)
      RDMK2 =RDMK*DMEKA
      RDMJ =DMAT(J)/DMEGA(1)
      RDMJ2 =RDMJ*DMEKA
      BIGC(1,J,K) = (RDMK2-1.)* (RDMJ2-1.)*GAM2*DROMK*DROMJ
      S160(1,J,K) =2.*GAMMA*(RDMK*(RDMJ2-1.)*RDMJ*(RDMK2-1.))
      IF(J,GT,1) GO TO 100
      RDM12 = (RDMK2-1.)*seq2
      RDMGAM=4.*GAM2*DROMK2
      AFAC(1,J,K) =S1(K)/(GAMAS(K)*CBAR*(RDM12*RDMGAM))
      CONTINUE
      ISTAT =1
      100
      READ (5,19) (PHIP(11),11=1,L)
      READ (5,19) (W(12),12=1,N)
      IF(INGAM,GT,1) GO TO 175
      WRITE (6,131)
      WRITE (6,132) (PHIP(11),11=1,L)
      WRITE (6,133)
      WRITE (6,132) (W(12),12=1,N)
      133 FORMAT (/22H as VECOS(N,THETA) ** /)
      131 FORMAT (//11H ** PHIP ** /)
      132 FORMAT (6E10.6)
      175 CONTINUE
      DO 120 I=1,L
      DO 120 K=1,N
      120 BIGA(I,K) =W(K)*AFACT(I,K)
      DO 160 I=1,L
      SUM1=0.
      SUM2=0.
      DO 150 J=1,N
      DO 150 K=1,N
      SUM1 =SUM1+BIGA(I,J)*BIGC(I,K)*BIGC(I,J,K)
      SUM2 =SUM2+BIGA(I,J)*BIGC(I,K)*BIGD(I,J,K)
      SUM =SQRT(SUM1*SUM1+SUM2*SUM2)
      PHIG(I) =SUM*PHIP(I)
      160 CONTINUE
      WRITE (6,200) ISTAT

```

200 FORMAT //29H SPECTRAL DENSITY AT STATION 12 /1
 WRITE (6,205) (PWIC(I), I=1,L1)
 205 FORMAT(1E10.6)
 ISTAT =ISTATE1

IF(ISTAT.LE.NSTAT) GO TO 110

MEAN =NCNAME1
CD TO 3
END

DATA	1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80	81	82	83	84	85	86	87	88	89	90	91	92	93	94	95	96	97	98	99	100	101	102	103	104	105	106	107	108	109	110	111	112	113	114	115	116	117	118	119	120	121	122	123	124	125	126	127	128	129	130	131	132	133	134	135	136	137	138	139	140	141	142	143	144	145	146	147	148	149	150	151	152	153	154	155	156	157	158	159	160	161	162	163	164	165	166	167	168	169	170	171	172	173	174	175	176	177	178	179	180	181	182	183	184	185	186	187	188	189	190	191	192	193	194	195	196	197	198	199	200	201	202	203	204	205	206	207	208	209	210	211	212	213	214	215	216	217	218	219	220	221	222	223	224	225	226	227	228	229	230	231	232	233	234	235	236	237	238	239	240	241	242	243	244	245	246	247	248	249	250	251	252	253	254	255	256	257	258	259	260	261	262	263	264	265	266	267	268	269	270	271	272	273	274	275	276	277	278	279	280	281	282	283	284	285	286	287	288	289	290	291	292	293	294	295	296	297	298	299	300	301	302	303	304	305	306	307	308	309	310	311	312	313	314	315	316	317	318	319	320	321	322	323	324	325	326	327	328	329	330	331	332	333	334	335	336	337	338	339	340	341	342	343	344	345	346	347	348	349	350	351	352	353	354	355	356	357	358	359	360	361	362	363	364	365	366	367	368	369	370	371	372	373	374	375	376	377	378	379	380	381	382	383	384	385	386	387	388	389	390	391	392	393	394	395	396	397	398	399	400	401	402	403	404	405	406	407	408	409	410	411	412	413	414	415	416	417	418	419	420	421	422	423	424	425	426	427	428	429	430	431	432	433	434	435	436	437	438	439	440	441	442	443	444	445	446	447	448	449	450	451	452	453	454	455	456	457	458	459	460	461	462	463	464	465	466	467	468	469	470	471	472	473	474	475	476	477	478	479	480	481	482	483	484	485	486	487	488	489	490	491	492	493	494	495	496	497	498	499	500	501	502	503	504	505	506	507	508	509	510	511	512	513	514	515	516	517	518	519	520	521	522	523	524	525	526	527	528	529	530	531	532	533	534	535	536	537	538	539	540	541	542	543	544	545	546	547	548	549	550	551	552	553	554	555	556	557	558	559	560	561	562	563	564	565	566	567	568	569	570	571	572	573	574	575	576	577	578	579	580	581	582	583	584	585	586	587	588	589	590	591	592	593	594	595	596	597	598	599	600	601	602	603	604	605	606	607	608	609	610	611	612	613	614	615	616	617	618	619	620	621	622	623	624	625	626	627	628	629	630	631	632	633	634	635	636	637	638	639	640	641	642	643	644	645	646	647	648	649	650	651	652	653	654	655	656	657	658	659	660	661	662	663	664	665	666	667	668	669	670	671	672	673	674	675	676	677	678	679	680	681	682	683	684	685	686	687	688	689	690	691	692	693	694	695	696	697	698	699	700	701	702	703	704	705	706	707	708	709	710	711	712	713	714	715	716	717	718	719	720	721	722	723	724	725	726	727	728	729	730	731	732	733	734	735	736	737	738	739	740	741	742	743	744	745	746	747	748	749	750	751	752	753	754	755	756	757	758	759	760	761	762	763	764	765	766	767	768	769	770	771	772	773	774	775	776	777	778	779	780	781	782	783	784	785	786	787	788	789	790	791	792	793	794	795	796	797	798	799	800	801	802	803	804	805	806	807	808	809	810	811	812	813	814	815	816	817	818	819	820	821	822	823	824	825	826	827	828	829	830	831	832	833	834	835	836	837	838	839	840	841	842	843	844	845	846	847	848	849	850	851	852	853	854	855	856	857	858	859	860	861	862	863	864	865	866	867	868	869	870	871	872	873	874	875	876	877	878	879	880	881	882	883	884	885	886	887	888	889	890	891	892	893	894	895	896	897	898	899	900	901	902	903	904	905	906	907	908	909	910	911	912	913	914	915	916	917	918	919	920	921	922	923	924	925	926	927	928	929	930	931	932	933	934	935	936	937	938	939	940	941	942	943	944	945	946	947	948	949	950	951	952	953	954	955	956	957	958	959	960	961	962	963	964	965	966	967	968	969	970	971	972	973	974	975	976	977	978	979	980	981	982	983	984	985	986	987	988	989	990	991	992	993	994	995	996	997	998	999	1000
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** INPUT DATA **
GAMMA = 0.320000E-02    NO. OF FREQ. =39    NO. OF MODES = 9    NO. OF STAT. = 1
** BS **
0.488000E 03      -0.716000E 02      -0.995000E 02      0.359000E 02      -0.926000E 01      -0.176000E 02
0.147000E 02      0.145300E 04      0.117000E 03
GENERALIZED MASS
0.377000E-01      0.243000E-01      0.290000E-01      0.263000E-01      0.198500E-01      0.192000E-01
0.218000E-01      0.902000E-01      0.693000E-01
NATURAL FREQUENCY
0.228000E 02      0.426000E 02      0.872500E 02      0.105000E 03      0.188800E 03      0.306600E 03
0.348500E 03      0.110320E 00      0.150000E 02
FORCING FREQ
0.200000E 02      0.220000E 02      0.228000E 02      0.235000E 02      0.340000E 02      0.420000E 02
0.426000E 02      0.440000E 02      0.500000E 02      0.600000E 02      0.660000E 02      0.700000E 02
0.720000E 02      0.820000E 02      0.850000E 02      0.872000E 02      0.900000E 02      0.950000E 02
0.100000E 03      0.105000E 03      0.107000E 03      0.130000E 03      0.150000E 03      0.175000E 03
0.188000E 03      0.188800E 03      0.200000E 03      0.220000E 03      0.235000E 03      0.250000E 03
0.290000E 03      0.300000E 03      0.306600E 03      0.320000E 03      0.340000E 03      0.348500E 03
0.360000E 03      0.390000E 03      0.430000E 03

** PHIP ***
0.440000E-06      0.360000E-06      0.310000E-06      0.290000E-06      0.140000E-06      0.250000E-06
0.290000E-06      0.820000E-06      0.100000E-05      0.100000E-04      0.250000E-04      0.220000E-04
0.250000E-04      0.140000E-04      0.200000E-04      0.210000E-04      0.240000E-04      0.270000E-04
0.210000E-04      0.140000E-04      0.150000E-04      0.200000E-04      0.100000E-04      0.130000E-04
0.900000E-05      0.950000E-05      0.950000E-05      0.140000E-04      0.820000E-05      0.120000E-04
0.650000E-05      0.950000E-05      0.650000E-05      0.600000E-05      0.400000E-05      0.300000E-05
0.200000E-05      0.140000E-05      0.560000E-05

** W*COSIN.THETAI ***
0.786000E 00      -0.209000E 00      0.990000E 00      0.984000E 00      0.210000E 00      -0.408000E 00
-0.753000E 00      0.988500E 00      -0.234000E-01

SPECTRAL DENSITY AT STATION 1
0.973517E-03      0.354362E-01      0.525688E 01      0.688463E-01      0.105903E-02      0.228949E-03
0.199017E-01      0.898716E-02      0.697639E-02      0.662441E-01      0.173591E 00      0.163858E 00
0.196326E 00      0.247472E 00      0.103572E 01      0.391247E 02      0.169957E 00      0.761150E-04
0.440743E-04      0.415575E 01      0.280199E 00      0.828050E-01      0.407135E-01      0.547471E-01
0.628662E-01      0.530788E-01      0.354139E-01      0.544874E-01      0.320720F-01      0.467976E-01
0.218230E-01      0.198914E-01      0.178531E 00      0.404128F-01      0.354046E-01      0.140265E 00
0.426419E-02      0.493526E-02      0.212011E-01

```

** INPUT DATA **
GAMMA = 0.400000E-02

NO. OF FREQ. =39 NO. OF MODES = 9 NO. OF STAT. = 1

SPECTRAL DENSITY AT STATION 1

0.973438E-03	0.352809E-01	0.336459E 01	0.684233E-01	0.106901F-02	0.266405E-03
0.134164E-01	0.8966676E-02	0.697614E-02	0.662432E-01	0.173587E 00	0.163851E 00
0.196313E 00	0.247190E 00	0.102781E 01	0.253496E 02	0.169386E 00	0.110924E-03
0.684499E-04	0.266633E 01	0.276243E 00	0.828031E-01	0.407133E-01	0.547445E-01
0.583419E-01	0.477025E-01	0.354187E-01	0.544879E-01	0.320722E-01	0.467980E-01
0.218347E-01	0.201553E-01	0.124915E 00	0.403725E-01	0.352469E-01	0.946550E-01
0.428887E-02	0.493573E-02	0.212014E-01			

** INPUT DATA **
GAMMA = 0.500000E-02

NO. OF FREQ. =39 NO. OF MODES = 9 NO. OF STAT. = 1

SPECTRAL DENSITY AT STATION 1

0.973313E-03	0.350411E-01	0.215353E 01	0.677727E-01	0.106898F-02	0.321109E-03
0.926581E-02	0.893533E-02	0.697577E-02	0.662418E-01	0.173581E 00	0.163840E 00
0.196294E 00	0.246752E 00	0.101572E 01	0.163686E 02	0.168508E 00	0.165362E-03
0.106578E-03	0.171311E 01	0.270302E 00	0.828003E-01	0.407131F-01	0.547405E-01
0.536975E-01	0.442617E-01	0.354260E-01	0.544886E-01	0.320725E-01	0.467985E-01
0.218528E-01	0.205554E-01	0.906001E-01	0.403100F-01	0.350063E-01	0.654640E-01
0.432690E-02	0.493646E-02	0.212018E-01			